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STATIC AND DYNAMIC EVALUATION OF A-37 CAST AND CAST CARCASS/INTEGRAL TREAD TIRES

Paul C. Ulrich Mechanical Branch Vehicle Equipment Division

November 1980

TECHNICAL REPORT AFWAL-TR-80-3055

Final Report for Period September 1977 - September 1979

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one-piece cast tire, the cast carcass/replaceable tread tire, and the cast carcass/integral tread tire.

The development of the one-piece cast tire was terminated after three design iterations (15 tires) were evaluated due to its shortened dynamic life caused by tread groove failures (areas of high stress concentration).

The development of the two-piece cast carcass/replaceable tread tire was terminated after two design iterations (10 tires) were evaluated primarily due to tread derailment problems. These designs, however, represented a considerable improvement over the one-piece cast tire designs as the tread groove failures (areas of high stress concentration) were eliminated by replacing the thermoplastic material with conventional tire materials in the tire tread.

Twenty-five itegral tire design iterations (105 tires), some of which included glass reinforcement, were tested and evaluated to the A-37 aircraft main gear tire specifications. The testing resulted in the following significant achievements:

- 1. Successfully completed one-hundred A-37 qualification takeoff cycles. Each of these test cycles consisted of a simulated takeoff (excluding the taxi) from 0 mph to a liftoff speed of 150 mph at an initial load of 6650 lbs which decreased linearly to 0 lbs at liftoff.
  - 2. Withstood 91% (400 psig) of the burst test pressure requirements.
- 3. Successfully completed 2.65 taxi miles (14,000 ft) at a rated load of 6650 lbs and a taxi speed of 30 mph.
- 4. Successfully completed 1500 continuous miles at a reduced load of 1500 lbs and a taxi speed of 30 mph.

Even though these results are considered significant achievements for a cast thermoplastic, polyester, elastomer tire, they fall far short of the full A-37 main gear tire qualification due to their inability to complete the required taxi rolls at rated (6650 lbs) load without incurring permanent structural damage.

The major shortcomings of the integral tire designs were the thermoplastic, elastomer material's susceptibility to material creep and flex cracking which occurred during the taxi rolls at rated load and at high tire deflection.

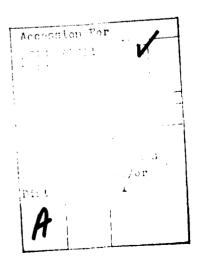
The major shortcomings of the rotational cast process were the inability to maintain a uniform wall thickness around the toroidal cross section of the tire causing areas of high stress concentration and localized heating, promoting material creep during dynamic testing, and the inability to obtain a proper material cure from tire to tire (poor repeatability) causing material degradation and a loss of material mechanical properties in some of the tires.

### **FOREWORD**

This report describes an in-house effort conducted by personnel of the Mechanical Branch (FIEM), Vehicle Equipment Division (FIE), Flight Dynamics Laboratory, Air Force Wright Aeronautical Laboratories, Wright-Patterson Air Force Base, Ohio, under project number 2402, "Mechanical Systems for Advanced Military Flight Vehicles," task number 240201, "High Performance Landing Gear for Advanced Military Flight Vehicles," work unit number 24020118, "Tire Ground Performance Criteria." This in-house effort was in support of Phase II and Phase III cast tire development work of contract F33615-76-C-3062. This report covers work performed during the period of September 1977 to September 1979, under the direction of the author, Paul C. Ulrich (AFWAL/FIEM), project engineer. The report was released by the author in March 1980. Previous results, Phase I, of this contract are reported in Technical Report AFFDL-TR-77-51, which was released in July 1977.

The author wishes to acknowledge the various suggestions received during this program from Aivars V. Petersons of the Flight Dynamics Laboratory and Dr. Howell K. Brewer of the Department of Transportation. The author also acknowledges the assistance contributed by Ted Dull, University of Cincinnati (co-op) student, R. W. Tatsch and W. Maggard of Systems Research Laboratories.





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### SECTION I

#### INTRODUCTION

#### BACKGROUND

Annually, replacement tires for aircraft ranks as a major logistics cost item in maintaining landing gear systems on US Air Force aircraft and the high rate of tire replacement represents considerable aircraft out-of-service time.

The conventional aircraft tire is an extremely complicated structure made from a variety of rubber, textile, and wire materials which are processed in numerous manufacturing stages and vulcanized into a tire shape. Variations and inconsistencies in manufacture are very difficult to control due to the multi-component assembly and the many hand operations required. Recently, developments in polymer chemistry offer the possibility of casting or molding tires from high strength, high molecular weight polymers utilizing automated systems. A comparison of a single component polyester elastomer tire and the multi-component conventional tire is shown in Figure 1. Several major tire companies have cast/molded automotive tires which have passed laboratory and service endurance tests but still require improvements in tread wear and traction. It is anticipated that if a cast carcass/replaceable tread aircraft tire can be developed, the tread belt can be changed without removing the tire/wheel/brake assembly from the aircraft, thus reducing maintenance costs and aircraft out-of-service time. In addition, ninety percent of the expensive conventional tire building equipment can be eliminated and human error could be reduced through cast tire automation. If a thermoplastic polyester material is used in the cast tire, additional savings can be realized by recycling the material after the tire has been removed from service, thus reducing the petroleum requirements of conventional tire building and eliminating conventional tire disposal problems. Therefore, with the advantages and potential cost savings offered by cast tires, the development of cast tires for military aircraft was pursued.

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During the course of the program, three one-piece cast tire designs, two cast carcass/replaceable tread tire designs, and 25 cast carcass/integral tread tire designs were developed and evaluated. The carcasses of all the tire designs were of a continuous toroidal construction without textile cord resoface sent on bead bundles. These tire carcasses were rotationally cast-solder from a thermoplantic, polyester, elastomer called 'hytre' it like cast. "If their materials were evaluated, some of which included corresponding to the second resolution.

Fach of the time designs was subjected to an extensive series of static and quasi static tests, and dynamic laboratory tests in accordance with the military armoraft time specification MIL-1-5041 and the A-37 main gear time specification, USAF DWG 67J1951. Test results are discussed and compared to baseline A-37 main gear conventional bias time results.

<sup>\*</sup>Dupont Trade Name

### 2. OBJECTIVES

### a. Overall Objective

To investigate if a cast carcass tire with or without a conventional rubber tread belt is a practical concept for military aircraft.

## b. Specific Objectives

## (1) Phase I

To survey and evaluate currently available off-the-shelf thermoset and thermoplastic materials for potential use in cast/molded tires for military aircraft.

### (2) Phase II

To develop, laboratory test, and evaluate a one-piece cast tire and/or a cast carcass/replaceable tread tire and/or a cast carcass/integral tread tire which can satisfactorily meet the dynamic requirements of a high performance military aircraft tire.

## (3) Phase III

To optimize a final prototype cast tire design.

#### SECTION II

#### SUMMARY

This report describes work undertaken during Phases II and III of a three Phase program initiated with Zedron Inc., to establish the potential of cast tires for application to Air Force aircraft. Phase I of this effort was a materials survey, the results of which are documented in AFFDL-IR-77-51, "Development of Cast Carcass Tires for Military Aircraft." The current program Phases II and III involved static, quasi-static, and dynamic laboratory test and evaluation of thirty 7.00-8 Type III cast tire designs. These designs included tire carcasses which were rotationally cast/molded from thermoplastic polyester materials (Hytrels) of various hardness with and without reinforcements. Three basic cast tire designs were developed and evaluated during Phase II efforts; the one-piece cast tire, the cast carcass/replaceable tread tire and the cast carcass/integral tread tire.

The development of the one-piece cast tire was terminated after three design iterations (15 tires) were evaluated due to its shortened dynamic life caused by tread groove failures (areas of high stress concentration).

The development of the cast carcass/replaceable tread tire was terminated after two design iterations (10 tires) were evaluated primarily due to tread derailment problems caused by insufficient expansion of the carcass required to keep the tread belt on. These designs, however, represent a considerable improvement over the one-piece cast tire designs as the tread groove failures (areas of high stress concentration) were eliminated by replacing the thermoplastic material with conventional tire materials in the tire tread.

The remainder of Phase II (Design Development) and all of Phase III (Design Optimization) involved the development, test, and evaluation of the cast carcass/integral tread tire. The integral tread tire is a

rotationally cast thermoplastic carcass with a conventional rubber compound (aramid cord reinforced) tread belt. Unlike the replaceable tread tire, the tread belt of the integral tire is fabricated, glued, and cured in place on the cast carcass which rendered it non-replaceable. The integral tread tire designs incorporated the same qualities or improvements as the replaceable tread tire but it eliminated the tread derailment problems of the replaceable tread tire.

Twenty-five integral tire design iterations (105) tires, some of which included glass reinforcement, were tested and evaluated to the A-37 aircraft main gear tire specifications. The testing resulted in the following significant achievements:

•Successfully completed one-hundred A-37 qualification takeoff cycles. Each of these test cycles consisted of a simulated takeoff (excluding the taxi) from 0 mph to a liftoff speed of 150 mph at an initial load of 6650 lbs which decreased linearly to 0 lbs at liftoff.

•Withstood 91% (400 psig) of the burst test pressure requirements.

•Successfully completed 2.65 taxi miles (14,000 ft) at a rated load of 6650 lbs and a taxi speed of 30 mph.

•Successfully completed 1500 continuous miles at a reduced load of 1500 lbs and a taxi speed of 30 mph.

Even though these results are considered significant achievements for a cast thermoplastic, polyester, elastomer tire, they fall far short of the full A-37 main gear tire qualification due to the tire's inability to complete the required taxi rolls at rated (6650 lbs) load without incurring permanent structural damage.

The major shortcomings of the integral tire designs were the thermoplastic, elastomer material's susceptibility to material creep and flex cracking which occurred during the taxi rolls at rated load and at high tire deflections.

The major shortcomings of the rotational cast process were the inability to maintain a uniform wall thickness around the toroidal cross section of the tire causing areas of high stress concentration and localized heating (promoting material creep) during dynamic testing and the inability to obtain a proper material cure from tire to tire (poor repeatability) causing material degradation and a loss of material mechanical properties in some of the tires.

### SECTION III

#### DESCRIPTION OF TEST TIRES

During the course of this contract, three basic 7.00-8 cast tire designs were developed; the one-piece cast tire, the cast carcass/ replaceable tread tire, and the cast carcass/integral tread tire shown in Figures through 10. The conventional 7.00-8/16 PR bias tire is shown in Figures 11 and 12 for comparison. The aspect ratio (section height/section width) of the low profile cast tire designs ranged from 0.63 to 0.67. The carcasses of all the tire designs were of a continuous closed toroidal construction without textile cord reinforcement or wire bead bundles. These tire carcasses were rotationally cast/molded from a thermoplastic, polyester, elastomer called "Hytrel". Five "Hytrel" base materials 6346, 5556, 5555HS, 5526, and 4056 were evaluated. A summary of the various cast tire designs is listed in Table 1. The hardness of the base polymer is designated in shore "D" hardness by the first two digits of the identification number. For example, base polymer 6346 had a hardness of 63 shore "D" prior to the addition of reinforcement fillers or plasticizers. The tread belt adhesives, and the thermal cure cycles for the cast carcasses and tread belts are listed in Table 2. The basic differences in the materials were hardness, melt index values or high temperature properties. These high strength, high molecular weight polymer systems consisted of a plastic (hard phase) and an elastomer (soft phase). These base polymers were also modified by adding carbon black, plasticizers such as Benzoflex and glass fillers such as glass flake, glass strand, and chopped glass. The glass loading and the strand or fiber length of the glass fillers were varied to evaluate how the reinforcement of the hard phase and the filler dispersal in the base polymer was affected. The glass fillers were also treated with various sizing agents such as silane to improve their adhesion to the polyester substrate. The carbon black was dropped in the later designs since it was added primarily for color or aesthetic purposes. Hence, the carcasses of the later designs appear white instead of black.

Due to the closed toroidal construction of the carcass, a rubber valve was glued in the tire's sidewall for inflation purposes. Internal air pressure leakage around the valve was a problem in some of the initial designs but was corrected by using a better adhesive. In the later designs, a second valve was added to the opposite sidewall in order to introduce nitrogen into the tire cavity during the cure process in order to eliminate internal oxidation and material property degradation. This second valve introduced valve retention problems in the last designs and actually prevented some of the last tires from being tested as the contract was terminated before this deficiency was corrected. It is felt that a mold rework on the second valve providing a better valve fit would correct the problem.

Reference 1 contains a detailed analysis of the polyester material properties with and without reinforcements, the tire carcass and tread manufacturing process, the tread rubber formulations, a description of the rotational cast equipment, and process and procedures.

The rotational cast/molding machine is shown in Figure 13, while a mold spider loaded with tire molds entering the oven is shown in Figure 14. Figure 15 shows a close-up of the one-piece cast tire mold. The replaceable tread belts were laid up on an orbitread building machine shown in Figure 16. A close-up of the expandable mandrel for the tread layup is shown in Figure 17. The four ply tread belts were laid up with two circumferential polyester belt plies  $\pm 1/2^{\circ}$ ,  $\pm 1/2^{\circ}$  cord angle, respectively, one radial polyester belt ply (89°) cord angle, then one circumferential polyester belt ply  $\pm 1/2^{\circ}$  cord angle, followed by circumferential high strength aramid reinforcement cords randomly dispersed across the tread section.

The tread belt of the cast carcass/integral tread tire was laid up, glued, and cured in place on the cast carcass rather than on the orbitread unit. The tread belt was reinforced by circumferential high strength aramid cords randomly dispersed in the upper tread area. This tire design evolved from the two previous designs and eliminated the tread derailment problems of the replaceable tread tire, and the groove cracking of the one-piece cast tire.

#### SECTION IV

#### TEST EQUIPMENT

The laboratory tire tests were conducted in the Flight Dynamics Laboratory Landing Gear Development Facility using the flat surfaced Tire Force Machine (TFM), the 120 inch programmable dynamometer, and the 84 inch conventional dynamometer.

### TIRE FORCE MACHINE (TFM)

The TFM was used for the quasi-static and some of the static mechanical tire property measurements recorded while under aircraft wheel loads with combined steering on a flat surface. The force measuring system consists of six load cells (3 vertical, 2 fore-aft and 1 lateral) instrumented to measure all six force and moment components developed by the tires. The machine is designed to permit low speed tests at yaw angles between  $\pm 90$  degrees (this  $\pm 90$  degrees position was used for the lateral stiffness tests) and any desired value of longitudinal slip.

A Houston Instrument Omnigraphic (X-Y plotter) recorder was used in the load-deflection tests. A Gould Brush (8 channel strip chart) recorder was used in the lateral force and aligning torque tests.

### 2. 120 INCH PROGRAMMABLE DYNAMOMETER

This dynamometer, incorporating a force measuring system similar to the TFM, has the capability of programmable yaw, camber, radial load, wheel velocity, wheel acceleration, and sink rate. The taxi takeoff cycles, takeoff only cycles, taxi rolls, and cambered taxi rolls were conducted on the 120 inch dynamometer.

## 3. 84 INCH CONVENTIONAL DYNAMOMETER

The 84 inch dynamometer is used for taxi takeoff cycles and high speed brake stops controlled by an electro-mechanical servo system. Some of the static measurements and taxi rolls were conducted on the 84 inch dynamometer.

Descriptions and capabilities of the TFM, the 120 inch, and 84 inch dynamometers are listed in the Flight Dynamics Laboratory Landing Gear Development Facility Brochure.

#### SECTION V

#### TEST REQUIREMENTS AND PROCEDURES

#### STATIC TESTS

### a. Dimensional and Physical Data

The cast tires were mounted on a 7.00-8 production aircraft wheel which has a 9.624 inch flange diameter, 8.0 inch bead seat diameter, and a 5.5 inch width between flanges. The wheel contours and dimensions are shown in Table 3. After mounting the tires, the outside diameter (OD) and the cross section (CS) of the tires were measured and recorded at 20 psig intervals from 0 psig to 160 psig inflation pressures to check tatic growth properties.

In addition, one tire of each cast tire design was inflated to rated inflation pressure (125 psig) and monitored continuously for 60 hours to check for material porosity and material permeability.

## to. Dimensional Stability Data

to rated in lation pressure (125 psig) and the OD and CS were recorded continuously for 60 hours to check the dimensional stability of the tires.

### c. Tire Contact Area

The contact area prints (footprints) were obtained for each cast tire design when loaded against a flat surface at rated load (6650 lbs), 60 rated load (3990 lbs), and rated pressure (125 psig). The gross contact area of the tire footprint was measured and is defined as the total area of the print including the tread ribs and the spaces (tread grooves) between the tread ribs. The net area of the print was also measured and is the summation of the individual tread rib (dark) areas where tread material contacts the flat surface.

#### d. Vertical Load Vs Deflection Data

Vertical load vs vertical deflection curves were obtained on each tire design when loaded on a flat surface at rated inflation pressure. In addition, vertical load vs vertical deflection loops were obtained on

some of the tire lesigns at inflation pressures of 90, 125, and 160 program Ail vertical load vs vertical deflection curve, and loops were obtained at a deflection rate of 20 inches per minute. Thre spring rate, and energy loss due to hysteresis were calculated.

#### e. Lateral Load Vs Deflection Data

Lateral load vs lateral deflection loops were obtained on the tungsten carbide surface on the time force machine at vertical loads of 6300 and 6650 lbs and at rated inflation pressure of 125 psig. The lateral load deflection tests were conducted by determining the lateral load required to produce 100 slip and then obtaining the lateral load deflection loops at a deflect on rate of 20 inches per minute until ±80 of the lateral slip load was achieved. Lateral stiffness was calculated from the lateral load deflection curves by two different methods. The first method involved taking the slope of a straight line of the load deflection curve from the origin to the tip of the loop. The second method used a straight line between the loop ends to establish the slope or stiffness.

## f. Fore-Aft Load Vs Deflection Data

Fore-aft load vs fore-aft deflection loops were to be obtained on the tungsten carbide surface of the tire force machine at vertical loads of 6300, 6650, and 7000 lbs, and at a rated inflation pressure of 125 psig. The fore-aft load deflection tests were conducted by determining the fore-aft load required to produce 100° slip and then obtaining the fore-aft load deflection loops at a deflection rate of 10 inches per minute until +80 of the fore-aft slip load was achieved. The same two methods used to calculate lateral stiffness were also to be used to determine fore-aft stiffness.

## g. Burst Test Data

One cast tire of each design was to be inflated at an inflation rate of approximately 30 psig per minute until the minimum burst pressure of 441 psig was reached and maintained for a minimum of ten seconds.

## 2. QUASI-STATIC (LOW SPEED) FLAT SURFACE TESTS

#### a. Lateral Force Data

Lateral force data was obtained on the dry and wet (1/2 inch water) tungsten carbide surface of the low speed (0.17 mph) tire force machine at vertical loads of 6300, 6650, and 7000 lbs, at a rated inflation pressure of 125 psig, and at slip angles of  $\pm 2^{\circ}$ ,  $\pm 4^{\circ}$ ,  $\pm 6^{\circ}$ ,  $\pm 8^{\circ}$ ,  $\pm 10^{\circ}$ , and  $\pm 12^{\circ}$ .

## b. Aligning Torque Data

Aligning torque data was obtained on the dry and wet (1/2 inch water) tungsten carbide surface of the low speed (0.17 mph) tire force machine at vertical loads of 6300, 6650, and 7000 lbs, at a rated inflation pressure of 125 psig, and at slip angles of  $\pm 2^{\circ}$ ,  $\pm 4^{\circ}$ ,  $\pm 6^{\circ}$ ,  $\pm 8^{\circ}$ ,  $\pm 10^{\circ}$ , and  $\pm 12^{\circ}$ .

### 3. DYNAMIC (HIGH SPEED) DYNAMOMETER TESTS

The following is taken from USAF Drawing Specification 67J1951:

- a. The tire shall withstand 100 cycles of (1) and (2) and 50 cycles of (3) without evidence of failure.
- (1) Taxi Takeoff Maximum Load The tire shall be taxied on the flywheel for 14,000 feet at 30 mph and at a load of 6650 pounds. Stop the flywheel, keeping the tire fully loaded. Then accelerate the flywheel (simulating takeoff) at a rate of 10 ft/sec/sec to a speed of 150 mph. The tire shall be unlanded after a roll distance of 2420 feet is reached. The load of 6650 pounds shall be maintained for 5 seconds at which time the load shall be decreased linearly with time to 0 load at approximately 22 seconds after the start of takeoff at which time the tire shall be unlanded.
- (2) Low Speed Mil-SPEC Landing A test cycle identical to the low speed (90 0 mph) dynamic test described in military specification MIL-T-504l calculated for a tire load of 6650 pounds shall be conducted.
- (3) Combined Radial-Side Load Roll Test The radial load shall be 6700 pounds; the side load acting inboard shall be 1500 pounds. The tire shall be rolled 1500 feet each cycle at 20 mph. This test may be conducted by camber or yaw conditions.

#### SECTION VI

#### TEST RESULTS AND DISCUSSION

#### 1. STATIC TESTS

## a. Dimensional and Physical Data

Dimensional growth in the tire's OD and CS width due to increases in inflation pressure for the one-piece, two-piece, and the integral cast tire are plotted and compared to the baseline bias tire in Figures 18, 19, and 20.

Unlike the baseline bias tire, the cast tire designs exhibited a much more linear growth in OD. The overall growth of the one-piece cast tire in OD over the inflation pressure range of O - 150 psig was less than the baseline bias tire for two of the designs and slightly greater for the third design. The growth in cross section width of all three one-piece cast tire designs was roughly equivalent and the curves were characteristically similar to the baseline bias tire.

Both the two-piece and integral cast tire designs exhibited considerably less growth in OD and slightly greater growth in CS than the baseline bias tire.

A summary of the various cast tire designs, carcass weights, tread belt weights, outside diameter, and section width dimensional data is listed in Table 1.

Since the primary objective of this effort was to investigate if the cast tire concept is a viable concept (structurally capable) for high performance military aircraft tires, minimal effort was expended to optimize the total tire weight or maintain the tire's dimensional envelope in accordance with the military specifications. Hence, many of the designs exceeded the 24 pound maximum allowable tire weight and the maximum OD and CS dimensions of 20.85 inches and 7.3 inches, respectively.

## b. Dimensional Stability Data

The dimensional stability in the OD and CS of the various cast tire designs were measured, recorded, plotted, and compared to the baseline bias tire in Figures 21, 27, and 23. This fimensional data was obtained with the tires inflated to 125 oni: reflation pressure and monitored for 60 hours. The dimensional stability of all the cast tire designs under static inflation was better than the baseline bias tire. The two integral cast tire design growth curves presented are typical of all the integral cast tire designs. The set-up for the dimensional tests is shown in Figures 24 and 25.

## c. Tire Contact Area (Footprint Data)

tire contact area prints were obtained for the baseline bias tire and all the cast tire designs at rated inflation pressure (125 psig) while loaded on a flat surface at rated load (6650 lbs) and at 60 rated load (3930 lbs). The maximum footprint length and width were measured and listed in Table 4. The gross and net contact areas of the footprints are listed in Table 4 and plotted in Figures 26, 27, 28, and 29. The 40 "D" Hytrel material integral cast tire designs came very close to matching the gross and net contact areas of the baseline bias tires whereas the remaining integral cast tire designs ranged from 7 to 36 less than the baseline bias tire. The one-piece cast tire designs ranged from 30 to 50 less in gross and net contact area while the two-piece cast tire designs ranged from 10 to 30 less in gross and net contact areas of the baseline bias tire. All of the contact area prints are presented in Appendix C.

### d. Vertical (Radial) Load Vs Vertical Deflection Data

Vertical load vs vertical deflection loops were obtained up to a radial load of 7000 lbs on a flat plate. Tire vertical spring rates and energy loss due to hysteresis are listed in Table 5. A plot of vertical load vs vertical deflection at 125 psig inflation pressure comparing the relative vertical stiffness of the various cast tire designs with that of the baseline tire is presented in Figure 30. The vertical stiffness of most of the cast tire designs averaged approximately 40 greater than the baseline bias tire and ranged from 16 below to 70 above the baseline

value. Many of the cast tire designs exhibited lower energy loss due to vertical load than the baseline bias tire. This fact plus the generally greater vertical stiffness resulting in lower tire deflections of the cast tires offer some explanation as to why the cast tires ran cooler during dynamic testing. Typical vertical load vs vertical deflection loops are presented in Figures 31 through 42. Vertical deflection vs vertical load curves were also obtained at three inflation pressures (90, 125, and 160 psig), at a rate of deflection of 20 inches per minute up to a load of 7000 lbs on a flat surface, and on the (curved surface) 84 inch diameter dynamometer. These plots are presented in Appendix D.

### e. Lateral (Side) Load Vs Lateral Deflection Data

Lateral load vs lateral deflection loops were obtained on the tungsten carbide surface of the tire force machine. Lateral load vs lateral deflection plots at vertical loads of 6650 lbs and 6300 lbs are shown in Figures 43 through 51 and 52 through 60, respectively. The tire lateral spring rates (lateral stiffness) and energy loss due to hysteresis are listed in Table 6. The lateral stiffness of the integral cast tire designs ranged from 15, below to 15, above the baseline bias tire value. The lateral energy loss of the integral cast tire designs ranged from 19 below to 22 above the baseline bias tire value. The effective coefficient of friction obtained during the lateral load deflection tests of the integral cast tires ranged from 3. to 18% lower than the baseline bias tire. The test set up for the lateral load deflection tests on the tire force machine prior to loading the tire is shown in Figure 61. One of the one-piece cast tires (Design 3) was loaded laterally to its structural limit and failed catastrophically (Figures 62 and 63) at a lateral load of 7000 lbs.

## f. Fore-Aft Load Vs Fore-Aft Deflection Data

Fore-aft load vs fore-aft deflection tests were set up on the tire force machine. Brake torque was applied to the tire through use of the TFM brake shown in Figure 64. The fore-aft load deflection tests were terminated due to slip at the tire/wheel interface. As much as one inch of circumferential slip occurred at 1000 ft-lbs of brake torque. This slippage problem would have to be addressed in later designs prior to dynamic brake qualification tests.

#### g. Burst Test Data

Static burst tests were to be conducted on all cast tire designs. The results of the burst tests are tabulated in Table 7 and plotted in Figures 65 and 66. None of the cast tire designs passed the minimum burst test requirement at the 30 psig per minute inflation rate even though some of the designs reached 91% of the minimum requirement. In order to check the effect of inflation rate and material creep due to internal inflation, a second integral cast tire of the S/N BO88HX (Design 8) series was inflated to 465 psig in 30 seconds (high inflation rate) and maintained at this pressure. It failed catastrophically in approximately 56 seconds. Since this tire withstood the minimum required burst test pressure for 10 seconds, it essentially passed the burst test requirement. This burst test requirement, however, has shortcomings and is inadequate for evaluating polyester elastomer tires which are susceptible to plastic deformation and material creep. Typical failures which occurred during the burst tests of the various designs are shown in Figures 67 through 76. All the burst test failures showed some evidence of material creep. The creep initiated in areas of high stress concentration, e.g., the tread grooves of the one-piece cast tire, or in the beads, which along with the tread grooves were usually the thinnest wall section. There was also evidence of creep in the shoulder areas at the edge of the tread belt and in the crown under the tread belt at locations where the tread belt was yielding prior to belt failure.

The brittle failures normally occurred in the harder base polymer materials or in those designs with glass reinforcement. The brittle failures (cracking) probably initiated at sites of material imperfections (improper material cure) or most likely at thinned sections (material creep of the soft phase) or at locations of shock loading (tread belt failures).

The design with the highest burst pressure was the 5556 material, Design 21 (Table 1) with the number 21 thermal cure cycle (Table 2). This design had no carcass reinforcement (glass), a carcass weight of 17 pounds, and a tread belt weight of 10 pounds. The reason glass reinforcement was

not tried with the 5556 material was the inability of the contractor to obtain good glass dispersion in the base polymer during earlier sample tests.

The parameters which tended to affect the burst pressure were the shore "D" hardness of the base polymer, the material thermal cure, the tread belt weight and reinforcement or number of belts and aramid cords per square inch (end count), the carcass weight (amount of polymer used or carcass thickness), and glass reinforcement in the base polymer. The proper balance of carcass weight (thickness), thermal cure, glass reinforcement of the carcass, tread belt weight, and belt reinforcement produced significant increases in burst pressure of the various designs. The addition of the plasticizer, Benzoflex, decreased the burst pressure of a design, while the addition of carbon black had little or no effect on the burst pressure through reinforcement of the soft phase of the rubber/ plastic matrix of the polyester polymer. The addition of glass flake or chopped glass strand did not significantly affect the burst strength of the designs.

### 2. QUASI-STATIC (LOW SPEED) FLAT SURFACE TESTS

## a. Lateral Force Data

Lateral force data was obtained on the dry and wet tungsten carbide surface of the tire force machine at three vertical loads and at rated inflation pressure. The test set up on the tire force machine is shown in Figures 77 and 78. Carpet plots of lateral force vs positive slip angle for three vertical loads are presented in Figures 79 through 97 and listed in Table 8 which compares the various cast designs with the baseline bias tire.

During the dry surface tests, the one-piece cast tire developed lateral forces approximately 40% greater than the baseline bias tire, whereas the two-piece cast tire developed lateral forces up to 7% greater than the baseline tire. The majority of the integral cast tire designs developed lateral forces which ranged from 1% to 17% greater than the baseline bias tire. Three of the designs tested, however, developed less lateral force than the baseline tire.

Under wet surface conditions, the bias tire exhibited a slight decrease in lateral force at all slip angles (Figure 95). Both the one-piece and two piece cast tire, exhibited a slight decrease in the lateral force at small. Tip angles but exhibited considerable degradation in developed lateral force at slip angles greater than 6° shown in Figures 96 and 37.

## b. Aligning Torque Data

Aligning torque data was obtained on the dry and wet tungsten carbide surface of the tire force machine at three vertical loads and at rated inflation pressure. Carpet plots of aligning torque vs positive slip angle for the three vertical loads are presented in Figures 98 through IIb and listed in Table 9 comparing the various cast tire designs with the baseline bias tire.

Ihe results of the dry surface tests showed that the one-piece and two-piece cast tires developed greater aligning torque than the bias tire at slip angles less than 4°, while all the integral cast tire designs developed much less aligning torque than the baseline tire at all slip angles.

The wet surface tests showed a degradation in aligning torque for the baseline bias tire at slip angles greater than 6° (Figure 114). Both the one-piece and two-piece cast tires exhibited considerable degradation in aligning torque at all slip angles as shown in Figures 115 and 116.

### DYNAMIC (HIGH SPEED) DYNAMOMETER TESTS

The results of the dynamometer tests for the various cast tire designs are tabulated in Table 10. It quickly became apparent that the most difficult phase of the A-37 main gear tire dynamic qualification tests was the 2.65 mile taxi rolls prior to each of the 100 takeoff cycles which required the tire to roll long distances at rated load (6650 lbs). Consequently, the 2.65 mile (14,000 ft) taxi roll was used as a relative gauge of merit for the various cast tire designs. The primary modes of failure during the taxi roll of the cast tire designs using the softer and more flexible polymer matrix was permanent set occurring through

localized heating in high stress concentration areas causing material creep and section thinning. Even though continuous contained air temperatures or surface temperatures could not be obtained on the cast tires, the results of periodic temperature measurements indicate that the cast tires ran much cooler than the bias tire. It is estimated that the cast tire contained air temperature never exceeded 150°F during the taxi tests. The tire designs which used the harder polymer or the softer polymer reinforced with glass tended to be brittle with a low resistance to flex cracking and subsequently failed due to a combination of fatigue cracking of the hard phase and material creep of the soft phase of the polymer matrix. The brittle failures (cracking) probably initiated at sites of material imperfections (improper material cure) or most likely at thinned sections or locations of shock loading (tread belt failure). The areas of high stress concentration were in the tread grooves and bead areas of the tire and at the shoulders along the edge of the tread belt.

The development of the one-piece cast tire was terminated after three design iterations (15 tires) were evaluated due to tread groove failures (areas of high stress concentration) as shown in Figures 117 and 118. The use of the base polymer 6346 was halted due to its poor resistance to flex cracking.

The development of the cast carcass/replaceable tread tire was terminated after two design iterations (10 tires) were evaluated due to tread derailment problems as shown in Figure 119. The probable cause of the tread derailment problems was the inability to obtain sufficient expansion in outside diameter of the cast polyester elastomer carcass to provide an adequate fit between the tread belt and carcass. These designs, however, represented a considerable improvement over the one-piece cast tire designs as the tread groove failures (areas of high stress concentration) were eliminated. In addition to the tread derailment problems, the two-piece replaceable tread tires failed in the bead areas due to section thinning or material creep (shown in Figure 120) and fatigue or flex cracking as shown in Figure 121. The stress concentration in the bead areas in the early designs were magnified by the sharp bead radius which existed in the 7.00-8 aircraft

wheel and in the cast tire mold. In order to reduce the stress concentration in the bead area without modifying the wheel, a set of aluminum bead rings with a larger bead radius was manufactured to fit the bead area of the wheel. The tire mold was then modified to fit the aluminum bead rings. This modification significantly reduced the stress concentration in the bead area of the later designs but the problem was not completely eliminated. The change in the bead radius contour can be seen by comparing Figures 7 and 10.

Two major problems with the rotational mold process itself or the contractors technique surfaced early in the program and hindered the development of a successful cast tire throughout the effort. These were the inability to maintain a uniform wall thickness around the toroidal cross section (an exaggerated case shown in Figure 120) and the inability to obtain a proper material cure from tire to tire (poor repeatability).

The non-uniform wall thickness around the toroid inherently caused areas of high stress concentration and localized heating (promoting creep) during dynamic testing. An improper material thermal cure (Figure 122) caused material degradation, a loss of material mechanical properties and provided failure initiation sites.

The integral cast tire design with the best taxi performance was series number B088J(X) (Design 10) which successfully completed one complete taxi roll of 2.65 miles at a constant load of 6650 lbs and a test inflation pressure of 134 psig. This design used unmodified 5556 material with no glass reinforcement, a carcass weight of 17.5 pounds, and a tread belt weight of 7.5 pounds. The thermal cure used is listed as number 10 in Table 2. This tire failed due to material creep in the shoulder area at the belt edge during the second 2.65 mile taxi roll after 0.3 additional miles.

Several other designs exceeded one mile of taxi roll before failure occurred as indicated in Table 10. The integral tire designs using the softer polymers failed due to section thinning or material creep in the shoulder areas at the edge of the tread belt or in the

bead radius area. Typical examples of these failures are shown in Figures 123 through 129. The integral tire designs using the harder polymers and glass reinforced polymers failed due to fatigue (flex cracking) of the hard phase and material creep of the soft phase in the sidewall, shoulder, and bead areas. Typical examples of these failures are shown in Figures 130 through 134.

To assess the high speed takeoff capabilities of the integral tires, a tire of the BO68C(X) series (Design 6) which did not perform that well during the taxi roll tests (failure at 0.37 miles), was subjected to the A-37 main gear tire takeoff profile (0 - 150 mph) with an initial load of 6650 lbs which decreased linearly to 0 lbs at lift off. The tire is shown in Figure 135 after successfully completing 20 takeoff cycles. The tire successfully completed 100 takeoff cycles and subsequently failed in the shoulder area (shown in Figure 136) during the first taxi test after 0.2 roll miles.

During several design iterations, the cast tire mold in the shoulder area was remachined to relieve the stress concentration in the cast tire carcass at the edge of the tread belt. This rework of the tire mold was successful in reducing the stress concentration in the shoulder area but it did not completely eliminate failures in the shoulder area.

Tire slippage at the tire/wheel interface (bead area) was a problem during one taxi test. Approximately 3.5 inches of circumferential slip (shown in Figure 137) was measured after the dynamic taxi test of the B068C(X) series tire (Design 6).

In order to check the effect of the tire load on taxi endurance life of a cast tire, three tires of the BO88I(X) series (Design 9) and two tires of the BO88J(X) series (Design 10) were tested at different loads. The results of these tests (Table 11) show the BO88I(X) series tire tested at 1500 lbs load rolled 1500 continuous miles before failure occurred. These results indicate the order of magnitude that load affects the cast tire life.

To check the effect of tire inflation pressure on taxi endurance life, the inflation pressure was varied on three tires of two different design series. The results of these tests (shown in Table 12) indicate that changes in inflation pressure of this magnitude do not affect the taxi endurance life of cast tires. In one case, both the inflation pressure and test load were reduced considerably. The endurance life of this tire was extended significantly but it is felt that the load reduction was the primary reason for the extended tire life.

In an attempt to reduce material oxidation which occurred inside the cast tires during their thermal cure cycle, nitrogen was pumped into the tire cavity during the thermal cure cycle at 1.5 psig pressure. This procedure appeared to reduce the internal material oxidation but the early termination of the contract did not allow sufficient development and evaluation time.

Early in the last year of the three year program, the contractor experienced insurmountable financial difficulties in other areas of the company and they were forced to prematurely terminate the Air Force cast tire contract. With the contract duration time considerably shortened, the probability of developing a successful integral cast tire design for use on military aircraft was significantly reduced. Even so, considerable achievements were accomplished even though they fell far short of the original expectations.

### SECTION VII

#### CONCLUSIONS

The results of the Phase II and Phase III static, quasi-static and dynamic testing of the various cast tire designs has led to the following conclusions:

- 1. The off-the-shelf materials tested in this program with and without glass reinforcement proved to be inadequate for use as a viable cast tire material for the A-37 military aircraft.
- 2. The integral cast tire proved to be the best tire design for quick or snort term cast tire success as it eliminated the tread groove failures of the one-piece cast tire and the tread derailment problems of the two-piece cast tire.
- .. The two-piece/replaceable tread cast tire has significant payoffs and patential for long term cast tire success but much development in the area of increased 00 expansion of the stiff polyester cast carcass and tread retention needs to be done.
- 4. The annodified, unreinforced 5556 "Hytrel" material had the test dynamic performance as it successfully completed three miles of taxi at rated land and attained the nighest burst pressure (400 psig).
- polymers by reinforcing the hard phase of the matrix but tended to make the resulting polymer brittle and have poor resistance to flex cracking. More development work is required in glass reinforcement techniques. In inificantly, material creep still occurred in the soft phase of the glass reinforced polymers, since, only the hard phase was reinforced. To elicinate the material creep failures, the soft phase will have to be reinforced.
- 6. The thermal cure greatly affected the dynamic performance of the tire, as an improper material cure caused material degradation and a locume causedmaterial degradation and cure and unitors saterial distribution around the toroidal tire proved year difficult to achieve with the rotational castimolding process.

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- 7. The control and repeatability of a uniform wall thickness around the toroidal tire and at sharp radii proved to be very difficult with the rotational cast/mold process. This was especially true for the heavier wall thickness designs which were felt to be necessary for a cast tire of adequate strength.
- 8. Material oxidation which occurs inside the tire during the thermal cure cycle caused material degradation and must be eliminated. The introduction of an inert gas in the tires cavity is a possible solution but the termination of the contract did not allow sufficient time for a complete evaluation.
- 9. The materials tested in this effort seemed to provide adequate, if not superior, static dimensional stability, vertical and lateral stiffness, hysteresis, and frictional characteristics of specific cast tire designs when compared to the baseline bias tire.
- 10. The cast tire designs and materials were inadequate with respect to net and gross contact areas (footprint), fore-aft brake loads, aligning torque characteristics, burst test requirements and the dynamic tire qualification test requirements.

#### SECTION VIII

## RECOMMENDATIONS

Since the contract was prematurely terminated before liquid injection molding (LIM) processes and thermoset polyurethane material tire designs could be evaluated, it is recommended that the LIM process and polyurethane material tire designs be evaluated at some future date.

In addition, with the advantages and potential cost savings offered by cast/molded tires, consideration should be given to the eventual development of polymers specifically structured for high performance military aircraft tire application without restricting the effort to off-the-shelf materials.

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APPENDIX A

TABLES

TWBLE 1 CAST TIRE DESIGN DOIN

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*310v0 *1010k	e-d	* ;	٠,		-7	£+-		ig r	,	. • •		era Lecal	- i		<i>;</i>		
CARCASS THICKGESS WALDING SAGGELAVE, (TA),CV_LE*	0.32 - 0.50	),25 ± 0,4⊍	3.30 - 0.56	RE:	0.25 - 0.45	0.30 - 0.56			0.29 - 0.33		0.30 - 0.75	1	د.ن 45.» د.غبات - 45.»		+ - M		
- ARGASS #@T(LBS)	17.5	1.4	17	3ELT) TI	<del>.</del>	17	7 I RE :	14	f 43	17.5	17.5	6.71	7.7		· - 4		
Ficter	/ Plasticizer	olack J.o Carbon Black	3.5 Carbon 3.ack	TASTREVE (CAST CARCASS REPLACEABLE TREAD BELT) TIRE	c.o Carbon	black Black	(GLUED TREAD BELT) TIRE	; ;	•	Flasticizer		1 000	oldss	i/s" Strand [1]sb sizins]	5 Chopsed	المادية المادية Strand	(chizis legion)
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CARCASS THICRRESS MOLDING TREAT SEL RAGE(AVE)(IY) _ CYCLE* GOTYCHS	0.36 - 0.62	0.39 - 0.62	6.9 <b>-</b> 55.0	42 - 6.79		1		0.30 - 0.56		1	1		ı	0.33 - 0.61	0.39 - 0.66	0.37 - 0.67	
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15.7.3% \$3.8% \$3.8%	***	<u></u>	₽ .4	t v a		. J		22					2.7	, ż	67	.S.	

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<b>4</b>	A Contract of the Contract of		
	<pre>Minites yes time that Op Minites yes adding - doors or a Minites collegionabler and placer; Minites time water stray Minites tall water stray Minites collegionabler - blowers on</pre>	<b>5</b> 4	N. Pr
	l. Minyres gwer time i amôt c. Minutes conlin: nimber - doors open	14/A	A/A
	<ul> <li>throstes over the county Of</li> <li>throates even three tests</li> <li>Minutes and med water spray</li> <li>Minutes over county - doors men</li> </ul>	14/A	N/A
A months	on I recensive Replace April 1864 (1961) TIRE:		
;	o. Minutes byen time to more c. Minutes cooling chamber	75.0 Min c 290 <sup>0</sup> F	%/A
	to Minutes oven time / 4009F In.D Minutes oven time d obJ9F In.D Minutes oven cooling with door open 5 Minutes air and water spray	90.0 Min 3 2300F	N/A
DATE ARAL ()	LAST CARCASS/GLUED TREAD BELT) TIRE:		
κ.	G.5 Minutes oven time @ 640°F G.5 Minutes oven cooling - doors open i.d Minute fine water spray io.5 Minutes full water spray	60. J. Min + 2900r	Whittaker OS-N/2
<i>i</i>	PELS Minutes oven time # 6509F 6.5 Minutes oven cooling - doors open 1.0 Minute fine water spray 19.5 Minutes full water spray	ნმ.მ <b>''in</b> მ 290 <b>0</b> F	Whittaker "Thixon" D-3822 Primer and CB-3166 cover coat
	Zolo Minutes oven time & 6500f Dio Minutes oven cooling - doors open Ilo Minute time water spray Ible Minutes full water spray	60.0 Min ⊕ 290 <sup>0</sup> F	Whittaker OS-N/2
,	e.5 Minutes oven time (1650 <sup>0</sup> f e.5 Minutes oven cooling - doors open l.) Minute time water spray also Minutes full water spray	50.0 Min ≈ 2900F	Whittaker OS-N/°

## TAGE / CONTINGE PAST TIME DERMAN COST DATA

MOLUTNO CYCLE NR	CARCASS THERMAE CYCLE	TREAL SELT	BEET ASHESLA
13	34.u Minutes oven time of op. 91 J.b Minutes oven cooling - Joors open J.u Minute fine water spray Z1.0 Minutes full water spray	60.0 <b>Міп</b> 1 79а <sup>9</sup> Г	Whittaker 65-577
11	34.0 Minutes oven time № 6500F -0.5 Minutes oven cooling - doors open -1.0 Minute fine water spray -20.0 Minutes full water spray	60.9 Min a signof	Whittaken OS-147
12	33.0 Minutes oven time @ 6000F 0.5 Minutes oven cooling - doors open 1.0 Minute fine water spray 21.5 Minutes full water spray	45.0 Min 3 36 1 <sup>0</sup> 1	Whittaker Despui Privor ind CB-3165 Joven coat
15	33.0 Minutes oven time # 6000F 0.5 Minutes oven cooling - doors open 1.0 Minute fine water spray 24.0 Minutes full water spray	∌∍.d Min ⊤∴'9μαf	Whittaker D-ose. Primer and CB-3166 cover coat
1.4	34.0 Minutes oven time w 600°F 0.5 Minutes oven cooling - doors open 1.0 Minute fine water spray 24.0 Minutes full water spray	45.0 Min ⊕ 29∂0F	Whittaker D-852. Primer and CB-3166 cover coat
15	34.0 Minutes oven time @ 6000F 0.5 Minutes oven cooling - doors Open 1.0 Minute fine water spray 24.0 Minutes full water spray	45.0 Min @ 2900F	Whittaker D-3822 Primer and CB-3166 cover cpat
lo	Joi: Minutes oven time № 650°F Jis Minutes oven cooling - doors open I.J Minute fine water spray 25 Minutes full water spray	45.0 Min @ 290°F w/50 PSIG cure pres	Whittaker D-4822 Primer and CB-3160 cover coat
17	45.0 Minutes oven time 4 600°F 0.5 Minutes oven cooling - doors open 1.0 Minute fine water spray 25.0 Minutes full water spray	45.0 Min 3 290°F w/75 PSIG cure pres	Whittaker D-6622 Primer and CB-3166 cover coat
16	34.5 Minutes oven time M 650 <sup>0</sup> [ 0.5 Minutes oven cooling - doors open 1.5 Minute fine water spray 25.J Minutes full water spray	4510 Min @ 290 <b>0</b> F w/50 PSIS cure pres	Whittaker 0-0022 Primer and CB-3166 cover coat
19	34.5 Minutes oven time ∅ 650°F 0.5 Minutes oven cooling - doors open 1.0 Minute fine water spray 25.0 Minutes full water spray	45.0 Min H 290 <sup>0</sup> F W/50 PSIG cure pres	Hughson "Chemlock" AP-135 Primer and 40. cover coat

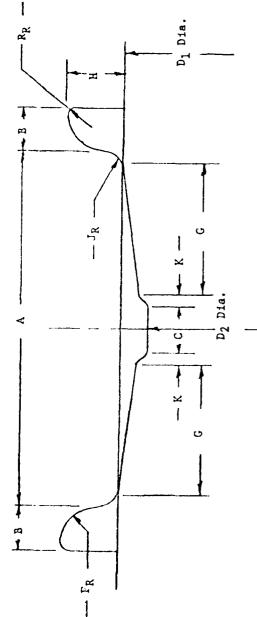
# TABLE 2 (CONTINUED) CAST TIRE THERMAL CUPL DATA

MÜLDING CYCLE NR	CARCASS THERMAL CYCLE	TREAD BELT CURE BELT ADMISIVE
Źu	34.5 Minutes oven time $^{(t)}$ 550 $^{O}$ f d.5 Minutes oven cooling - doors open l.0 Minute fine water spray 55.0 Minutes full water spray	45.0 Min Hughson ≈ 290°F "Chemlock" AP-133 w/50 PSIG Primer and 402 cure pres cover coat
21	36.5 Minutes oven time $\approx$ 000 $^{0}\mathrm{F}$ , 1.5 $^{-1}\mathrm{SIG}$ $4_{2}$ = 0.5 Minutes oven cooling - doors open, 1.5 $^{-1}\mathrm{SIG}$ N <sub>2</sub> = 1.0 Minute fine water spray, 1.5 $^{-2}\mathrm{SIG}$ 4 <sub>2</sub> = 25.0 Minutes full water spray, 1.5 $^{-2}\mathrm{SIG}$ 3 <sub>2</sub>	45.0 Min Hughson "Chemlock" AP-133 w/75 PSIG Primer and 402 cover coat
Ċζ	31.5 Minutes oven time $\approx 600^{0}\mathrm{F}$ , 1.5 PSIC N2 5.0 Minutes oven time $\approx 600^{0}\mathrm{F}$ , no N2 flow 0.5 Minutes oven cooling - doors open, 1.5 PSIG N2 1.0 Minute fine water spray, 1.5 PSIG N2 25.0 Minutes full water spray, 1.5 PSIG N2	45.0 Min Hughson  # 2900F "Chemlock" AP-133  w/75 PSIG Primer and 402  cure cover coat  pressure
23	47.0 Minutes oven time @ 550°F, 1.5 PSIG N <sub>2</sub> 1.0 Minute oven time @ 550°F, no N <sub>2</sub> flow 0.5 Minutes oven cooling - doors open, 1.5 PSIG N <sub>2</sub> 3.0 Minutes fine water spray, 1.5 PSIG N <sub>2</sub> 15.0 Minutes full water spray, 1.5 PSIG N <sub>2</sub> 1.0 Minute air circulation, 1.5 PSIG N <sub>2</sub>	60.0 Min Hughson @ 2900F "Chemlock" AP-133 w/75 PSIG Primer and 402 cure cover coat pressure
24	$43.0$ Minutes oven time 0 550°F, 1.5 PSIG N $_2$ 0.5 Minutes oven cooling - doors open, 1.5 PSIG N $_2$ 3.0 Minutes fine water spray, 1.5 PSIG N $_2$ 15.0 Minutes full water spray, 1.5 PSIG N $_2$ 1.0 Minute air circulation, 1.5 PSIG N $_2$	60.0 Min Hughson @ 290 <sup>0</sup> F "Chemlock" AP-133 w/75 PSIG Primer and 402 cure cover coat pressure
<b>2</b> 5	44.5 Minutes oven time @ 550°F, 1.5 PSIG N <sub>2</sub> 4.0 Minutes oven time @ 550°F, no %2 flow 3.5 Minutes oven cooling - doors open, 1.5 PSIG N <sub>2</sub> 3.0 Minutes fine water spray, 1.5 PSIG N <sub>2</sub> 15.0 Minutes full water spray, 1.5 PSIG N <sub>2</sub> 1.0 Minute air circulation, 1.5 PSIG N <sub>2</sub>	60.0 Min Whittaker @ 290°F "Thixon" P-8 w/75 PSIG Primer and 501 cure cover coat pressure
26	47.5 Minutes oven time @ $550^{\rm OF}$ , 1.5 PSIG N <sub>2</sub> 1.0 Minute oven time @ $550^{\rm OF}$ , no N <sub>2</sub> flow 0.5 Minutes oven cooling - doors open, 1.5 PSIG N <sub>2</sub> 3.0 Minutes fine water spray, 1.5 PSIG N <sub>2</sub> 15.0 Minutes full water spray, 1.5 PSIG N <sub>2</sub> 1.0 Minute air circulation, 1.5 PSIG N <sub>2</sub>	60.0 Min Whittaker @ 290 <sup>0</sup> F "Thixon" OS-N/2 w/75 PSIG (two coats) cure pressure
21	41.5 Minutes oven time 0 550°F, 1.5 PSIG N <sub>2</sub> 4.0 Minutes oven time 0 550°F, no N <sub>2</sub> flow 0.5 Minutes oven cooling - doors open, 1.5 PSIG N <sub>2</sub> 3.0 Minutes fine water spray, 1.5 PSIG N <sub>2</sub> 15.0 Minutes full water spray, 1.5 PSIG N <sub>2</sub> 1.0 Minute air circulation, 1.5 PSIG N <sub>2</sub>	60.0 Min Hughson @ 290°F "Chemlock" AP-133 w/75 PSIG Primer and 402 cure cover coat pressure

# TABLE 2 (CONTINUED) CAST TIRE THERMAL CURE DATA

MOLDING CYCLE NR	CARCASS THERMAL CYCLE	TREAD BELT	BELT ADHESIVE
ЗĦ	41.5 Minutes oven time @ 550°F, 1.5 PSIG N <sub>2</sub> 4.0 Minutes oven time 0.550°F, no N <sub>2</sub> flow 0.5 Minutes oven cooling - doors open, 1.5 PSIG N <sub>2</sub> 3.0 Minutes fine water spray, 1.5 PSIG N <sub>2</sub> 15.3 Minutes full water spray, 1.5 PSIG N <sub>2</sub> 1.0 Minute air circulation, 1.5 PSIG N <sub>2</sub>	60.0 Min M 290 <sup>0</sup> F W/75 PS15 cure pressure	Whittaker "Thixon" P-8 Primer and 501 cover coat
<b>29</b>	41.6 Minutes oven time @ 5709F, 1.5 PSIG N <sub>2</sub> 4.0 Minutes oven time @ 5709F, no N <sub>2</sub> flow 0.5 Minutes oven cooling - doors open, 1.5 PSIG N <sub>2</sub> 5.0 Minutes fine water spray, 1.5 PSIG N <sub>2</sub> 15.0 Minutes full water spray, 1.5 PSIG N <sub>2</sub> 1.0 Minute air circulation, 1.5 PSIG N <sub>2</sub>	60.0 Min @ 2900F w/75 PSIG cure pressure	Whittaker "Thixon" P-8 Primer and 501 cover coat
30	44.0 Minutes oven time @ 560°F, 1.5 PSIG N <sub>2</sub> 0.5 Minutes oven cooling - doors open, 1.5 PSIG N <sub>2</sub> 3.0 Minutes fine water spray, 1.5 PSIG N <sub>2</sub> 15.0 Minutes full water spray, 1.5 PSIG N <sub>2</sub> 1.3 Minute air circulation, 1.5 PSIG N <sub>2</sub>	60.0 Min @ 29n0r w/75 PSIG cure pressure	Whittaker "Thixon" P-3 Primer and 501 cover coat

TABLE 3 CONTOUR AND DIMENSIONS OF 7..05-8 WHEEL



Standard Rim Contours for Type 'III Aircraft Tires

	82	υ	D <sub>1</sub> Dia.	D <sub>2</sub> Dia.	er er
5.50	0.656	0.925	8.30	7.381	0.406

F.	0.052
J <sub>R</sub>	0.750
J.	0.203
н	C.812
Ŋ	1.400

Note: All Dimensions in Inches

TABLE 4

TIRE CONTACT AREA DATA
CAST TIRE FOOTPRINTS @ 125 PSIG

DEFL	32.41 21.24	26.80 18.24	32.77 22.34	30.57	25.24 16.22	22.95 14.64
NET AREA (IN <sup>2</sup> ).	44.26 29.85	22.90 14.45	30.60 17.47	28.76 17.13	36.85 25.50	35.74 23.82
GROSS AREA (IN <sup>2</sup> )	49.48 34.55	28.93 16.93	37.96 23.17	35.72 21.91	43.36 30.44	40.76 28.76
WIDTH (IN).	6.14	5.23	6.00	5.72	6.49 6.24	6.37
LENGTH (IN)	9.34 7.61	7.17	8.46	8.02 6.38	7.63 5.95	7.27 5.78
LOAD (LBS)	6650 3990	6650 3990	6650 3990	6650 3990	6650 3990	6650 3990
N/S	0360	A077A1	A097BB1	A028C4	B097A3	602884
TYPE DESIGN	STD TIRE	ONE PIECE TIRE	ONE PIECE TIRE	ONE PIECE TIRE	TWO PIECE TIRE	TWO PIECE TIRE

TABLE 4 (CONTINUED)

TIRE CONTACT AREA DATA
CAST TIRE FOOTPRINTS @ 125 PSIG

% DEFL	25.99 17.14	26.06 17.13	20.79 15.93	25.64 17.06	20.43 13.71	23.22 15.71	19.07 13.01	15.60 12.77
NET AREA (IN <sup>2</sup> )	37.46 25.49	36.19 24.01	35.31 23.78	38.02 25.61	34.38 23.80	36.07 23.91	30.77 20.07	31.95 20.23
GROSS AREA (IN <sup>2</sup> )	41.19 29.21	40.83 28.23	38.98 26.47	42.08 29.39	38.04 26.42	38.99 26.96	34.23 22.84	34.80 22.50
WIDTH (IN)	6.13 5.94	6.16 5.94	6.16 5.75	6.19 6.00	6.19 5.81	6.19 5.78	6.06 5.38	6.25 5.31
LENGTH (IN)	7.56 6.16	7.90 6.06	7.31 5.94	7.75	7.00	7.25 5.94	6.75 5.44	6.38 5.38
LOAD (LBS)	6650 3990	9990 3990	9960 3990	990 3990	6650 3990	6650 3990	9990 3990	6650 3990
INTEGRAL TIRE S/N	B068C1	B078C4	В088Н3	B088I4	B088J3	B088K4	B098L3	B098M2

TABLE 4 (CONTINUED)

TIRE CONTACT AREA DATA
CAST TIRE FOOTPRINTS @ 125 PSIG

_J, tu , tu , Cu ,	19.5¢ 13.5°	कुक चुक दुक दुक लो	29.32 19.24	19.37	32.17	19.45 13.66	18.46	25.30
NET AREA (IN2)	33.55 21.04	35.19 21.73	40.55	33.76 22.56	44.12 29.30	<b>31.1</b> 2 20.82	30.49 19.42	38.77 26.56
GROSS AREA (IN <sup>2</sup> )	37.17 24.78	38.00 24.86	45.27 30.96	37.62 25.69	48.58 34.2 <b>6</b>	34.46 23.43	33.30 22.18	43.04 30.17
WIDTH (IN)	6.25 5.63	6.31 5.63	6.50 6.25	6.38 5.75	6.63 6.44	6.19 5.34	6.19 5.25	6.37 6.06
LENGTH (IN)	6.88 5.75	6.94 5.50	8.00	7.00 5.75	8.63 6.83	6.75 5.56	6.63 5.19	7.75
LOAD (LBS)	6650 3990	6650 3990	6650 3990	6650 3990	6650 3990	6650 3990	6650 3990	6650 3990
INTEGRAL TIRE S/N	B098N2	809802	B098P2	809843	8098R2	609853	609812	612803

TABLE 4 (CONTINUED)

TIRE CONTACT AREA DATA CAST TIRE FOOTPRINTS ~ 125 F515

		CAST TIRE FO	AST TIRE FOOTPRINTS ~ 125 FSTS	C		
INTEGRAL TIRE S/N	LOAD (LBS)	LENGTH (IN)	WIDTH (1W)	GROSE AREA (IN2)	Fred C	
8128V3	98650 3990	7.18 6.31	6.25 5.69	42.39 23.39	C. १ च १ इ १ ०,	
6029W3	6650 3990	7.50 6.00	6.31 6.00	39. At 27. 15		
B029x3	96650 3990	7.38 6.06	6.31 6.04	다. () () () ()	7 7 7	
8029Y3	665û 399û	7.02 5.91	е Ж. В. В.	î. <del>ŝ</del>		
802923	665€ 399∪	2.72	\$ .20 \$ .56	in the second se		
p.c3#63.a	გგგე	7.7.	0.35 0.15			
है उसे पुरुष	නික්ති 2998	7.78	(			
5 7 <del>3</del> 7 9	\$ 7 \$ 20 \$ 20 \$ 20 \$ 20 \$ 20 \$ 20 \$ 20 \$ 20	7.07 6.16	· ;		 	
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78.5. TELEMIA

THE PASS (SE	critis (Bilus - Grown & Tubewal) britis Failus - Crown & Sidewal)	Material Greek - Lutboard Center Shoove	Brittle Failure - Sidewall	Material Greep - Imboand Sead Extruded	Orittle Failure - Bead Gracked	Material Creep - Crown & Selt Edge Extruded	Material Creep - Crown Extruded/Belt Broke	Material Creen - Crown Extruded/Belt Broke	Material Creep - Bead Failure	Haterial Creep - Inboard Sidewall	Material Greep - Selt Edge Extruded & Gracked	Material Oreen - Belt Edge Extruded & Oracked	Brittle Failure - Outboard Sidewall	Brittle Failure - Outboard Sidewall	Brittle Failure - Crown/Some Extrusion
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	974[1-40]	19.54 - 18.55	ئيء أج−ق!	\$30 (1-1-3K)	140-116.	integral Tire	integral line	integral line	Futegral Ince	integral line	Integral Tire	integral Tine	integral line	integral line	integral Tine

\* Jenz refers to Senzoflex, a plasticizer \*\* kapid or quick inflation rate

Minimus Required Surst Pressure - 441 PSIS Approximate Inflation Rate - 30 PSIG/MIN

TABLE ? LUTILEE.

	Magnerial mass a subboard has fathuded	Material Order - Inwar Extraday (alt bross	Material Green - Jutopuru Sidewal' Extruded	Material Orger - Tutboard Gead Extruded	Material (reeg - Sutboard Dead Estruded	Unittle Failune - Orden & Sidewall Selt Ordee	Smittle Failune - Jutboard Sidewall	Smittle Failure - Crown v Sidesail Smit smake	brittle Failure - Crowr & Sidewall Selt Prore	Brittle Failure - Crown & SidewallySelt Broke	Material Greep - Crown & Sidewall Extruded	Smittle Failune - Selt Edge Chasked	Brittle Failure - Crown & Sidewall/Selt Pr⊃∙e	Brittle Failure - Crown & Belt Edue/Belt Froke	Pubber Valve Failune - Valvo Slow of	Material Greep - Outboard Seat Extrade:	Rubber Valve Failure - Valve like lut
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chikonsis Jaat Julos,	.3	77	ŝ	24	77	~1	2,	17	17	18	: :	93	10	17.	17.	13.	≈ •
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न्त्रहेत्वा चार्		Integral Tire									Integral Tire					integral Tire	Integral Tire

\* benz refers to Benzoflex, a plasticizer Minimur Bequired burst Pressure - 441 PSIS Autroximate inflation Rate - 30 PSIG, MIN

TABLE 8
LATERAL FORCE (QUASI-STATIC TFM DATA)

<u>C.</u>	(4) (4) (5)	CCS.	3968 3968	0.27	ক গ্ৰ	(S)24	2690	7.	() () ()	5967	2700	2730	tu ni ng	,-,, , f - , , j	€ € •€ • • 1	1	, Ā	F I	;	1
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0 <sup>0</sup>	1880	1750	2650	2220	1900	lou(	1950	2050	125∪	2200	2130	2230	2050	1650	1950	]   150 	1900	5000	1900	1905
30.553aa 10177341 o <sup>9</sup> 1	1440	1300	2250	2100	1550	1430	1550	1650	ეგ6	1750	1700	1750	160ປ	1250	1500	1250	1500	1580	1500	1500
AMD 128	086	350	1750	1600	1100	1050	1100	1150	ევ9	1250	1200	1250	1100	850	1050	340	1100	1100	1050	1050
0F 6650 LBS 20	200	400	1000	350	600	580	9009	009	350	009	009	009	550	400	580	450	580	580	550	650
3 VERTICAL LOAD SLIP AMSLE SURFACE CONDITION	DRY	WET	DRY	WET	ORY	WET	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY
LATERAL FORCE (LBS)	BIAS TIRE	E BIAS TIRE	ECE TIRE S/N	CCE TIRE S/N	E TIRE S/N	ECE TIRE S/N	L TIRE S/N	INTEGRAL TIRE S/N BO88J2	TIRE S/N	TIRE S/N	TIRE S/N	. TIRE S/N	INTEGRAL TIRE S/N B09802	. TIRE S/W	. TIRE S/N	TIRE S/N	TIRE S/N	TIRE S/N	TIRE S/N	INTEGRAL TIRE S/N 8128V3

TABLE 9
ALIGNING TORQUE (QUASI-STATIC TFM DATA)

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PSIG INFLATION PRESSURE 60 80	255	245	165	122	210	70	180	165	215	220	195	235	200	195	125	185	82	100	175	165
<b>&amp;</b> 125 P	175	172	195	125	185	55	143	135	170	155	140	180	145	135	110	140	09	80	130	125
0F 6650 LBS	92	90	170	155	135	25	85	65	105	65	70	98	70	65	65	75	35	09	75	75
@ VERTICAL LOAD SLIP ANGLE RFACE CONDITION	DRY	WET	DRY	WET	DRY	WET	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY	DRY
ALIGNING TORQUE (FT-LBS) SU	ВІ	AS TIRE S/N	TIRE S/N A02	TIRE S/N A02	S/N B02	TIRE S/N 802	L TIRE S/N B08	. TIRE S/N B08	L TIRE S/N B08	TIRE S/N B09	_ TIRE S/N B09	. TIRE S/N B09	_ TIRE S/N B09	. TIRE S/N B09	. TIRE S/N B09	. TIRE S/N B09	L TIRE S/N B09	. TIRE S/N B09	S/N 812	INTEGRAL TIRE S/N B128V3

148LL 10

	) FAILURE MODE		Groove Failure Groove Failure Groove Failure		Tread Selt Derailment Tread Selt Derailment		Showlder 3 Belt Edge	Shoulder a But Edge & a Bead	U.B. Shoulder 3 Selt Edge Outboard Sead Extruded	Successfully Completed Taxi	O.B. Showlder & Belt Edge	Both Shoulders 3 Self Edge	ı aı	0.8. Sidewall & Shoulder & Selt Edge Inhouse Boad Esteedad	Mibbard bead Extraded Split Crown Under Tread	O.B. Bead Extruded	Blowout @ Crown & Shoulder	Blowdut 3 Crown & Shoulder Blowdut 3 Crown & 0.5. Shoulder	Outboard Bead Extruded	Outboard Sidewall	Intlation Valve Falled	Infraction valve raffed Valve Failed During Inflation	Inflation Valve Failed	inflation Valve Falled Inflation Valve Falled	Valved Failed During Inflation
	(MILES)		(0.35) (0.20) (0.14)		(9.22) (0.61)		(0.37)	(0.37)	(0.51)	(2.65)	(0.44)	(0.30)	(0.30)	(0.40)	(1.80)	(1.10)	(1.00)	(3.50)	(1.46)	(1.13)	(0.34)	(0.33)	(0.56)	(0.31) (0.10)	:
TAXI TEST DATA, (30 MPH TAXI SPEED)	TAXI MILEAGE - FAILUPE		la.	1) TIRE:	.1		•																	I.	
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	TEST PRESSURE 1 (PSIG)		125 125 125	S/REPLACEABLE	125 140	GLUED TREAD E	134	134	134 134	134	134	134	134	134	134	134	134	1 5 T	140	134	134	134	134	134	134
	TEST LOAD (LBS)		5650 6650 6650	CASS/RE	0999 0999		6650	665U	0690 6650	6650	5650 5650	6650 6650	6650	5650 6650	0630 6630	6650	665U	6650	9650	6650				6650	0630
	IR SERIES NR	CE CAST TIRE	AUZZA(X) AUZZG(X) AUZZG(X)	CE CAST CARCAS	8097A(X) 80258(X)	CAST CARCASS	8065C(X)	8075C(X)	8088I(X)	B0383(X)	BOOKK(X)	8096M(X)	8098N(X)	80980(X)	80980(X)	B093R(X)	80985(X)	8128U(X)	B128V(X)	B029W(X)	BUZ9X(X)	B029Z(X)	B029AA(X)	802988(X) 8029CC(X)	8023DD(X)
	DESTGN VR	ONE-PIEC	નિલ્લ	TWO-PIEC	មាខ	INTEGRAL	a,		וכים	ા:	11	13	77	다. 다.	17	2:	<u>a</u> :	21	22	23	<b>2</b> ′.	97 97	72	87 67	30

TABLE 11
TACT TEST DATA
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SERIES NR		***		· · · · · · · · · · · · · · · · · · ·	
15. 55. 54. A.	(3)	7	~		:

TANEE 1. TANEE 1837 DATA EFFECT OF TEST POR SAURE LAMPED A.

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PRESSURE 7815.	-1	33.	***			: †
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TSTE SHEET WEET	247	e Tens	S.	- 1 - 125	7	v.
22 23 24 25 20 20 20 20 20 20 20 20 20 20 20 20 20			· Maria			\$ 35.4 B
JESTON NR	*		1 -	r . 	•	F

faiso injudas effect of reduced test 1743

APPENDIX B
FIGURES AND PHOTOGRAPHS

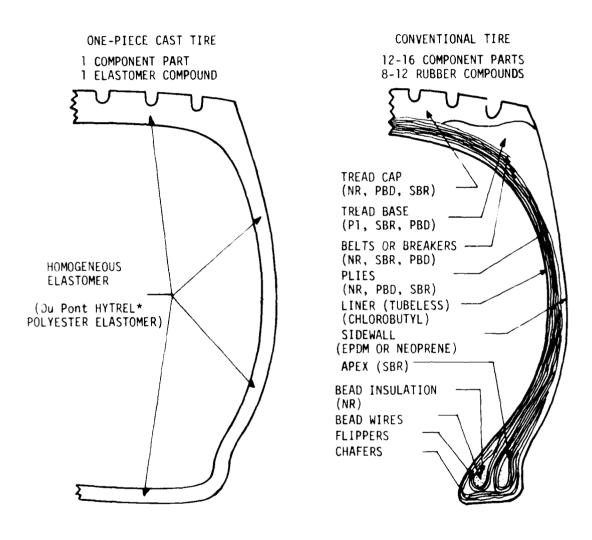


Figure 1. One-Piece Cast Tire Vs Conventional Tire-Comparison



Figure 2. 7.00-8 One-Piece Cast Tire

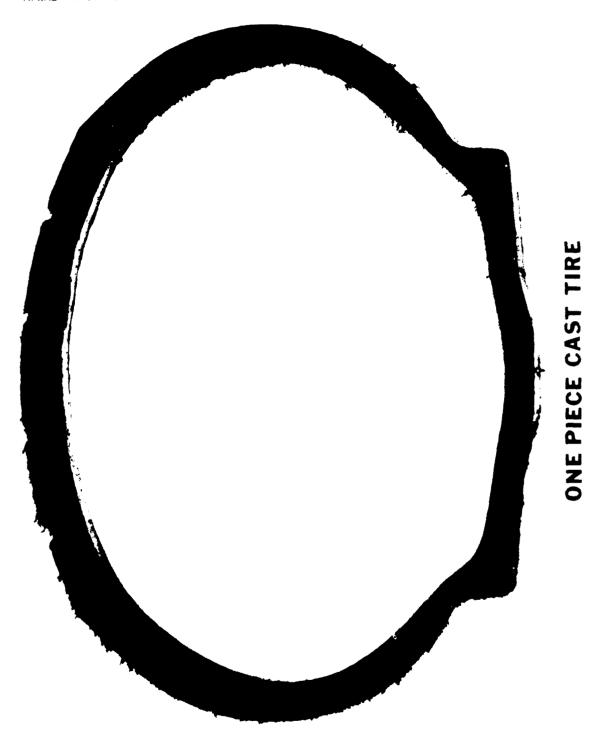


Figure 3. 7.00-8 One-Piece Cast Tire (Section)



Figure 4. 7.00-8 Two-Piece Cast Tire



Figure 5. 7.00-8 Two-Piece Cast Tire (Carcass & Belt)

Figure 6. 7.00-5 Two-Piece Cast Tire (Carcass & Belt)

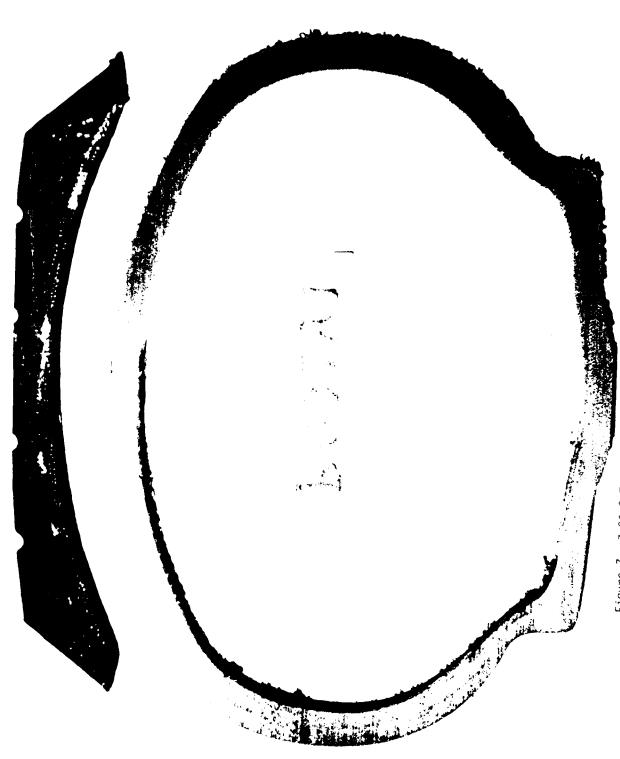
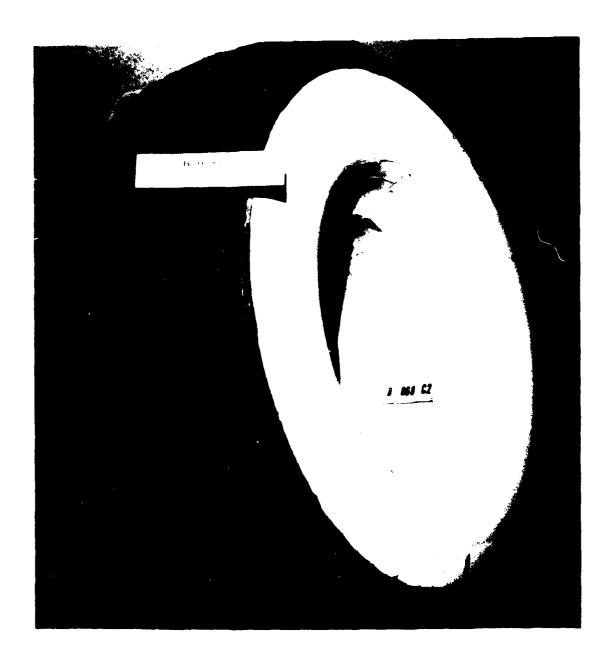


Figure 7. 7.00-8 Two-Piece Cast Tire (Carcass & Belt Section)



Timure 8. 7.90-8 Integral Cast Tire

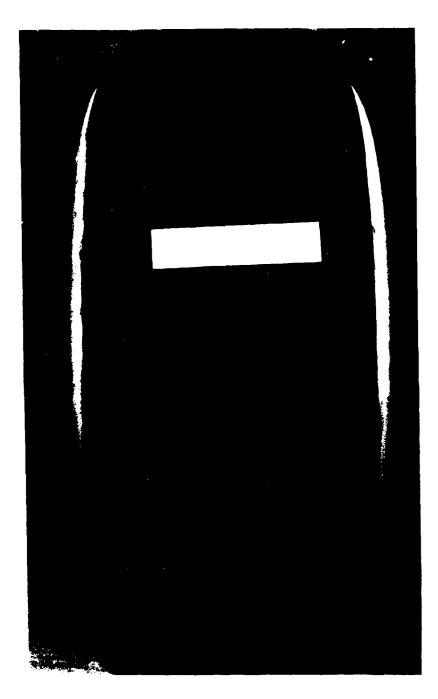


Figure 9. Integral Cast Tire (Tread)

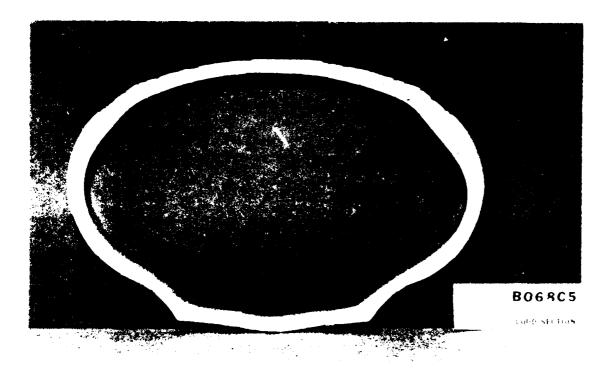


Figure 10. Integral Cast Tire (Section)



Figure 11. Baseline Bias Tire (7.00-8/16 PR)



Figure 12. Baseline Bias Tire-Section (7.00-8/16 PR)

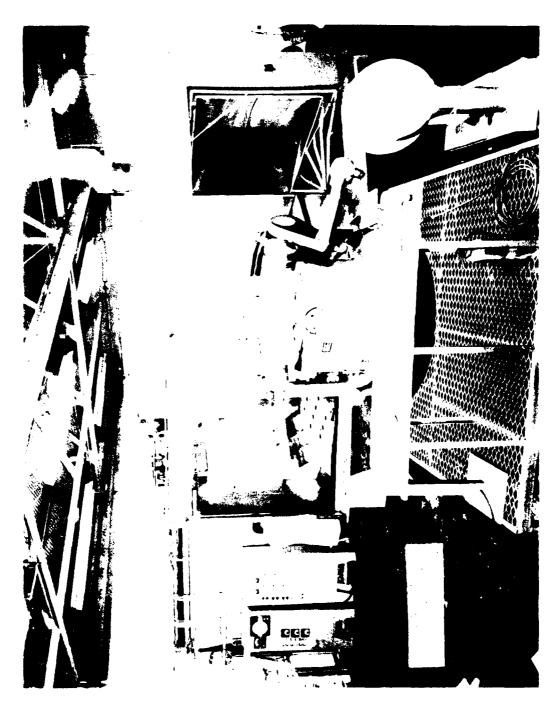


Figure 13. Rotational Molding Machine



Figure 14. Potational Mold fixture (Spider)



Figure 15. One-Piece Cast Tire Mold

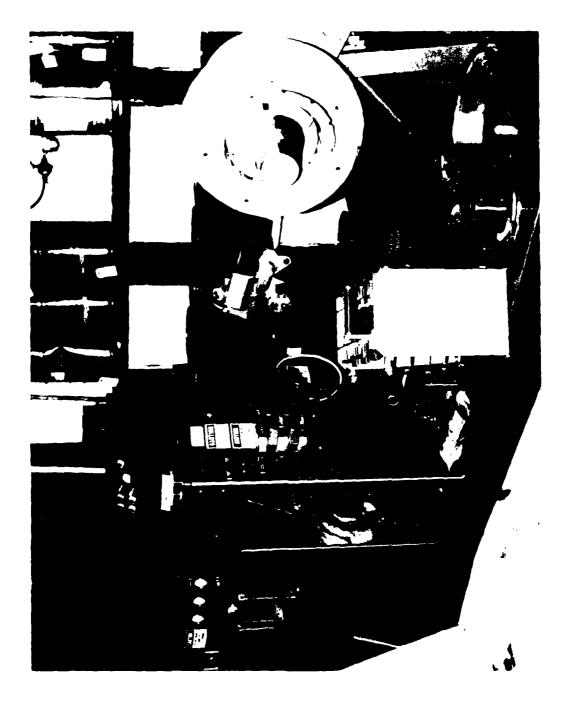
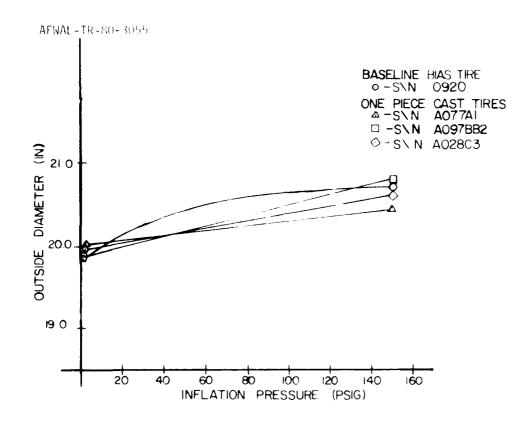


Figure 16. AMF Orbitread Machine



Figure 17. Tread Belt Mandrel & Mold



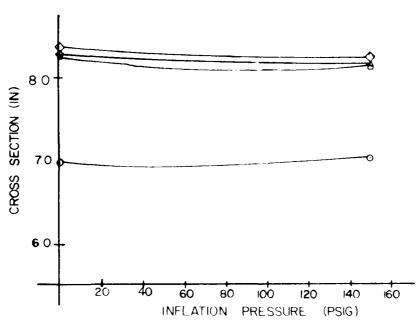
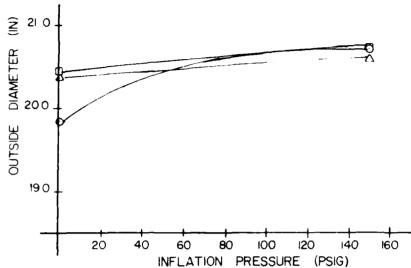


Figure 18. Static Growth Measurements - One-Piece Cast Tires





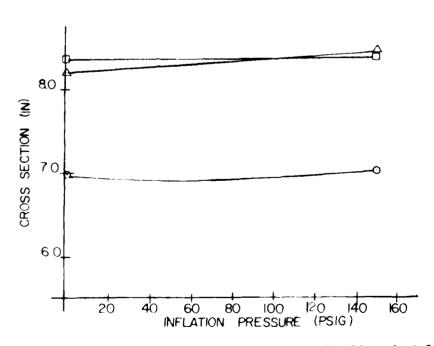
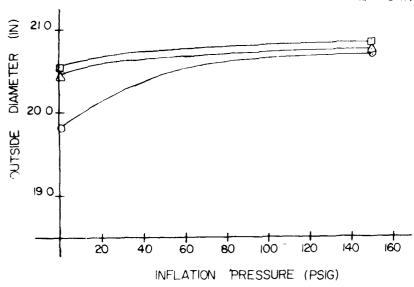


Figure 19. Static Growth Measurements - Two-Piece Cast Tires

BASELINE BIAS TIRE O - S\N 0920 INTEGRAL TIRES A - S\N B068CI D S\N B078C4



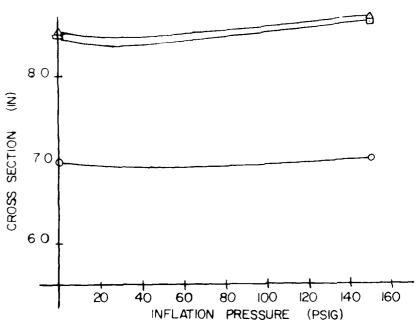


Figure 20. Static Growth Measurements - Integral Tires

BASELINE BIAS TIRE

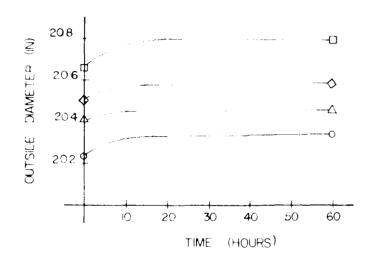
o - S\N 0920

ONE-PIECE CAST TIRES

△ - S\1: A077AI

□ - S\N A097BB2

◇ - S\N A028C3



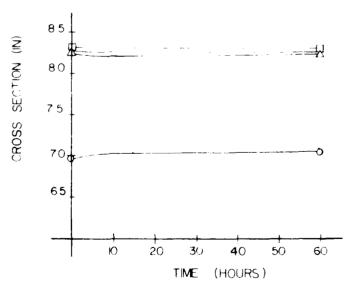
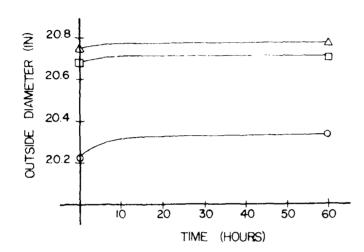


Figure 21. Dimensional Stability Data @ 125 (PSIG) Inflation Pressure - One-Piece Cast Tires

BASELINE BIAS TIRE
0 S\N 0920
TWO-PIECE TIRE

\$\Delta\$-S\N B097A3

\$\Omega\$-S\N B028B3



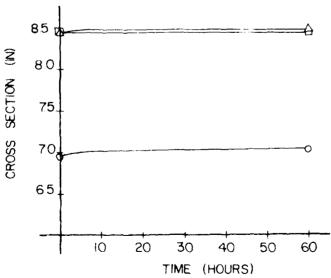
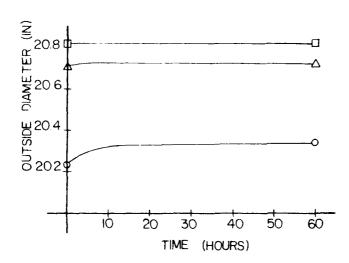


Figure 22. Dimensional Stability Data @ 125 (PSIG) Inflation Pressure - Two-Piece Tire

BASELINE BIAS TIRE
o-S\N 0920
INTEGRAL TIRE

a-S\N B068CI

-S\N B078C4



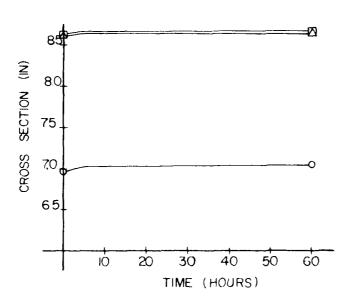


Figure 23. Dimensional Stability Data @ 125 (PSIG) Inflation Pressure - Integral Tire

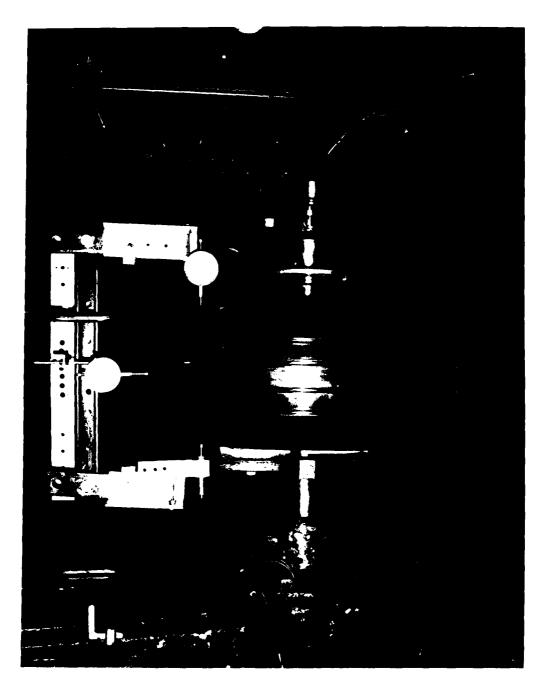


Figure 24. Dimensional Growth Tests (One-Piece Cast Tire)

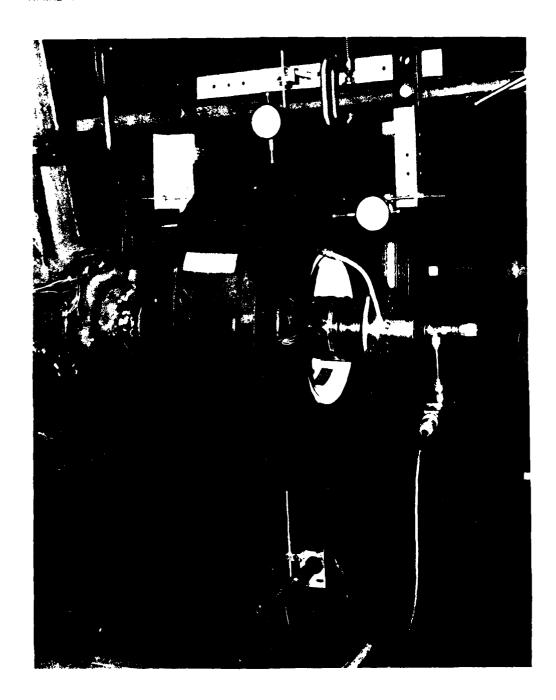


Figure 25. Dimensional Growth Tests (Two-Piece Cast Tire)

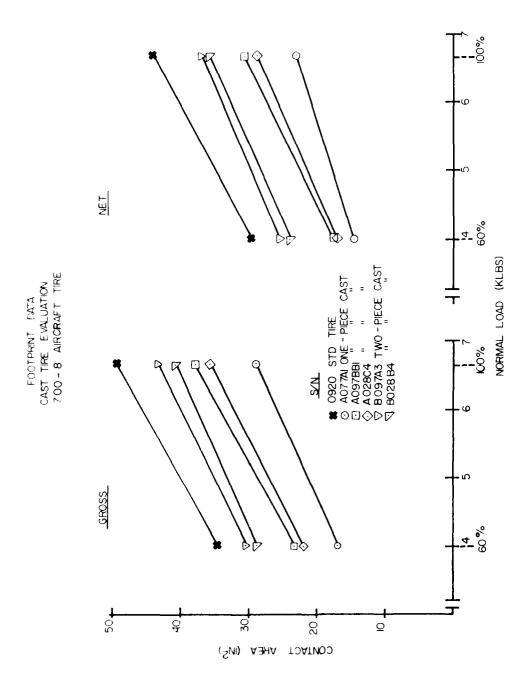
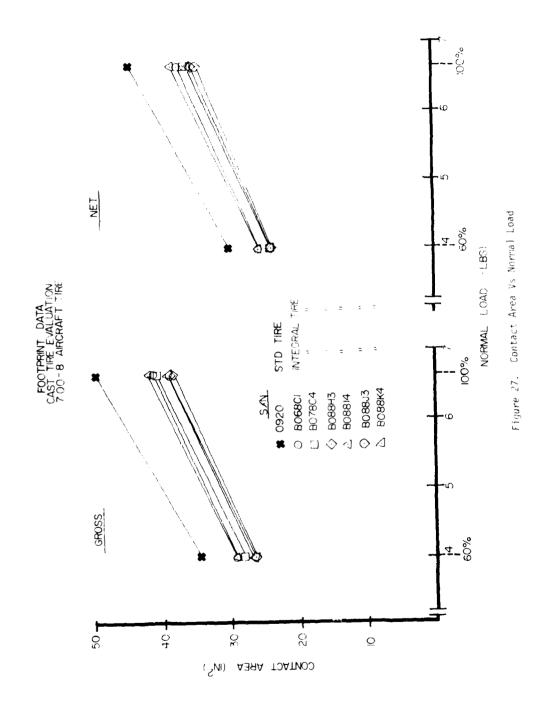
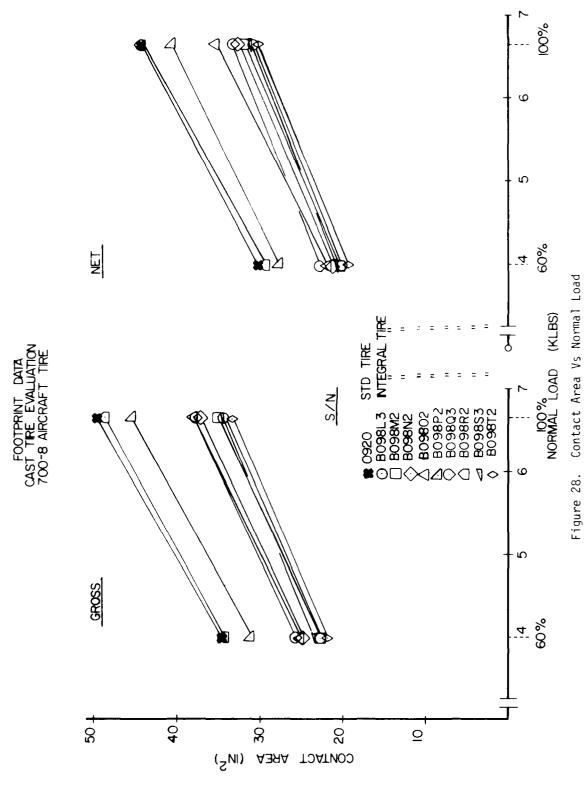
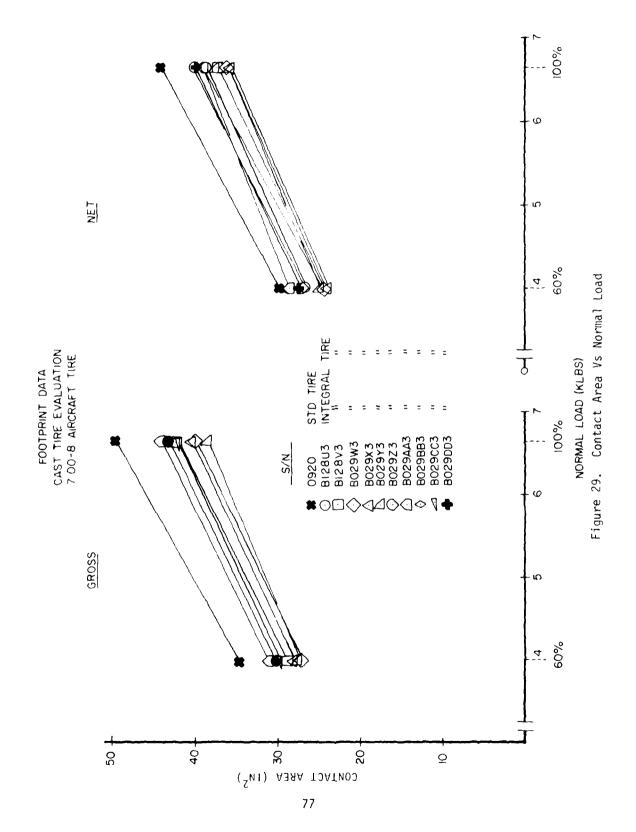


Figure 26. Contact Area Vs Normal Load







## AFWAL-TR-80-3055

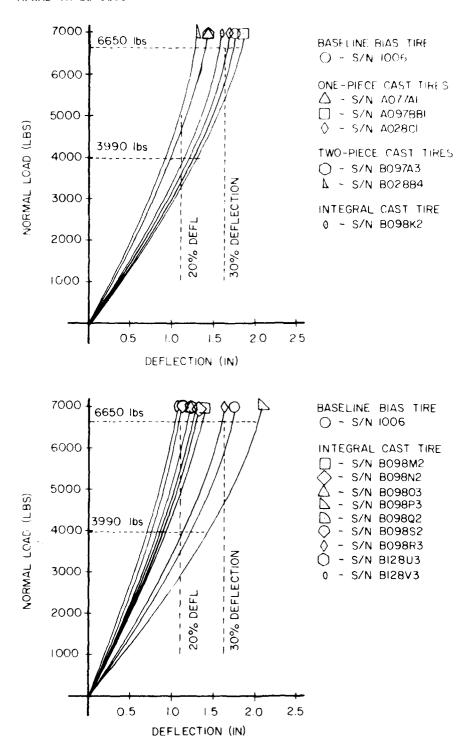


Figure 30. Vertical Load Vs Deflection @ 125 PSIG

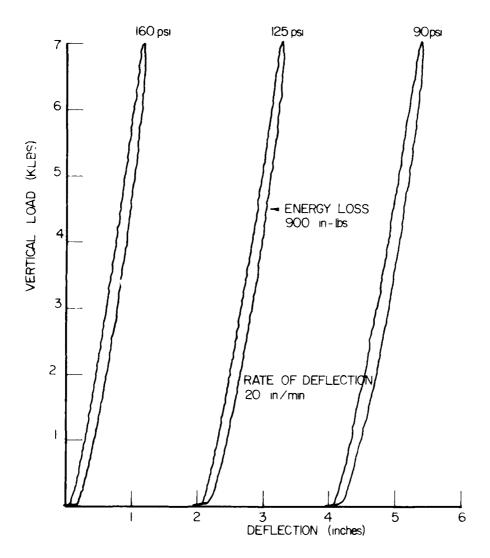


Figure 31. Vertical Load Vs Vertical Deflection, Integral Tire S/N G128U3

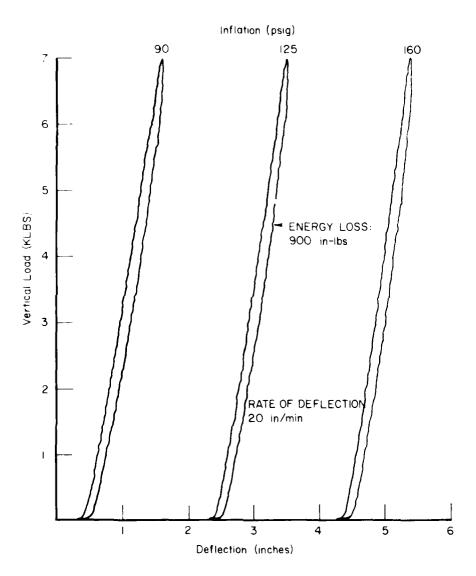


Figure 32. Vertical Load Vs Vertical Deflection, Integral Tire S/N B128V3

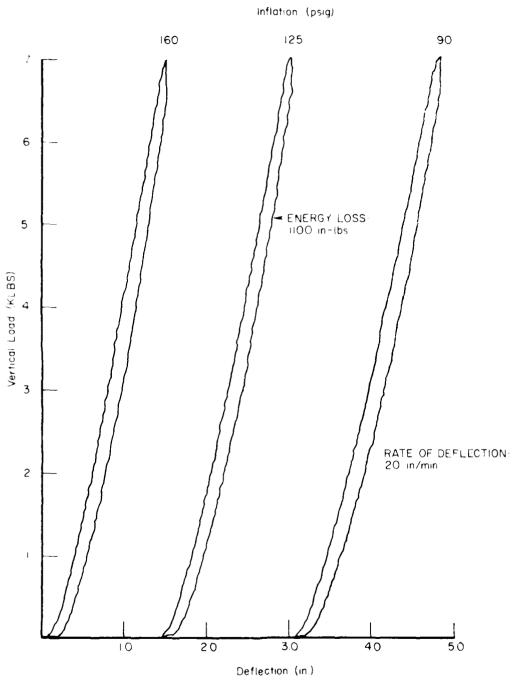


Figure 33. Vertical Load Vs Vertical Deflection, Integral Tire S/N B088I2

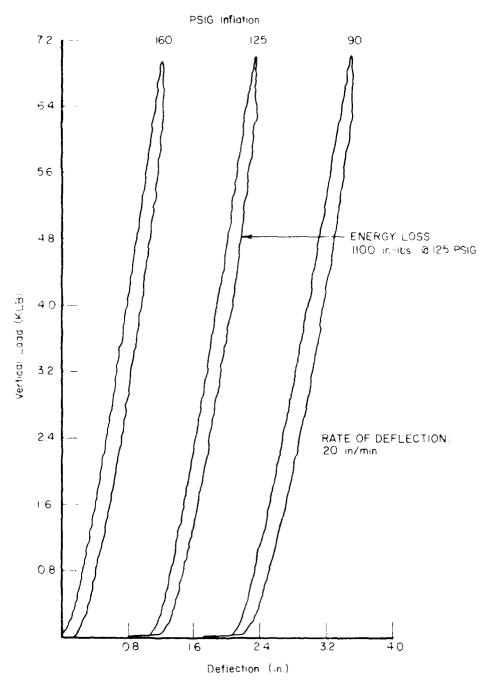


Figure 34. Vertical Load Vs Vertical Deflection, Integral Tire S/N B098L2

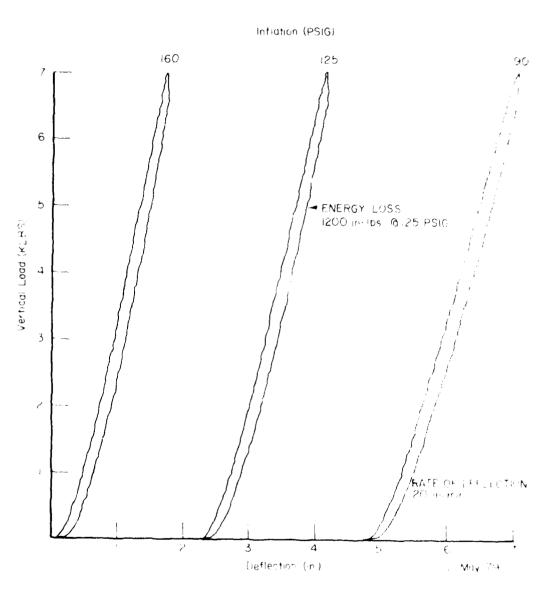


Figure 35. Vertical Load V. Vertical Deflection, Baseline of a Tire 5/N 1006

AIR FORCE WRIGHT AERONAUTICAL LABS WRIGHT-PATTERSON AFB OH F/6 1/3 STATIC AND DYNAMIC EVALUATION OF A-37 CAST AND CAST CARCASS/INT--ETC(1) AD-A097 684 NOV 80 P C ULRICH AFWAL-TR-80-3055 UNCLASSIFIED 2 of 4 3

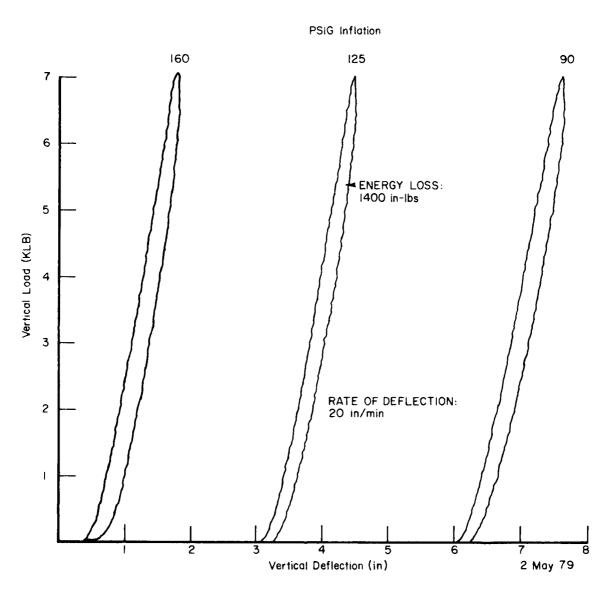


Figure 36. Vertical Load Vs Vertical Deflection, Integral Tire  $\ensuremath{\text{S/N}}$  B098M2

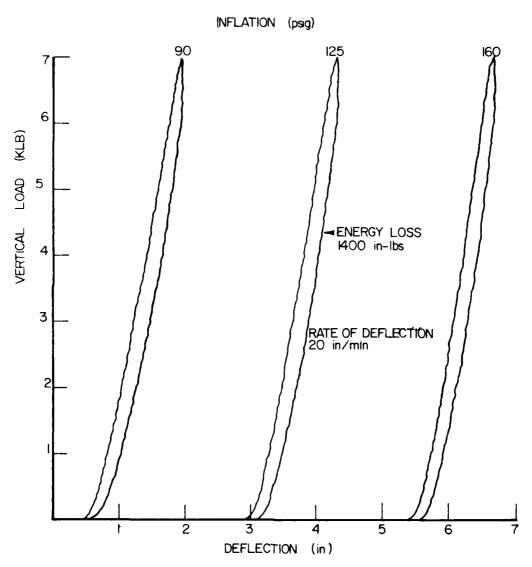


Figure 37. Vertical Load Vs Vertical Deflection, Integral Tire S/N B098N3

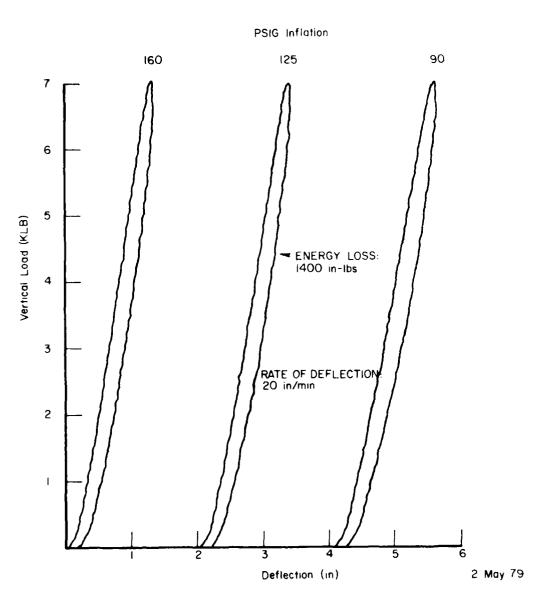


Figure 38. Vertical Load Vs Vertical Deflection, Integral Tire  $\,$  S/N B09802

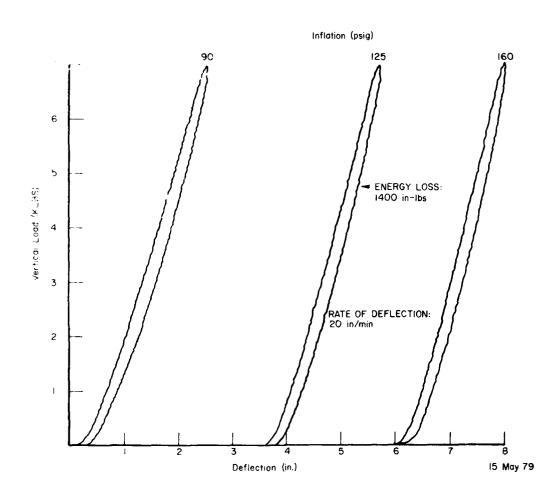


Figure 39. Vertical Load Vs Vertical Deflection, Integral Tire  $_{\mbox{\footnotesize S/N}}$  B098P3

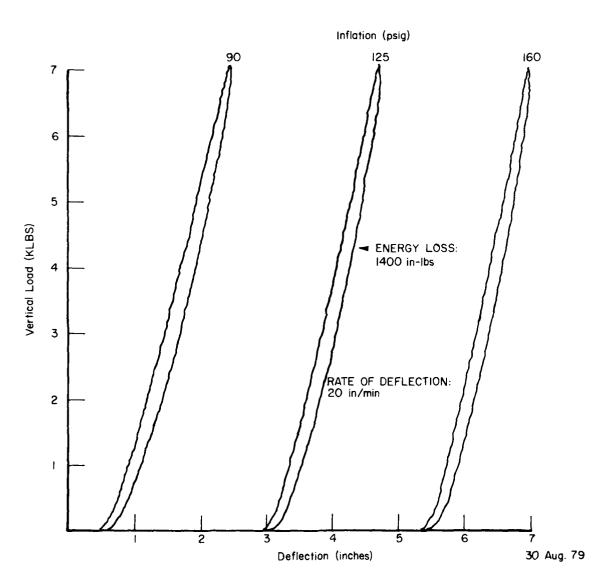


Figure 40. Vertical Load Vs Vertical Deflection, Integral Tire S/N B098R3

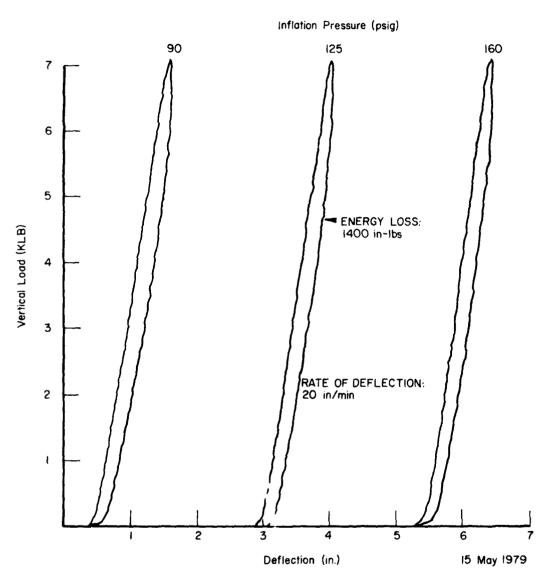


Figure 41. Vertical Load Vs Vertical Deflection, Integral Tire  $\ensuremath{\mathrm{S/N}}$  B098S2

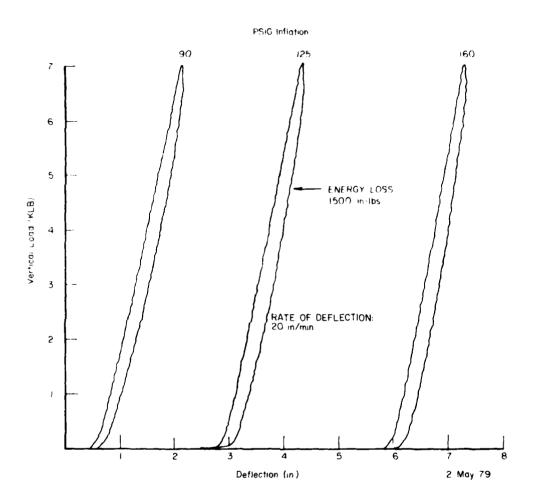


Figure 42. Vertical Load Vs Vertical Deflection, Integral Tire S/N B088K2

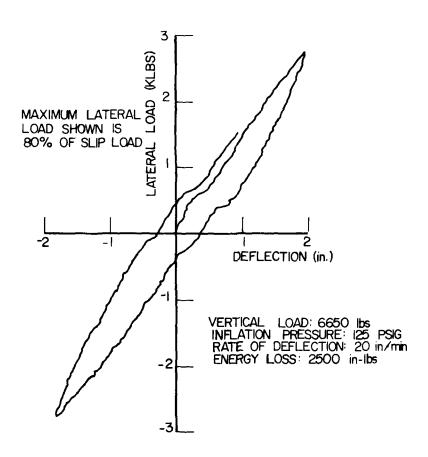


Figure 43. Lateral Load Vs Lateral Deflection, Integral Tire S/N B098S2

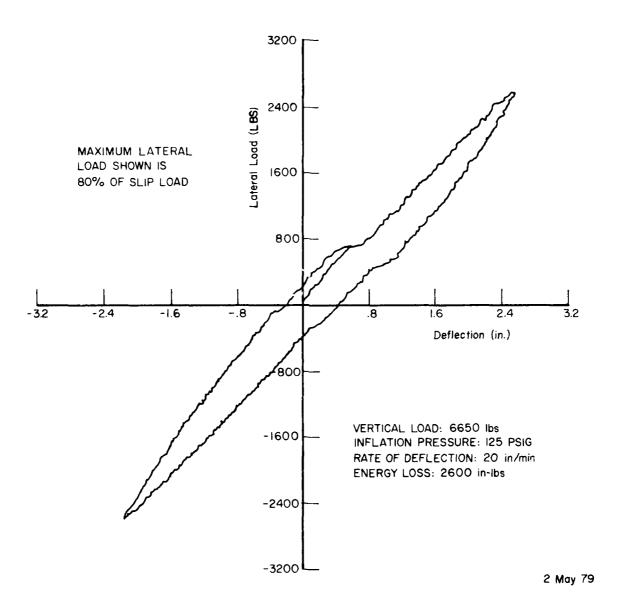


Figure 44. Lateral Load Vs Lateral Deflection, Integral Tire S/N B088I2

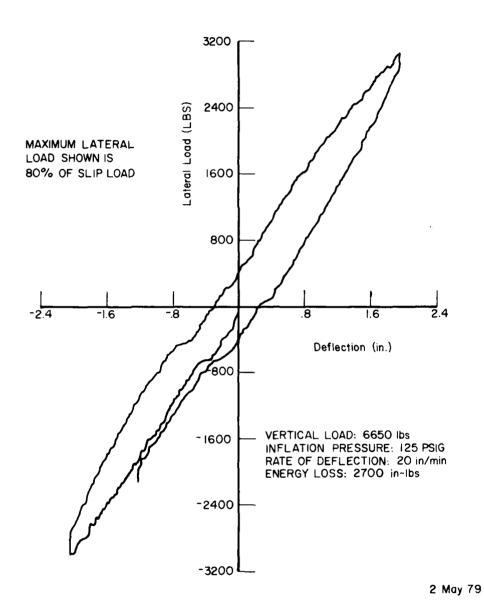


Figure 45. Lateral Load Vs Lateral Deflection, Integral Tire S/N B098L2

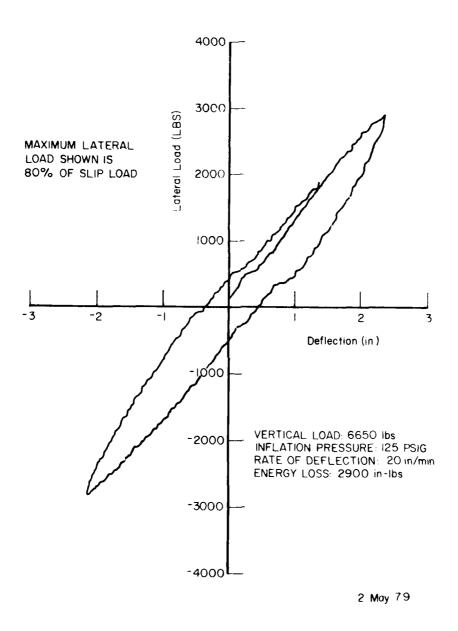


Figure 46. Lateral Load Vs Lateral Deflection, Integral Tire S/N B09802

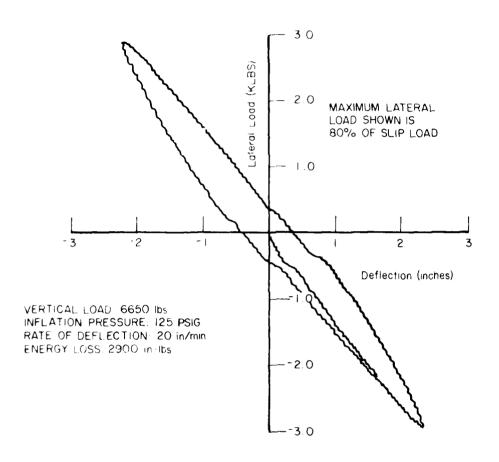
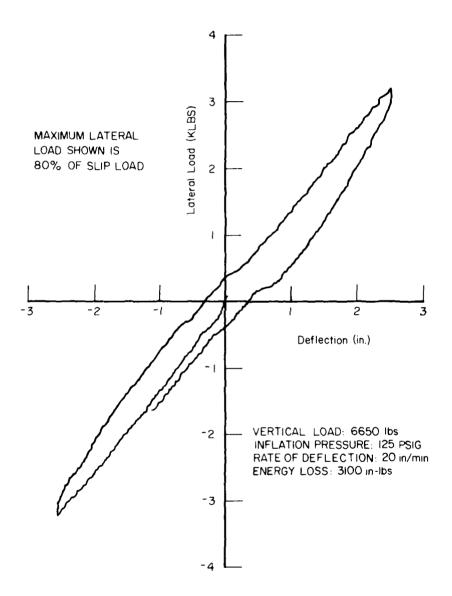


Figure 47. Lateral Load Vs Lateral Deflection, Integral Tire S/N B128U3



2 May 79

Figure 48. Lateral Load Vs Lateral Deflection, Baseline Bias Tire S/N 1006

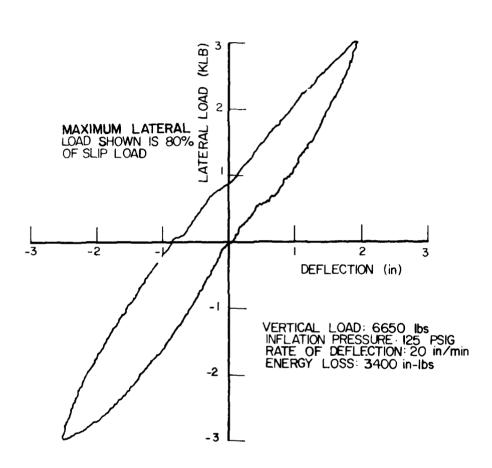


Figure 49. Lateral Load Vs Lateral Deflection, Integral Tire S/N B098M2

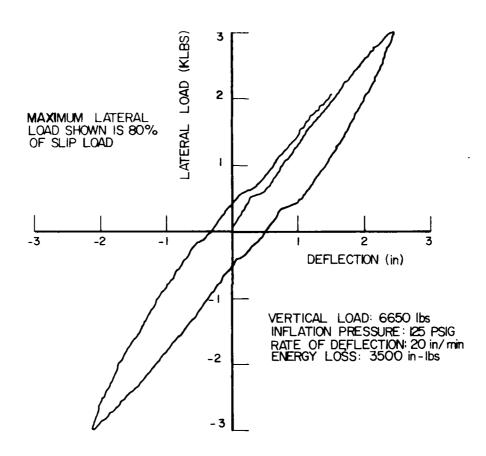


Figure 50. Lateral Load Vs Lateral Deflection, Integral Tire S/N B098N3

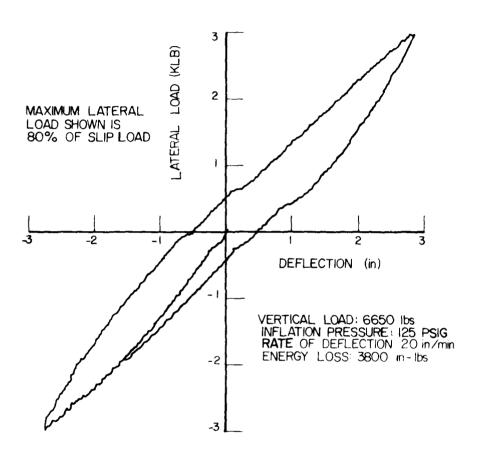


Figure 51. Lateral Load Vs Lateral Deflection, Integral Tire S/N B088K2

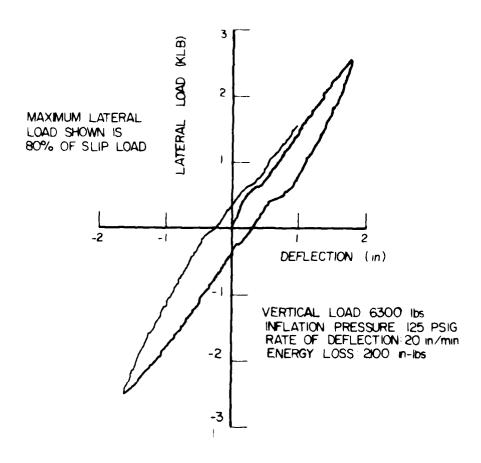


Figure 52. Lateral Load Vs Lateral Deflection, Integral Tire S/N B098S2

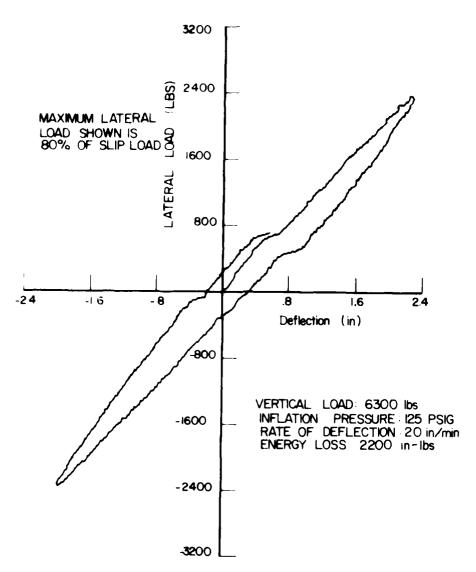


Figure 53. Lateral Load Vs Lateral Deflection, Integral Tire S/N B08812

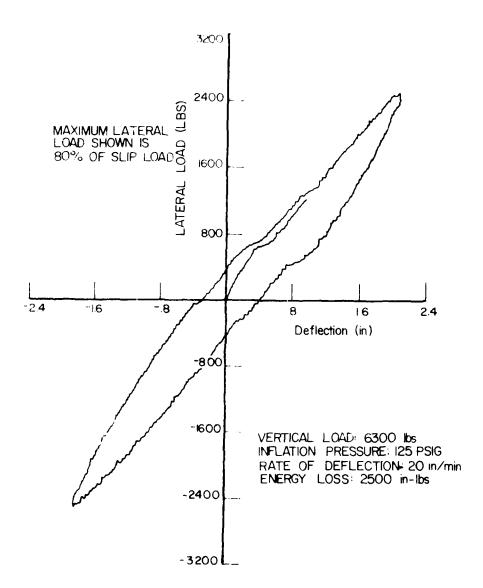


Figure 54. Lateral Load Vs Lateral Deflection, Integral Tire S/N 8098L2

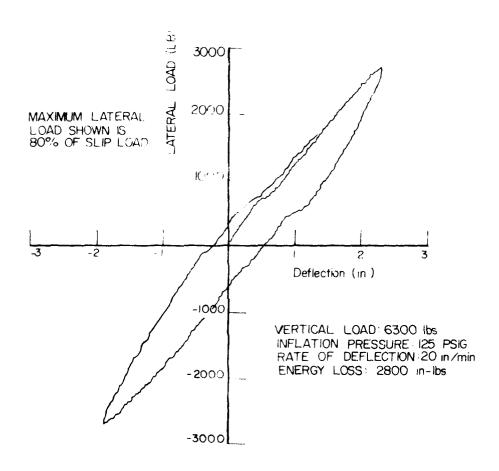


Figure 55. Lateral Load Vs Lateral Deflection, Integral Tire S/N B09802

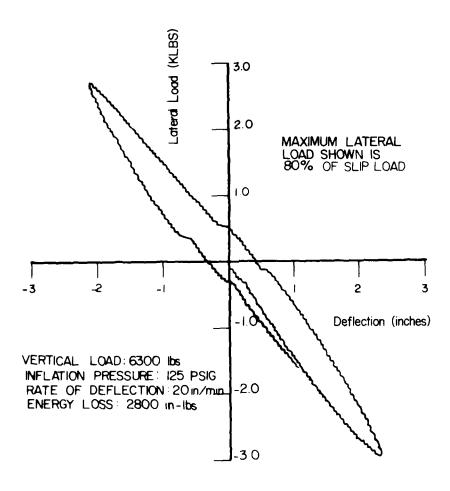


Figure 56. Lateral Load Vs Lateral Deflection, Integral Tire S/N B128U3

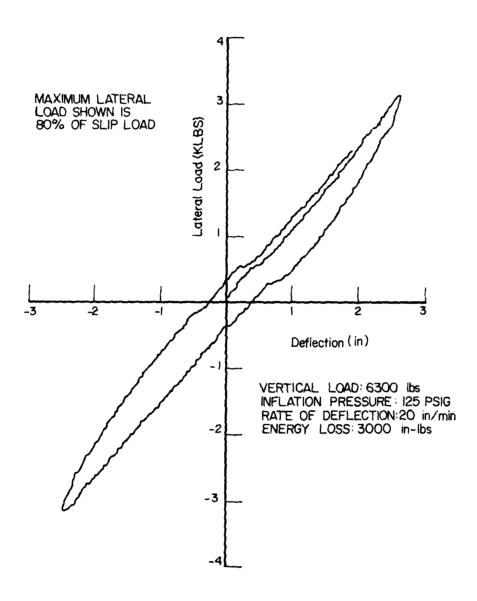


Figure 57. Lateral Load Vs Lateral Deflection, Baseline Bias Tire  $\mbox{S/N}$  1006

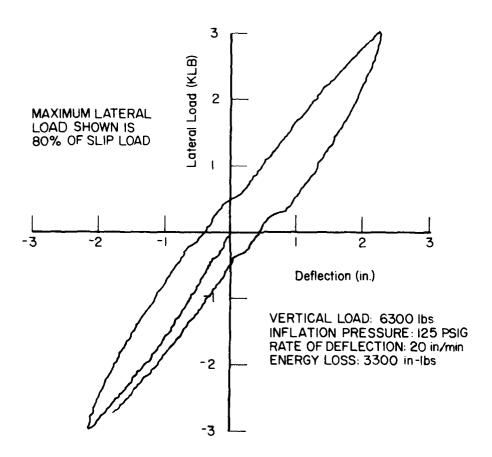


Figure 58. Lateral Load Vs Lateral Deflection, Integral Tire S/N 8098M2

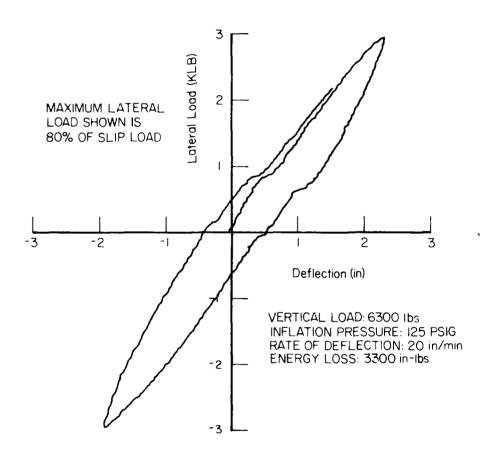


Figure 59. Lateral Load Vs Lateral Deflection, Integral Tire S/N B098N3

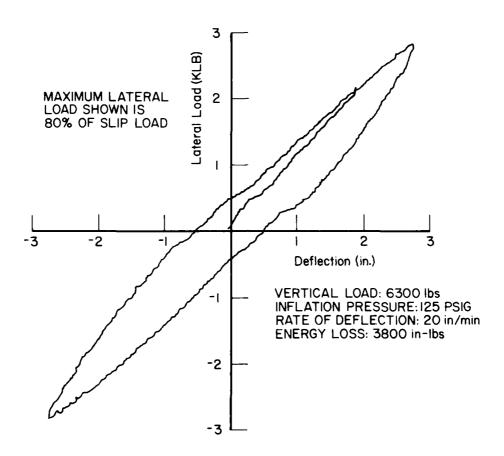
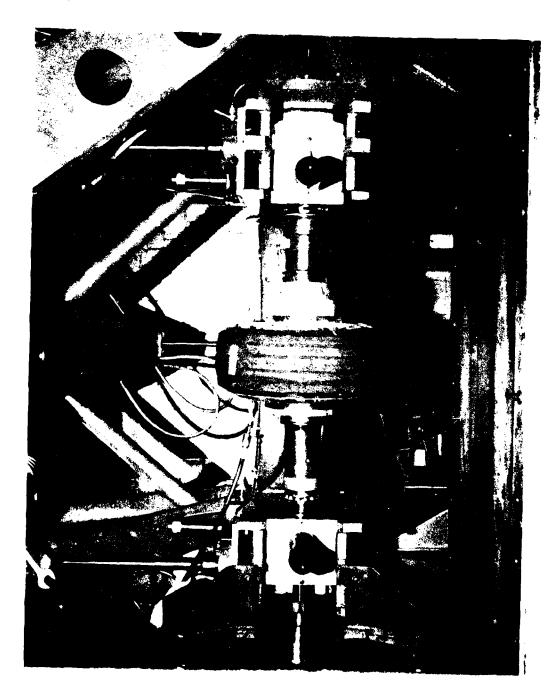


Figure 60. Lateral Load Vs Lateral Deflection, Integral Tire  $\,$  S/N B088K2



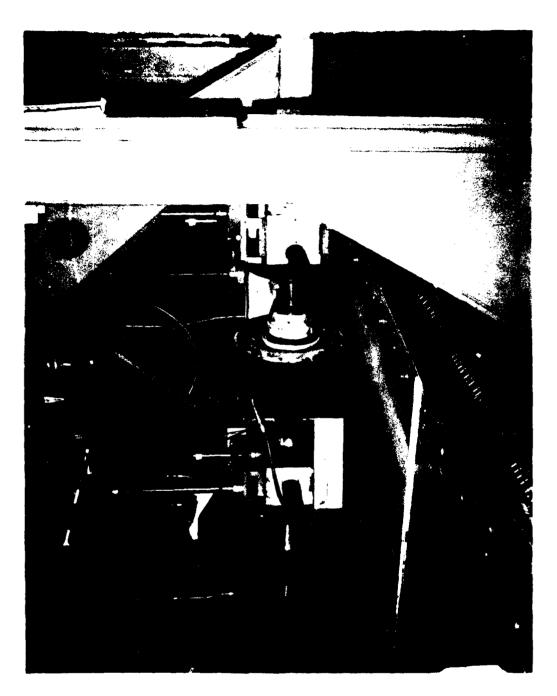
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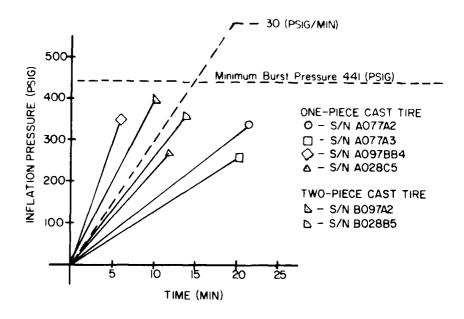




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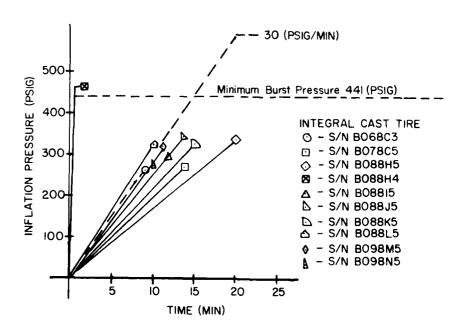
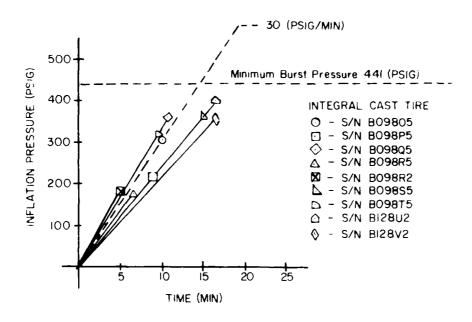


Figure 65. Burst Test Data



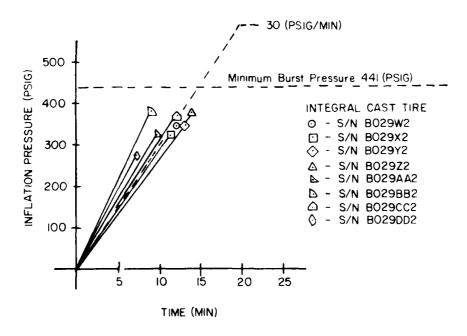


Figure 66. Burst Test Data

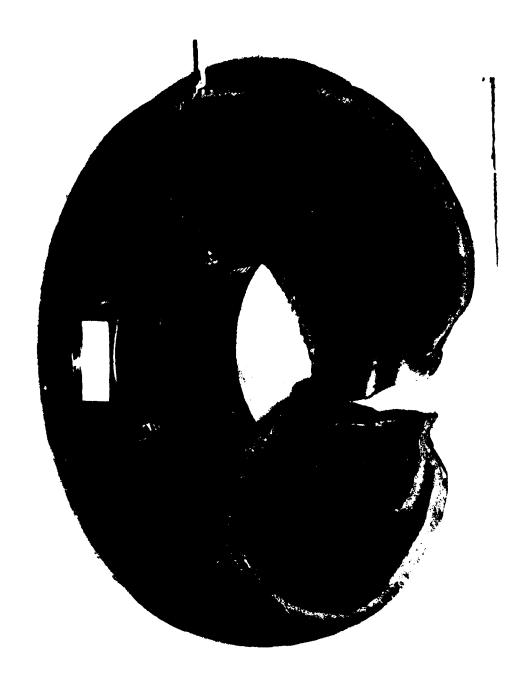


Figure 67. Burst Test One-Piece Cast Tire (S/N A077A3) Brittle Failure-Crown & Sidewall @ 260 PSIG

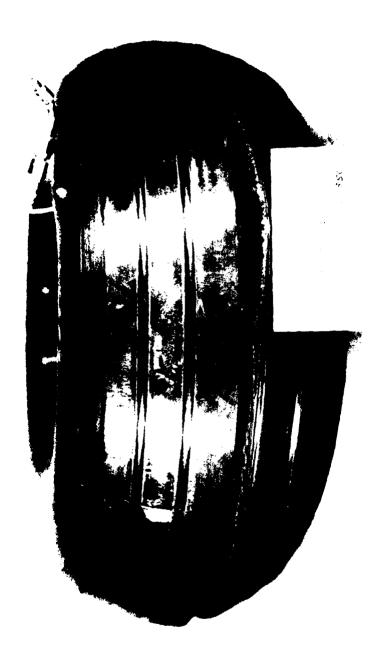


Figure 68. Burst Test One-Piece Cast Tire (S/N A097BB4)
Material Creep Failure-Tread Grooves @ 350 PSIG

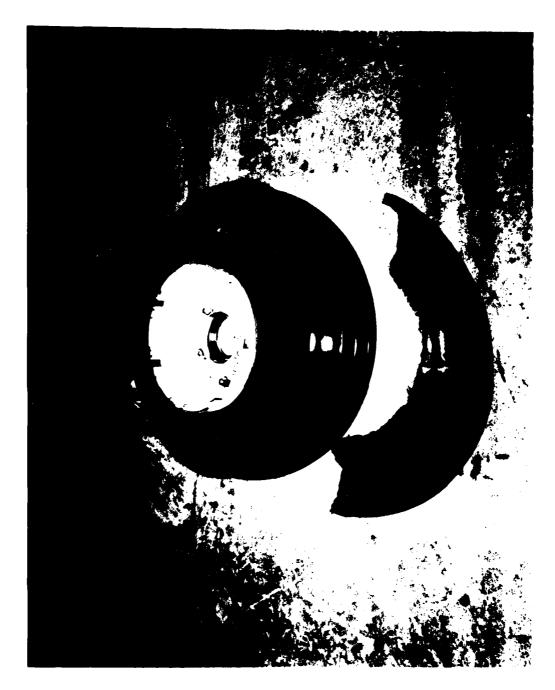


Figure 69. Burst Test One-Piece Cast Tire (S/N A028C5) Brittle Failure-Sidewall @ 270 PSIG

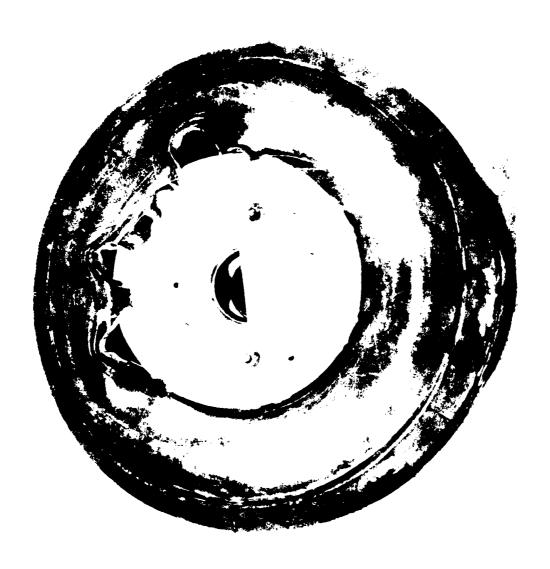


Figure 1. Burst Test Two-stock cost time (SZN posZAC) Material Cresb Falline Sead (1895 P.L.)

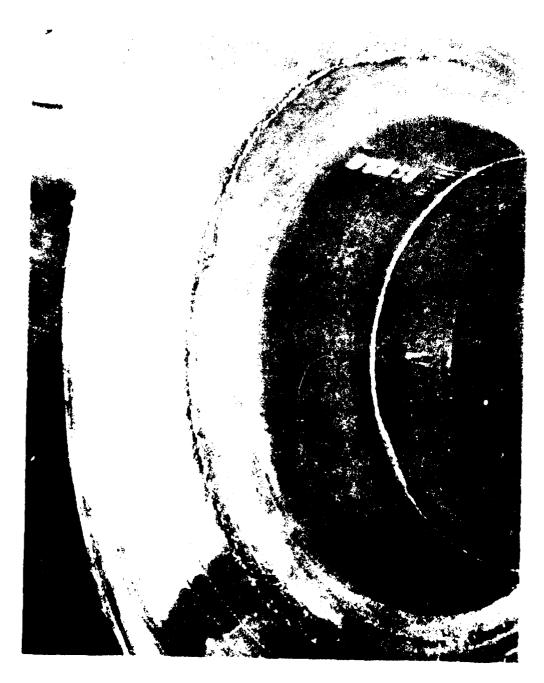


Figure 71. But it lest two-flece cast time (5.% RC, 288): Northe Facture Bead in Asia 5.16



Figure 72. Burst Test Integral Cast Tire (S/N B068C3)
Material Creep Failure-Crown & Belt Edge @ 265 PSIG



Figure 73. Burst Test Integral Cast Tire (S/N B088H4) Material Creep Failure-Bead @ 465 PSIG



Figure 74. Burst Test Integral Cast Tire (S/N B08815) Material Creep Failure-Sidewall @ 295 PSIG

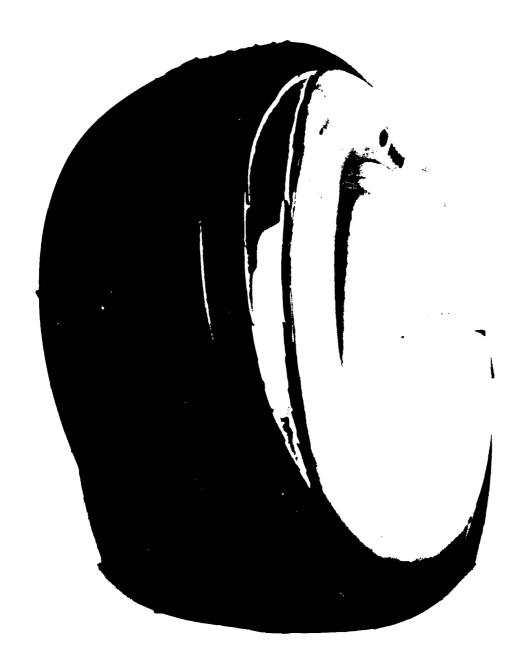


Figure 75. Burst Test Integral Cast Tire (S/N B088J5) Material Creep Failure-Belt Edge @ 345 PSIG

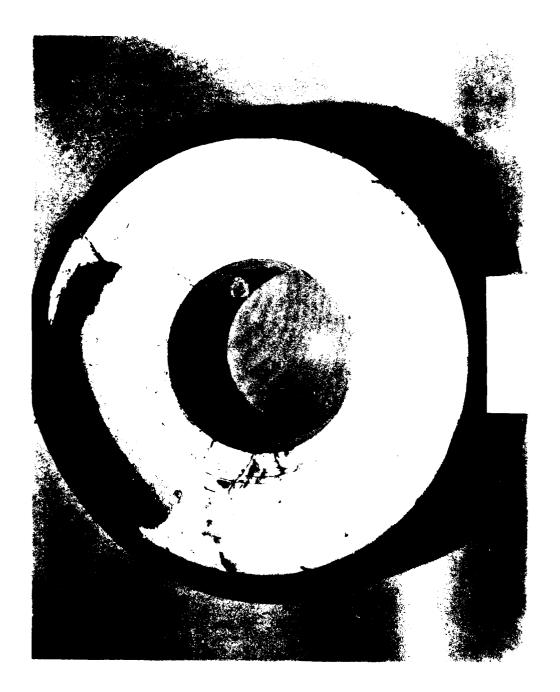


Figure 76. Burst Test Integral Cast Tire (S/N B098L5)
Brittle Failure-Sidewall @ 325 PSIG

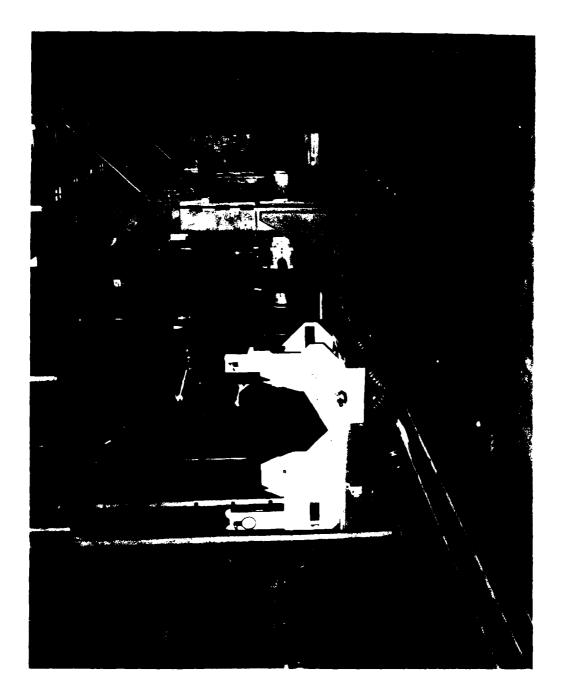


Figure 77. Lateral Force and Aligning Torque Test-Set Up,
Tire Force Machine

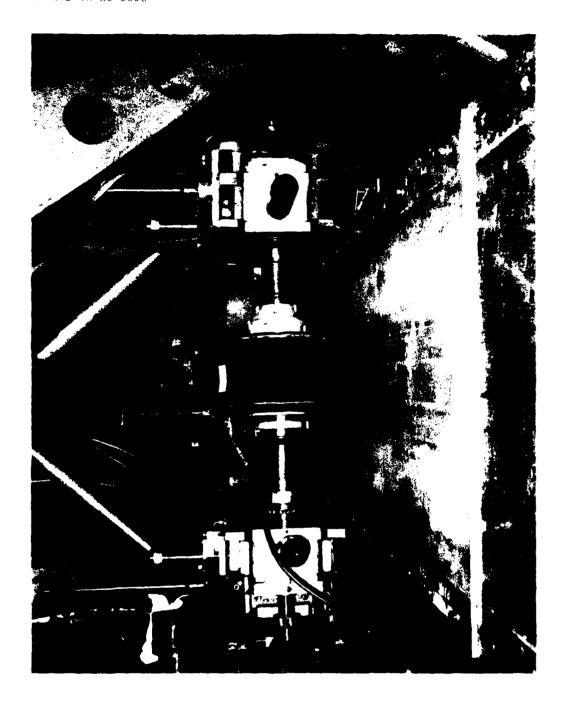


Figure 78. Lateral Force and Aligning Torque Test-Set Up, Tire Force Machine @ 6° Slip Angle

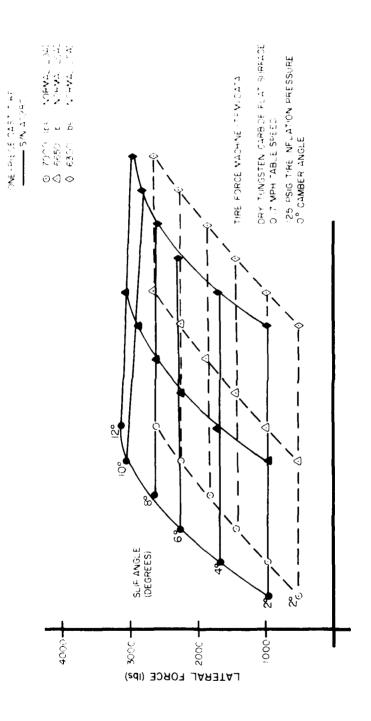


Figure 79. Lateral Force Vs Slip Angle and Vertical Load (Carpet Plots)

BASELINE BIAS TIRE

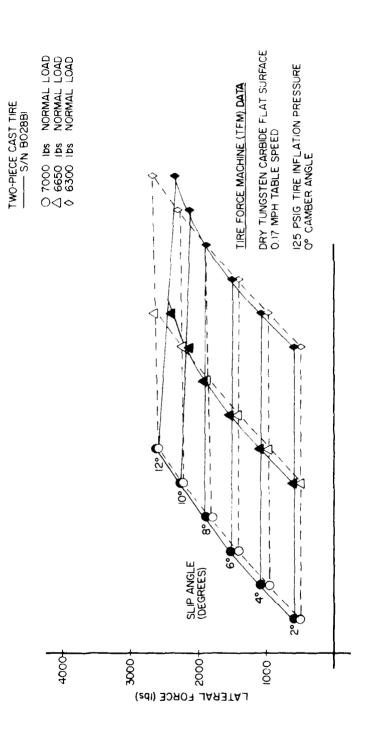


Figure 80. Lateral Force Vs Slip Angle and Vertical Load (Carpet Plots)

BASELINE BIAS TIRE

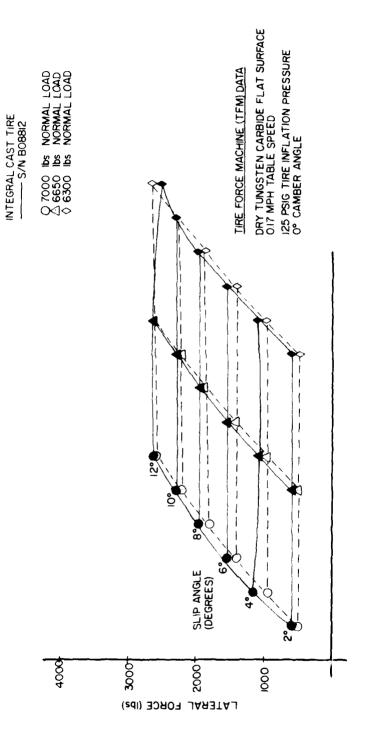


Figure 81. Lateral Force Vs Slip Angle and Vertical Load (Carpet Plots)

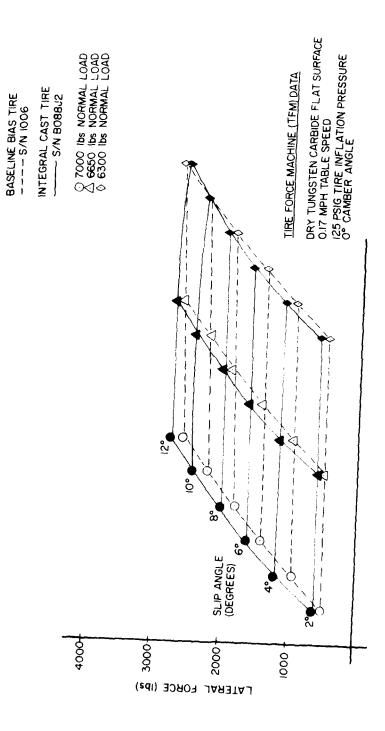
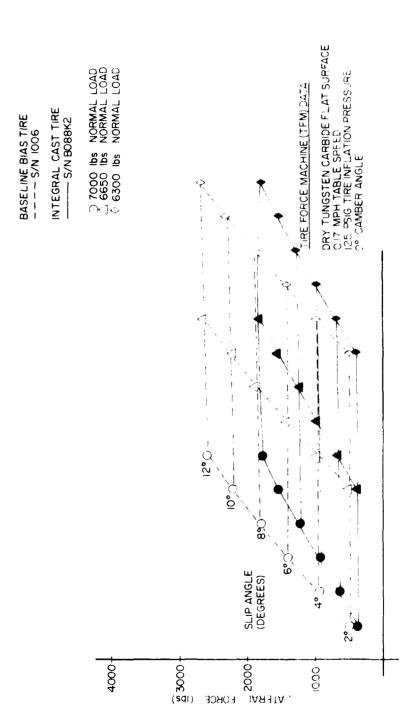


Figure 82. Lateral Force Vs Slip Angle and Vertical Load (Carpet Plots)



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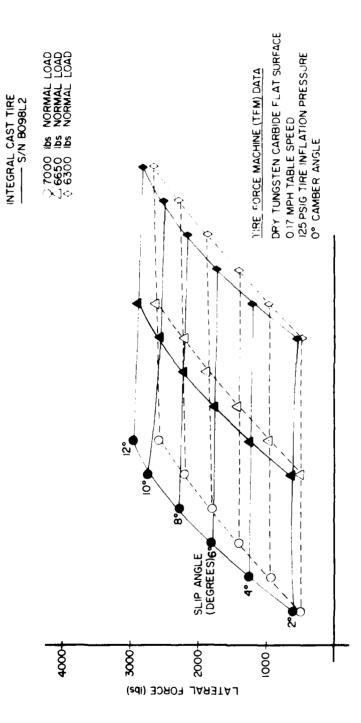


Figure 84. Lateral Force Vs Slip Angle and Vertical Load (Carpet Plots)

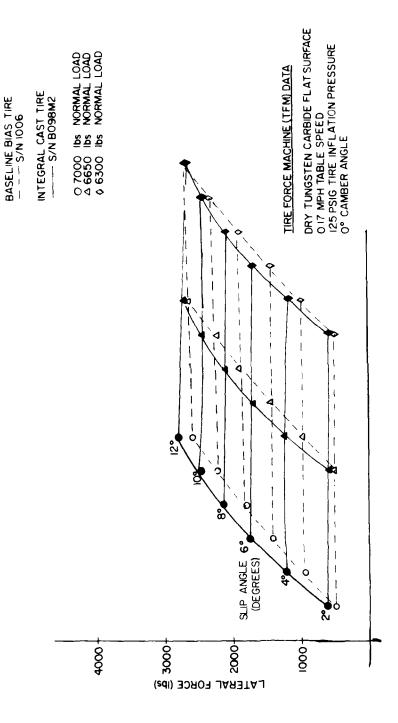


Figure 85. Lateral Force Vs Slip Angle and Vertical Load (Carpet Plots)

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DRY TUNGLTEN CARB'DE FLAT SURFACE 0.17 MPH TABLE SPEED 125 PSIG THE INFLUTON PRESSURE 0° CAMBER ANGLE A 7000 IBS TICRMAL DAD A 6650 DS TICRMAL DAD Q 6300 DS NORMAL DAD TIRE FORCE MACHINE (TEM) DATA 8 SLIP ANGLE (DEGREES: LATERAL FORCE UDS: 0000 000

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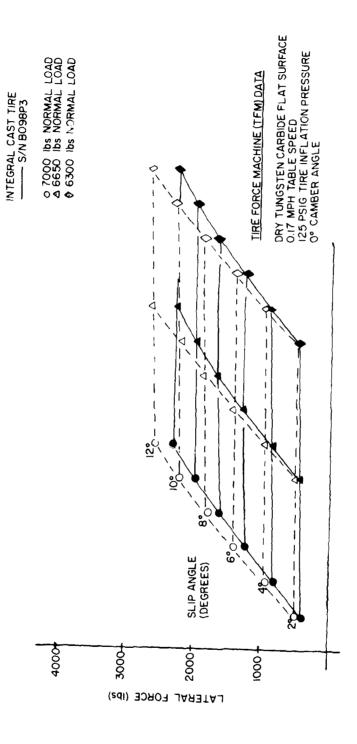


Figure 88. Lateral Force Vs Slip Angle and Vertical Load (Carpet Plots)

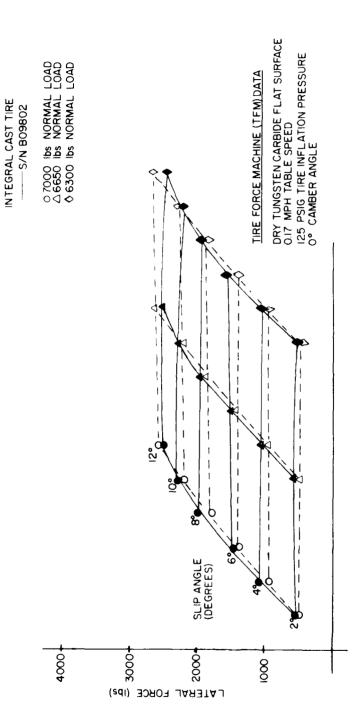
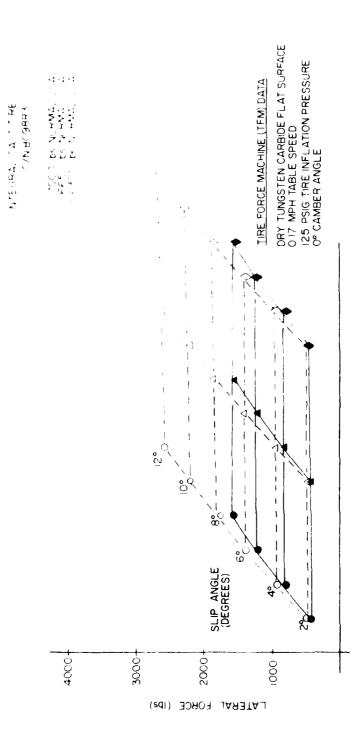


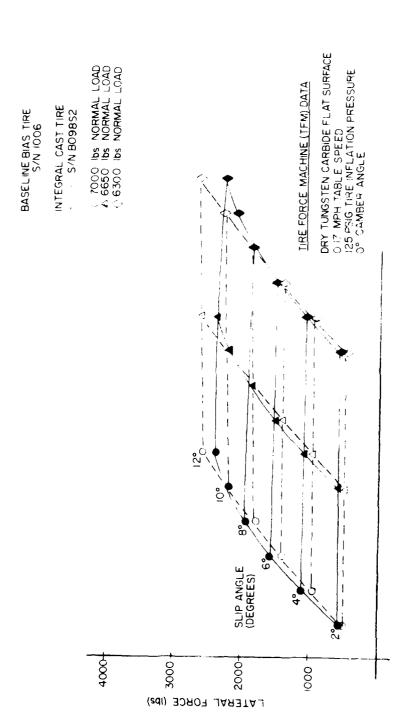
Figure 89. Lateral Force Vs Slip Angle and Vertical Load (Carpet Plots)



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Figure 30. Lateral Force Ms Ship Anthe and tertical Lead (Carbet Phots)





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BASELINE BIAS TIRE ---S/N 1006

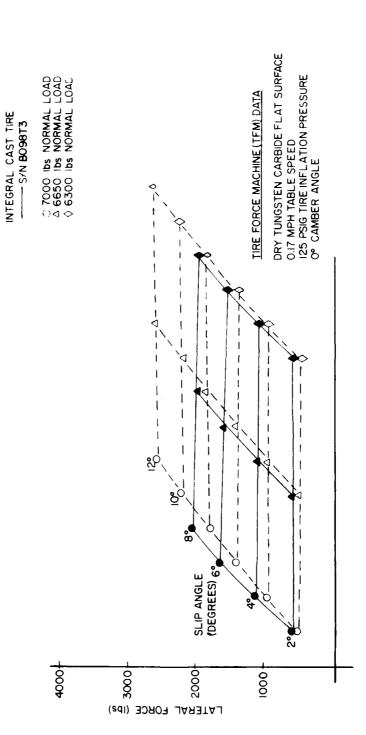


Figure 92. Lateral Force Vs Slip Angle and Vertical Load (Carpet Plots)

BASELINE BIAS TIRE --- S/N 1006

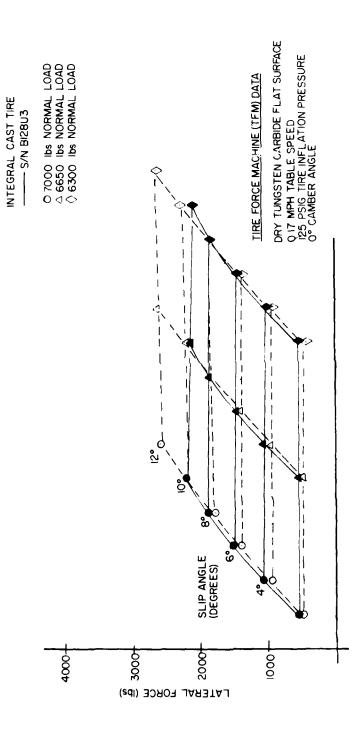
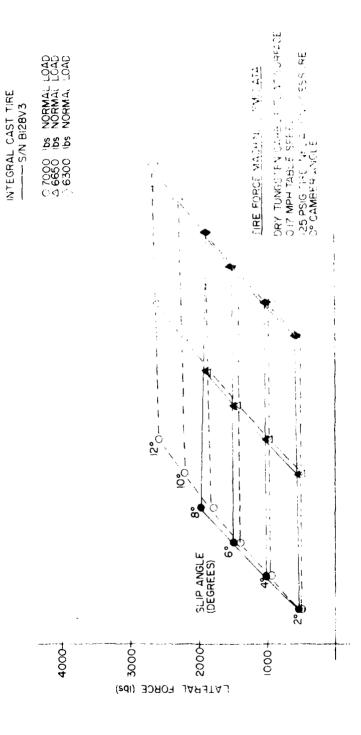


Figure 93. Lateral Force Vs Slip Angle and Vertical Load (Carpet Plots)



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TUNGSTEN CARBIDE FLAT SURFALL C 17 MPH TABLE SPEED 25 PROFTIRE INFLATION PRESSUME OF CAMBER ANGLE TIRE FORCE MACHINE (TEM, DETA 6650 bs 10PMs 100 6300 bs 10PMs 100 6300 bs NOPMs 100 HE MET THE BIAS THE SEPTIME SEPTIMES OF SOME WITTER SLIP ANGLE (DEGREES) LATERAL FORCE (Ibs) 1000 4000

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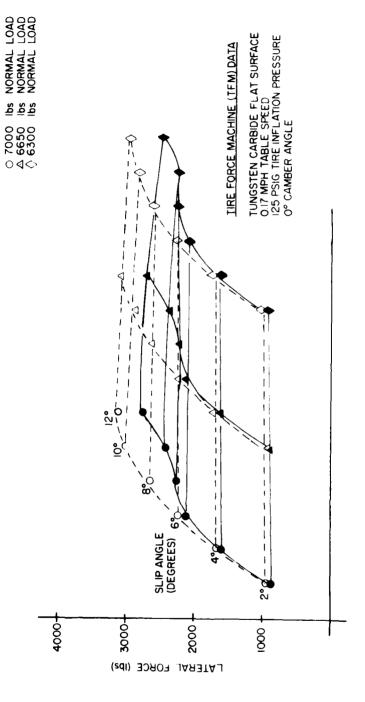


Figure 96. Lateral Force Vs Slip Angle and Vertical Load (Carpet Plots)

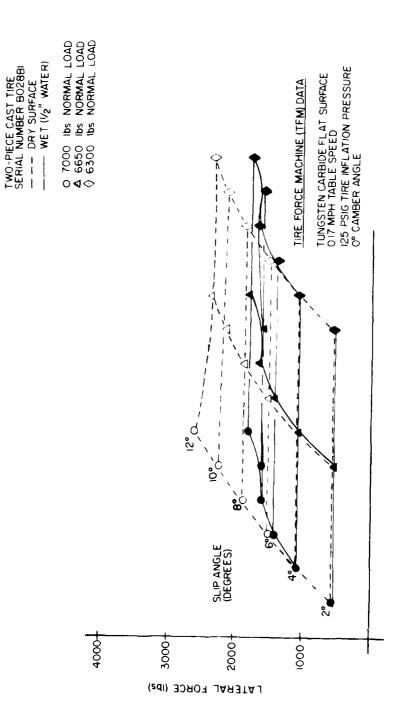
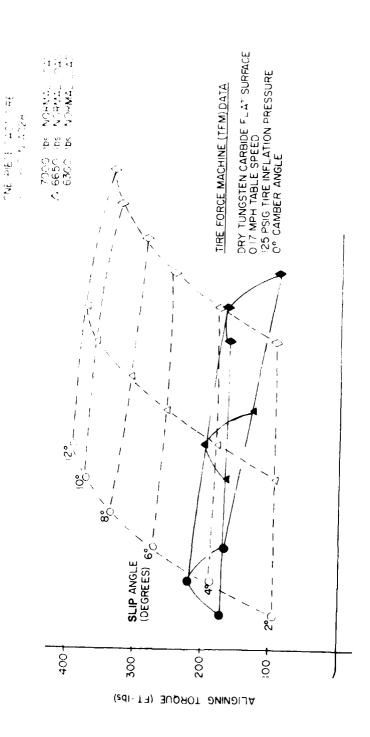


Figure 97. Lateral Force Vs Slip Angle and Vertical Load (Carbet Plots)



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Figure 92. Aligning Torque Vs Siip Angle and ...'t.al Load (Carpet Plots)

BASELINE BIAS TIRE --- S/N 1006

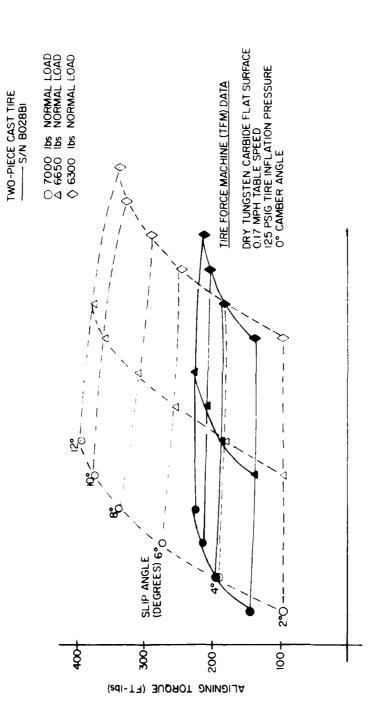


Figure 99. Aligning Torque is slip with and perforal task (Caroet Plets)

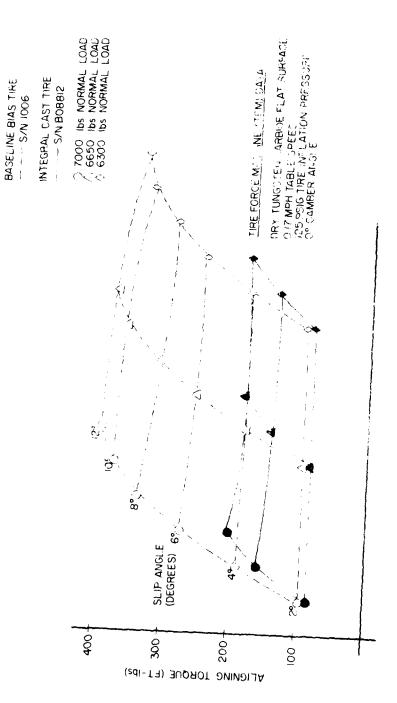
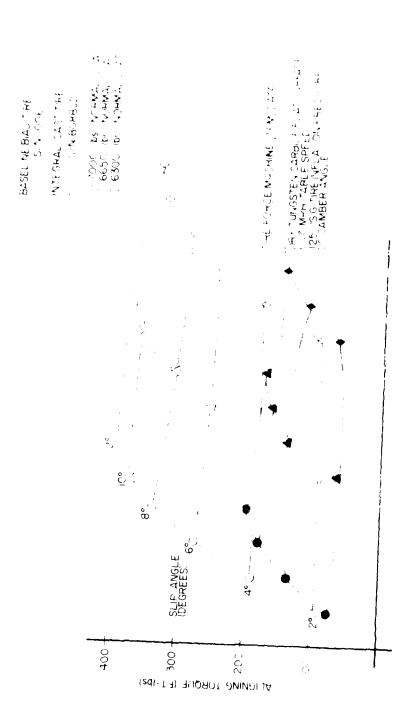
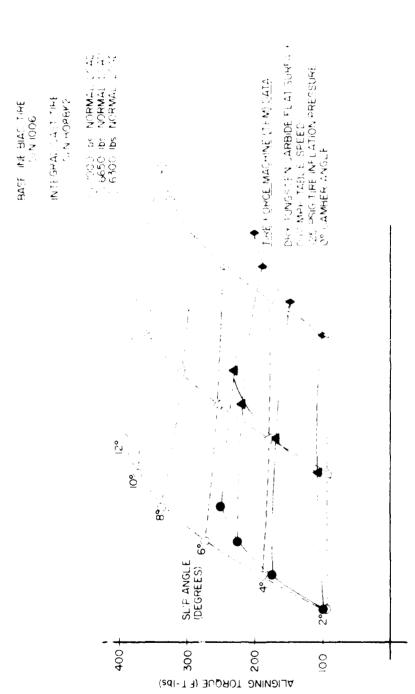


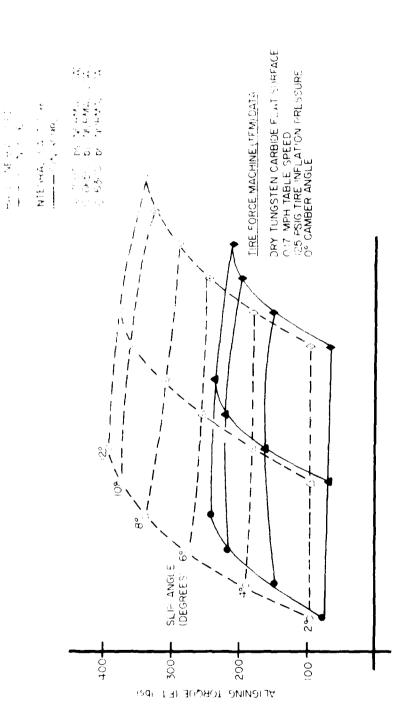
Figure 100, Aliching Thrup in Sib Angle and Verson of Angles



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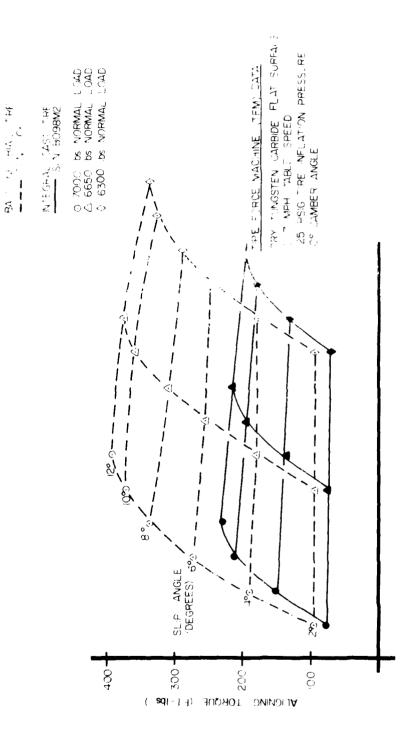
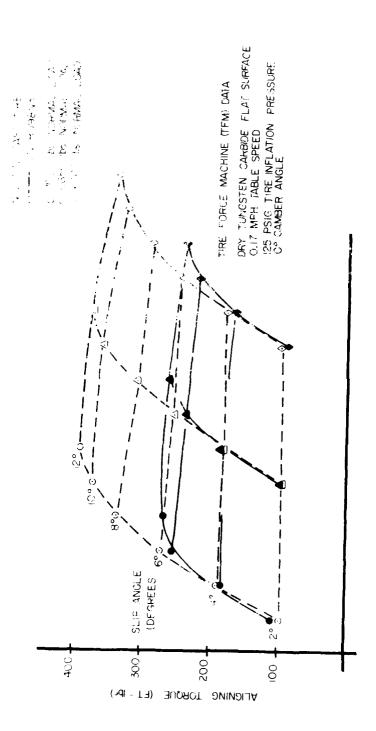


Figure Mid. Aligning Torque As STD Angle of Lieffstal Lead (Carnet Plots)



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Figure 105. Aligning Torque Vs Slip Ingle and Variscal Lous (Carpet Plots)

BASELINE BIAS TIRE ---S/N ICC6 INTEGRAL CAST TIRE S/N B09802

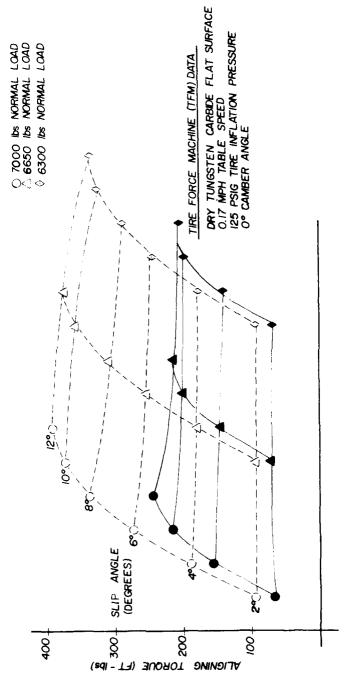
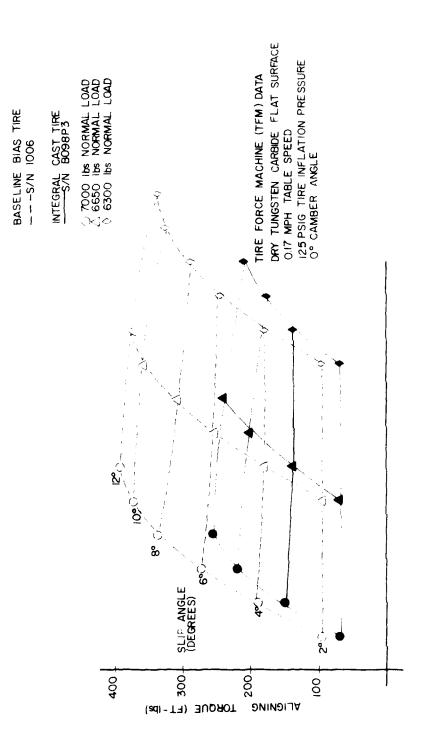
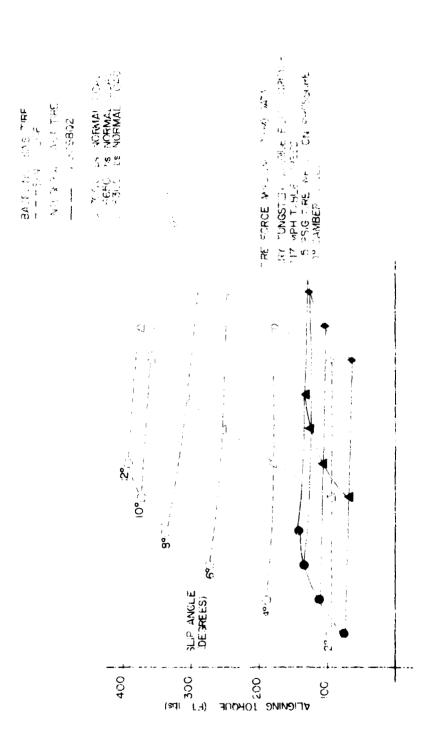


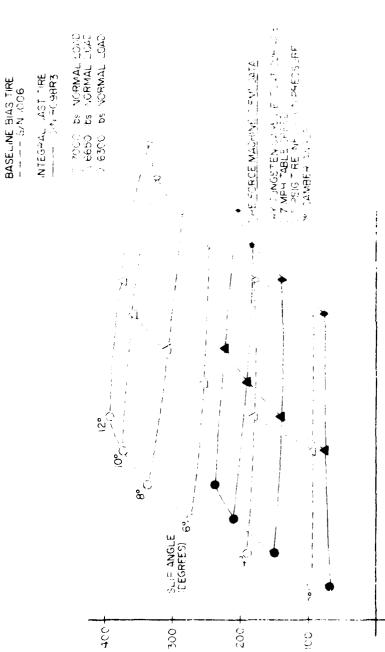
Figure 106. Aligning Torque Vs Slip Angle and Vertical Load (Carpet Plots)



Frunce 107. Aligning Torque Vs 191p Ample and Sertical Isad (Campet Plots)



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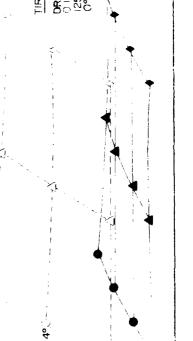
ALIGNING TORQUE (FT-Ibs)

INTEGRAL CAST TIRE

7000 lbs NORMAL LOAD 6650 lbs NORMAL LOAD 6300 lbs NORMAL LOAD

TIRE FORCE MACHINE (THM) DAIA

DRY TUNGSTEN CHARLE LEAT SUPPLIE 0 I7 MPH TABLE SPEC. 125 PSIG TIRE INFLATON OF ESSURE 0° CAMBER ANGLE



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Figure 11 . Aligning Torque 7/ ... Toring and Lower Courset Plots)

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SLIF ANGLE (DEGREES)

(FT - 1bs)

400

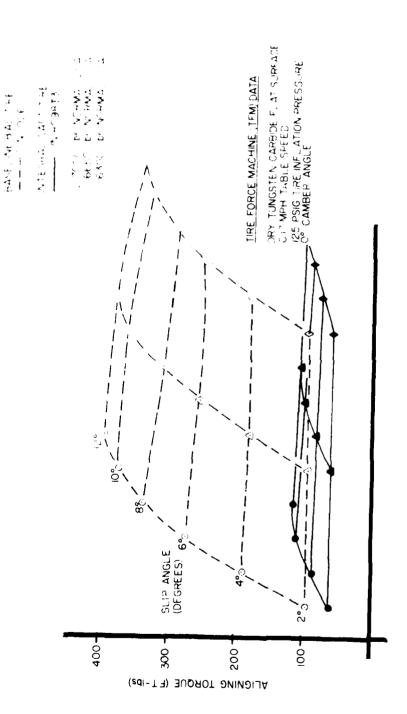


Figure 117 - Alipping Torque V. Sily Aribo got vertors of a de

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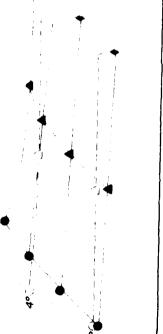
SLP ANGLE 6" C

ALIGNING TOROUE (FT - INS)

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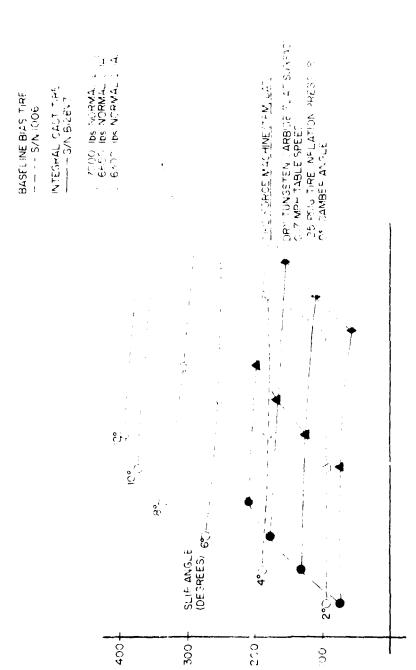
400

DRY TUNGSTEN CARBIDE FLAT SUPFA :: 0.17 MPH TABLE SPEED 12F PSIS TIRE INFLATION PRESSUR:: 0° CAMBER ANGLE TIRE FORCE MACHINE (TEM) DATE



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Figure (II) Alignops formule . The Anche as seen of a confidence (Campet Biots)



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Figure (1). Aliming Torque W. Silto Andle and Lewings (1) de Campel Plots.

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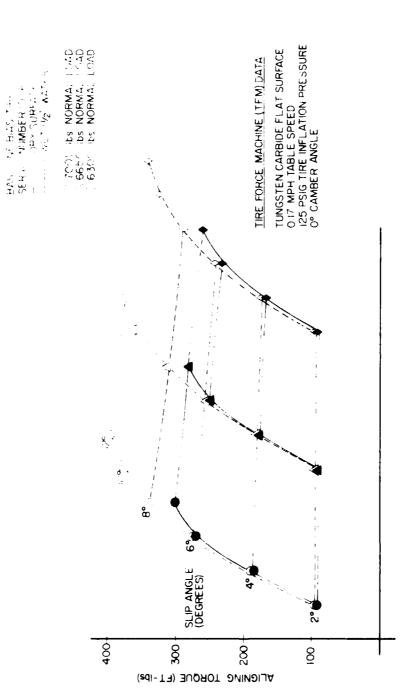


Figure 114, Aligning Torque Vs Stit Angle and with al-load (Carpet Plots)

ONE - PIECE CAST TIRE SERIAL NUMBER AO2BCI --- DRY SURFACE --- WET (1/2 "WATER) 7000 Ibs NORMAL LOAD 6650 Ibs NORMAL LOAD 6300 Ibs NORMAL LOAD

TIRE FORCE MACHINE (TFM) DATA

TUNGSTEN CARBIDE FLAT SURFACE 0.17 MPH TABLE SPEED 125 PSIG TIRE INFLATION PRESSURE 0° CAMBER ANGLE

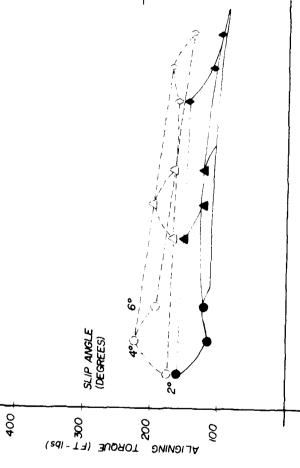


Figure 115. Aligning Torque Vs Slip Angle and Tertical Load (Carpet Plots)

TWO-PIECE CAST TIRE SERIAL NUMBER BO28BI ---- DRY SURFACE ---- WET (1/2" WATER)

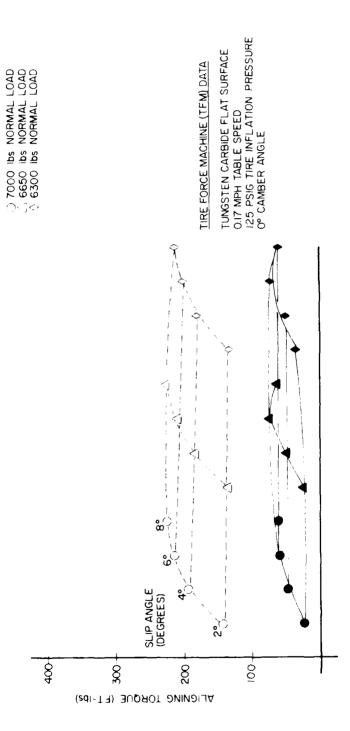


Figure 116. Aligning Torque Vs Slip Angle and Vertical Load (Carpet Plots)

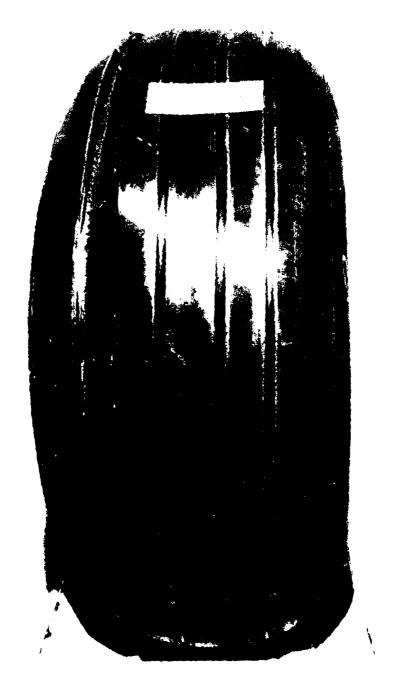


Figure 117. Dynamic Taxi Test One-Piece cast Tire (\* N & 900 bo) Material Creep Failure-Tread Groove



figure IIC. Synamic Yazi Yest One Piece Cast Tire (S/N A077A4) Material Creep Failure Tread Groove

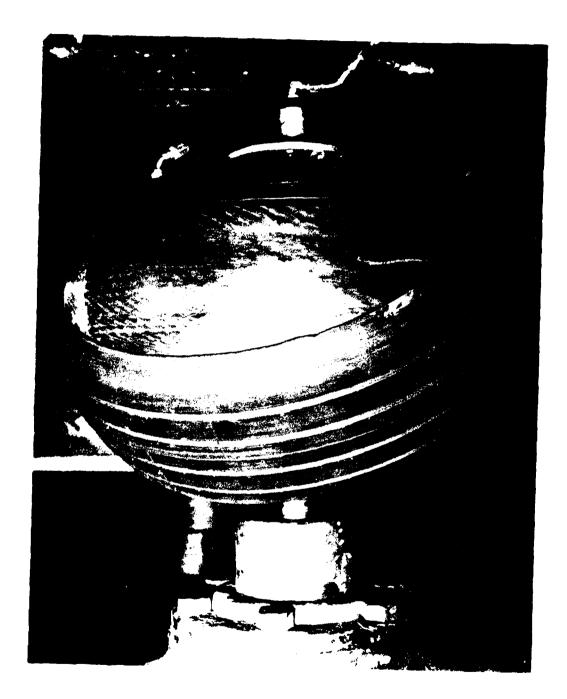


Figure 119. (Mainter Cont. Two-Piece Cast Tire (S/N 509782). Tread December Filting

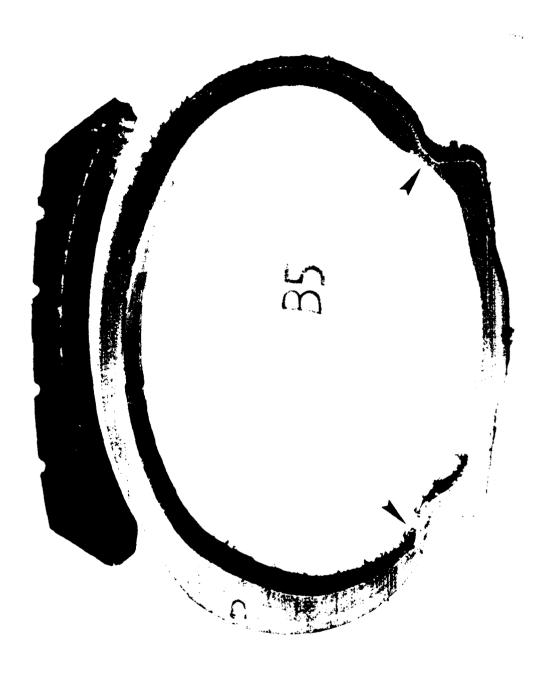


Figure 120. Dynamic Taxi Test Two-Piece Cast Tire (S/N B028B5) Material Creep Failure-Bead Radius Area



Figure 121. Dynamic Taxi Test Two-Piece Cast Tire (S/N B028B1) Flex Crack Failure-Bead Radius Area



Figure 122. Two-Piece Cast Tire Section, Improper Material Thermal Cure

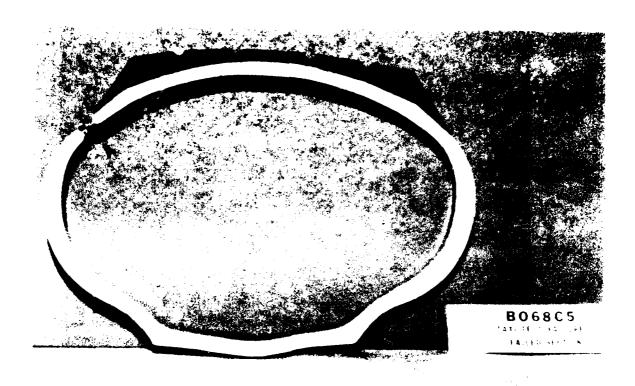


Figure 123. Dynamic Taxi Test Integral Cast Tire (S/N 8068C5)
Material Creep Failure-Shoulder @ Belt Edge

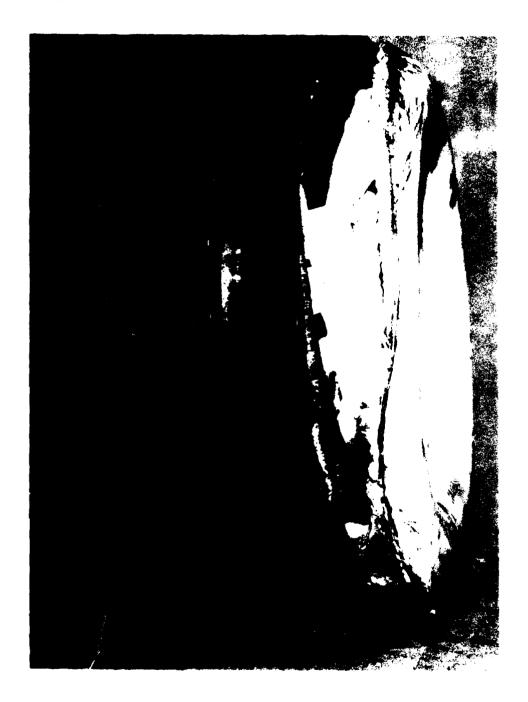


Figure 124. Dynamic Taxi Test Integral Cast Tire (S/N B078C2) Material Creep Failure-Shoulder @ Belt Edge

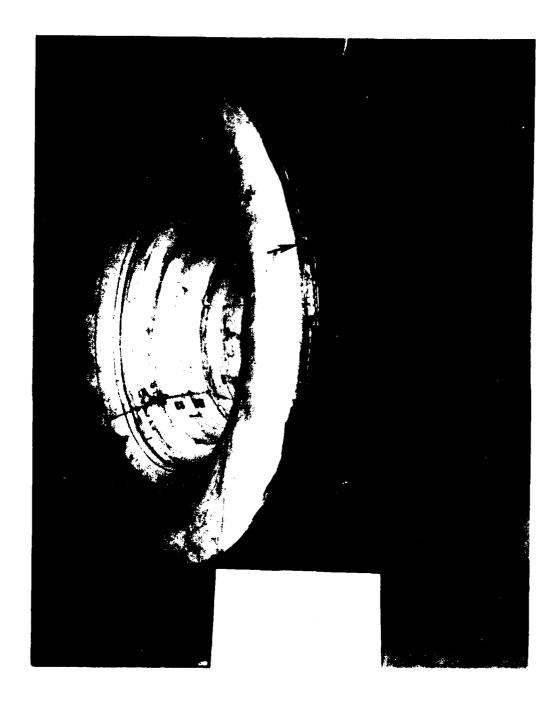


Figure 125. Dynamic Taxi Test Integral Cast Tire (S/N B088J3) Material Creep Failure-Shoulder @ Belt Edge

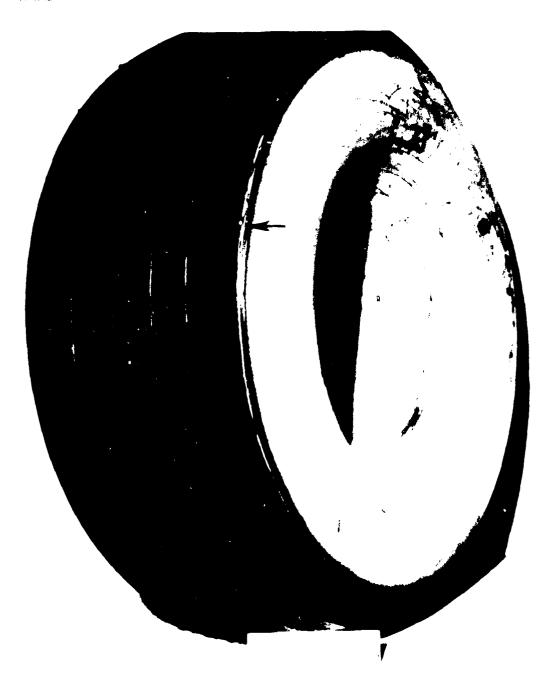


Figure 126. Dynamic Taxi Test Integral Cast Tire (S/N B088K1) Material Creep Failure-Shoulder @ Belt Edge

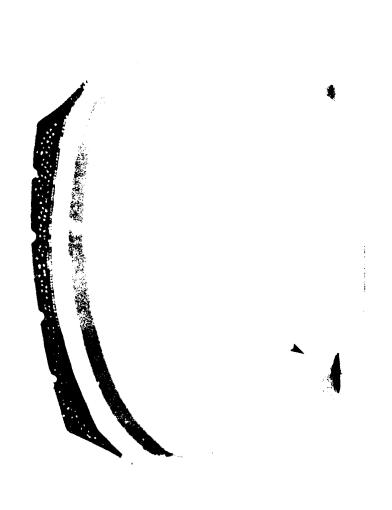


Figure 127. Dynamic Taxi Test Integral Cast Tire (S7N B07RC!) Material Creep Failure-Bead Radius Area



Figure 128. Dynamic Taxi Test Integral Cast Tire (S/N B088I1) Material Creep Failure-Bead Radius Area

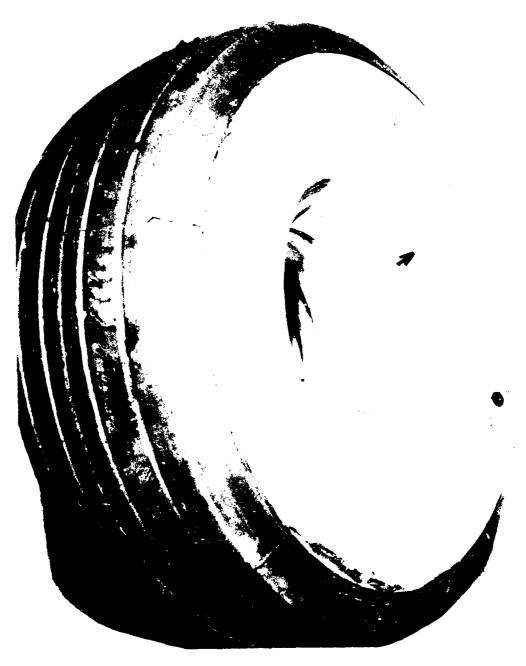
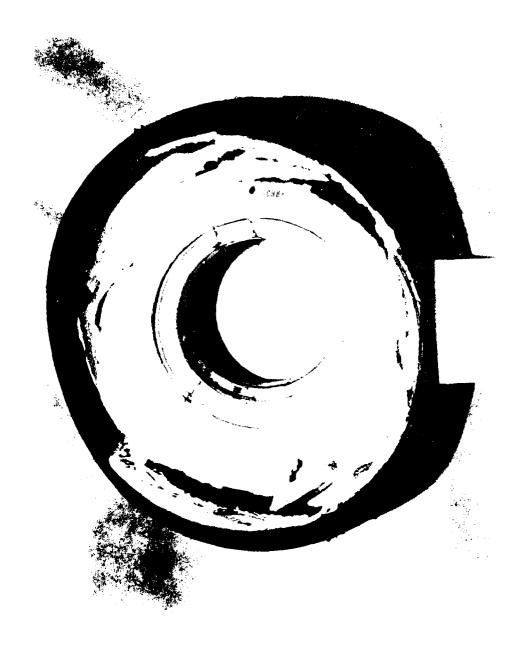


Figure 129. Dynamic Taxi Test Integral Cast Tire (S/N B098R1) Material Creep Failure-Bead Radius Area



Figure 130. Dynamic Taxi Test Integral Cast Tire (S/N B098L1) brittle Failure-Sidewall (Glass Reinforced Tire)



Tiron [3]. Evnamme Last Test Integral Cast Tiro (1). Smittle Failume-Shoulder & Sidewall (1). Reinforced (196)

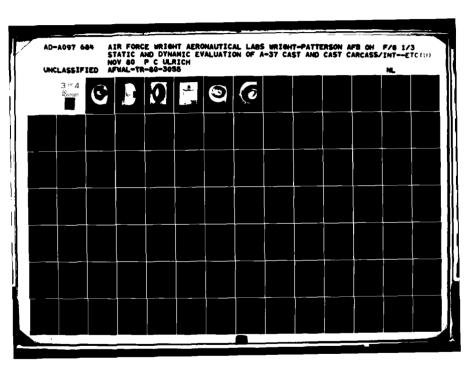




Figure 132. Dynamic Taxi Test Integral Cast Tire (S/N B098S1) Brittle Failure-Shoulder & Sidewall (Glass Reinforced Tire)



Figure 133. Dynamic Taxi Test Integral Cast Tire (S/N Bl28U1) Brittle Failure-Shoulder & Belt Edge



Figure 134. Dynamic Taxi Test Integral Cast Tire (S/N B029Y1) Brittle Failure-Sidewall & Bead Radius Area

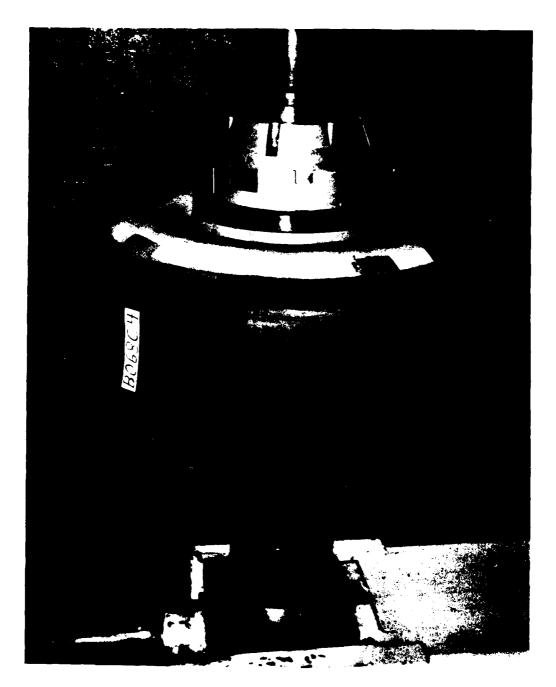


Figure 135. Dynamic Take Off Test Integral Cast Tire (S/N B068C4) Successfully Completed 20 (0-150 MPH) Take Offs



Figure 136. Dynamic Take Off Test Integral Cast Tire (S/N B068C4) Successfully Completed 100 (0-150 MPH)
Take Offs.Failed During First Taxi Test-Creep Failure
@ Belt Edge



Figure 137. Dynamic Taxi Test Integral Cast Tire (S/N B068C5) Tire to Wheel Slippage (3.5 Inches)

## APPENDIX C TIRE CONTACT PRINTS (FOOTPRINTS)

TEST TIRE FOOTPRINT:			
MEW . X USED	RETREAD BY	N/A	
5. 0. NO	CODE NO.	3-N FLPL .X	FLWGL
SKID DEPTH	IN.	MAX. FOOTPRINT LGTH	l2:33 IN.
RATED INFLATION	PSI	MAX. FOOTPRINT WOTH	ا ۱۱۸۰ - ۱۱۸۰ - ۱۱۸۰ از در ایرا
LUU % RATED LOAD 5030.	LBS.	NEI CONTACT AREA	"1"1.E". SO. IN.
34.44 DEFLECTION		GRUSS CONTACT AREA	.33:30. SQ. 1N.
OPERATOR	DATE 3/6	5/78 SERIAL NR	()

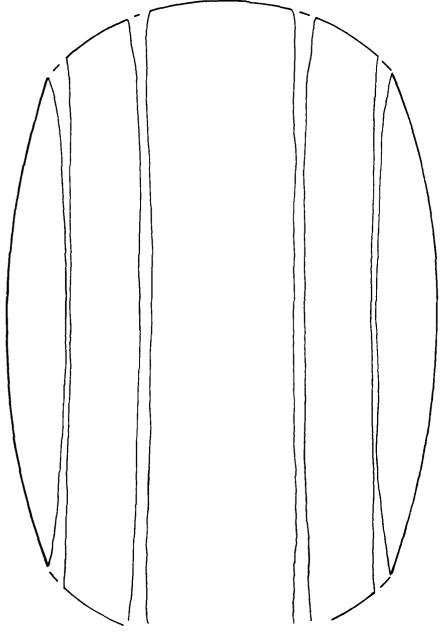


Figure C-1. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE .	7.00-8/16	MFR Goodyear
NEW X USED			
S 0. NO. 77-21	CODE NO.	2-N FLPL	X. FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT LGT	H7.61 IN.
RATED INFLATION125	PSI	MAX. FOOTPRINT WDT	H. 5.37 IN.
60 % RATED IDAN 3990	IRS	NET CONTACT AREA .	
21.24 DEFLECTION		GROSS CONTACT AREA	.34.55 SQ. IN.
OPERATOR	DATE 3/6	<sup>78</sup> SERIAL NRº	920

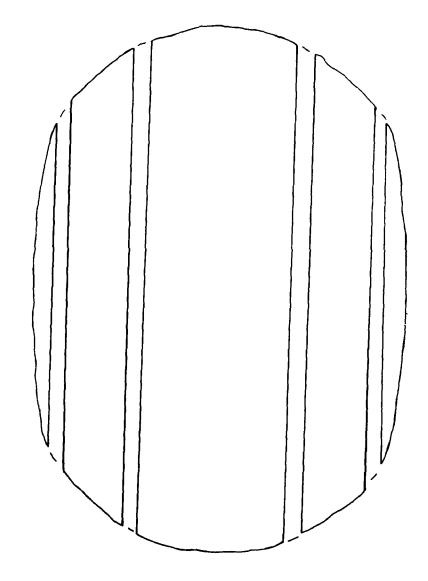


Figure C-2. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE .	7.00-8	MFR Zedron
NEW .X USED			
S. O. NO77-21			
SKID DEPTH	IN.	MAX. FOOTPRINT	LGTH <sup>7</sup> . <sup>17</sup> IN.
RATED INFLATION125	PSI	MAX. FOOTPRINT	WDTH
100 % RATED LOAD 665	0 LBS.	NET CONTACT ARE	A <sup>22</sup> :90 SQ. IN.
26.80 DEFLECTION		GROSS CONTACT A	REA 28:93 SQ. IN.
26.80. DEFLECTION OPERATOR	DATE <sup>3/1</sup>	1/78 SERTAL NR.	A077A1

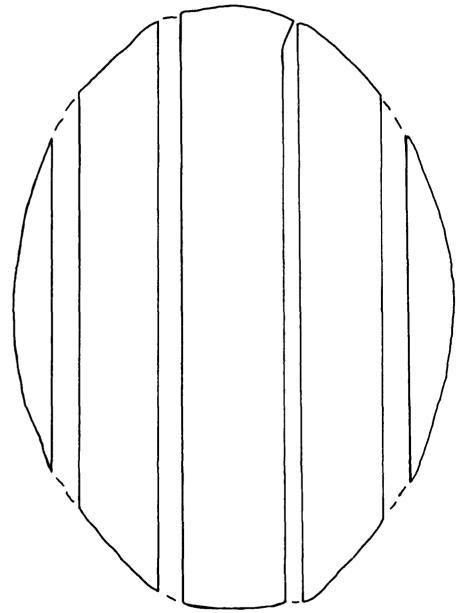


Figure C-3. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:			
NEWX USED			
S. O. NO	CODE NO.	! FLF	YLÄ. FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT	Г LGTH <u>5.62</u> IN.
RATED INFLATION125	PSI	MAX. FOOTPRINT	r WDTH. <u> 3</u> . 6 <sup>3</sup> . IN .
. 60. % RATED LOAD3990	LBS.	NET CONTACT AF	REA14:45 SQ. IN.
1 <b>9</b> :24. DEFLECTION		GROSS CONTACT	AREA $16.93$ SQ. IN.
OPERATOR	DATE 3/1	1778 SERIAL NR.	A077A'

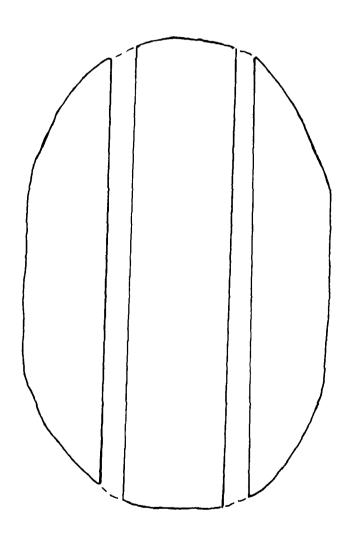


Figure C-4. Tire Contact Prints (Footprints)

Hall Har Emsternal.	T188 SEEL .	.700:5	Mik 464000
them are the first of the contract of	. PrileralieY		
St. 9. W 7/: M	Com A.A.		J.X. Hwat
5840 Section	Iħ.	MAX. FOSTPRINT L	61H. 3539 M.
FALLD INFLATIONE	P	- MAX. FOOTPHIRE W	9fa. 25/99 Pa
LAST RATED FOATPP	St 188.	THE CONTACT AREA	39-99-59-15
ASSEZ BEFLECTION		GRUSS CONTACT AR	EA 27-25. SO. 15.
00100000	bair 🖓 🗥	<sup>178</sup> SERIAL NR	.A097881

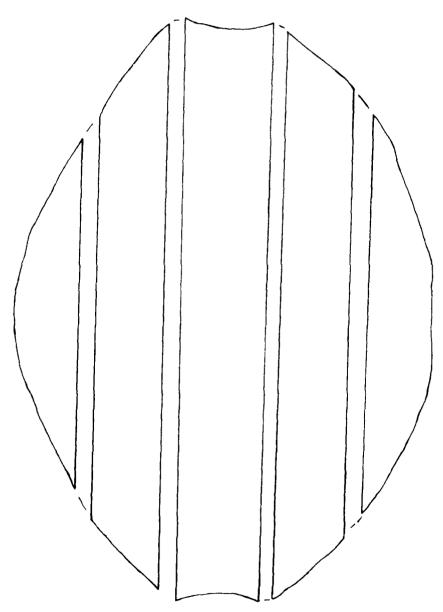


Figure C-5. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:				
NEW X USED				
S. O. NO77-21	CODE NO.		FLPL . X . FLI	WHL
SKID DEPTH	IN.	MAX. FOOTPI	RINT LGTH. 6.77	IN.
RATED INFLATION	isa PSI	MAX. FOOTPI	RINT WDTH4.	.35 IN.
60. % RATED LOAD3	990 LBS.	NET CONTACT	T AREA	. SQ. IN.
22:34. DEFLECTION		GROSS CONTA	ACT AREA 23:1.	. SQ. IN.
OPERATOR	DATE 2.24	78 SERIAL	NR. A097BB1	

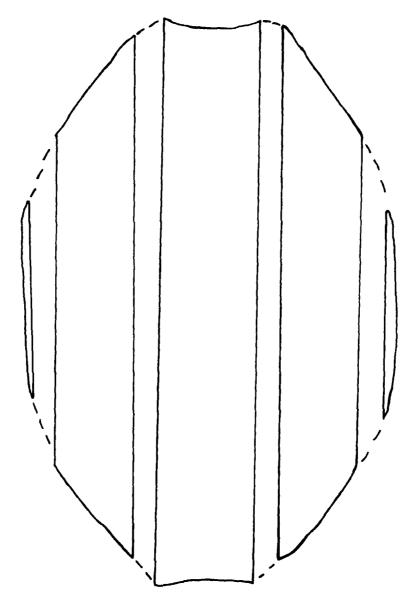


Figure C-6. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:			
NEWX USED	RETREAD BY	N/A	
S. O. NO77-21	CODE NO.	19 FLF	^L FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT	r LGTH <sup>8</sup> :.02 IN.
RATED INFLATION125			
.100 % RATED LOAD .6650	LBS.	NET CONTACT AF	REA . 28.76 SQ. IN.
30.57. DEFLECTION OPERATOR		GROSS CONTACT	AREA 35.72, SQ. IN.
OPERATOR	DATE 2/2	8/78 SERIAL NR.	A028C4`

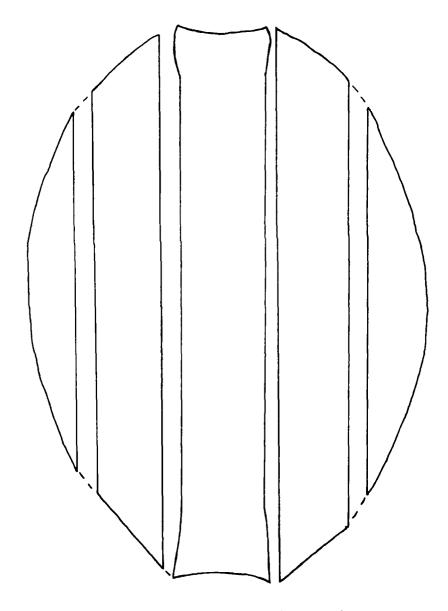


Figure C-7. Tire Contact Prints (Footprints)

HEST TIRE FOOTPRINT:	TIRE SIZE .	7.00-8	MFR Zedron
NELL . N USED	RETREAD BY	N/A	
S. O. NO77-21 SKID DEPTH	CODE NO.	<sup>19</sup> FLPL	X FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT	LGTH6.38 IN.
RATED INFLATION 125	PSI	MAX. FOOTPRINT	WDTH. 4.36 IN.
PARL & RATED LOAD .3990	LBS.	NET CONTACT ARE	A17.13 SQ. IN.
JT.SZ DEFLECTION		GROSS CONTACT A	AREA .21.91 SQ. IN.
OPERATOR	DATE 2/2	28/78 SERIAL NR.	A028C4

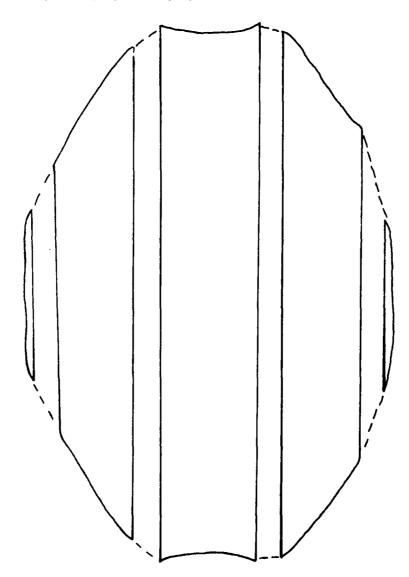


Figure C-8. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE .	7,00-8	MFR Zedron
New York USED	RETREAD BY	N/A	
5. 0. No 7.7-21	CODE NO.	3 FLPL .X	. FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT LGTH.	7.63 IN.
RATED INFLATION125			
100 % RATED LOAD 6650 25:24 DEFLECTION	LBS.	NET CONTACT AREA	6.85. SO. IN.
22:24 DEFLECTION		GROSS CONTACT AREA 4	3.36. SQ. IN.
OPERATOR	DATE 2/28	<sup>3/78</sup> SERIAL NR	B097A3

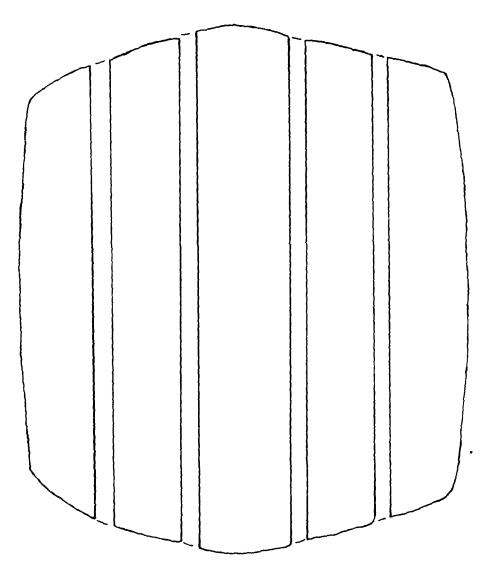


Figure C-9. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:			
NEW .X USED	RETREAD BY		
S. O. NO77-21	CODE NO.	.3 FLPL	X FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT	LGTH
RATED INFLATION 125	PSI	MAX. FOOTPRINT	WDTH 6:24 IN.
60 % RATED LOAD 3990	LBS.	NET CONTACT ARE	A $^{25}.5$ SQ. IN.
16.22. DEFLECTION	- 1	_GROSS CONTACT A	REA 30:44. SQ. IN.
60. % RATED LOAD 3990 16.22. DEFLECTION OPERATOR	DATE 2/3	<sup>28/78</sup> SERIAL NR.	B097A3

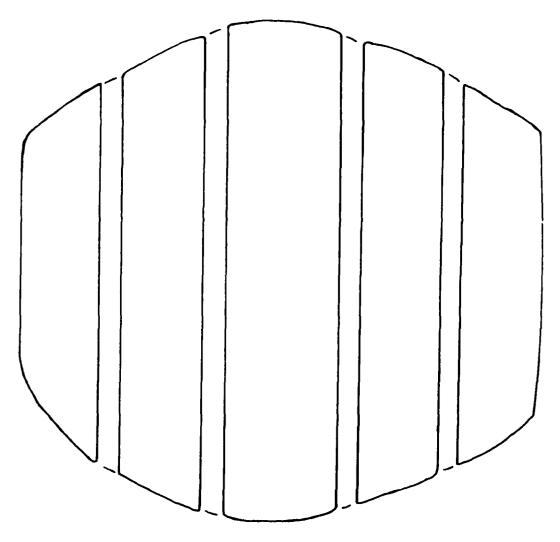


Figure C-10. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE 7.00-8 PETREAD BY NZA	MER "Çedrop.
S. O. NO	CODE NO. BO2884 FLPL XIN. MAX. FOOTPRINT LGTH.	FLWHL
	PSI MAX. FOOTPRINT WDTH.	6.37 IN.
22.95. DEFLECTION		0. <b>7</b> 6 SQ. IN.

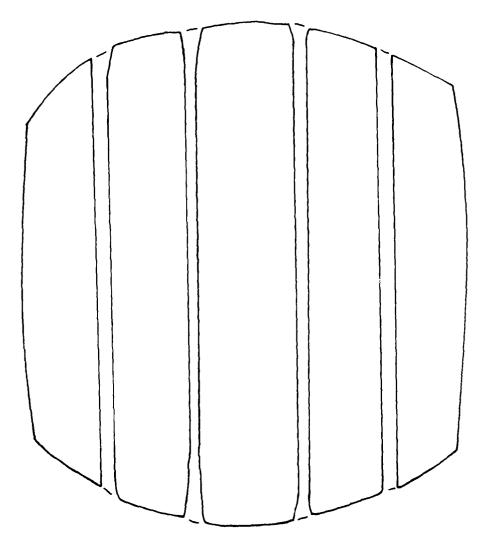


Figure C-11. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT: NEW .X USED	TIRE SIZE .	7.00-8	MFR Zedron
NEW . X USED	RETREAD BY	<u>Ν</u> /Λ	
S. O. No77-21			
SKID DEPTH	IN.	MAX. FOOTPRINT L	GTH 5 : 78 IN.
RATED INFLATION $125$	PSI	MAX. FOOTPRINT W	IDTH6:25 IN.
.60 S RATED LOAD 3990 .14.64 DEFLECTION OPERATOR	LBS.	NET CONTACT AREA	$1 \dots \frac{23}{33} : \frac{82}{3} = SQ$ . IN.
.14.54 DEFLECTION	3	, GROSS CONTACT AF	REA 28:/6 SQ. IN.
OPERATOR	. DATE	11/18 SERIAL NR	B028B4

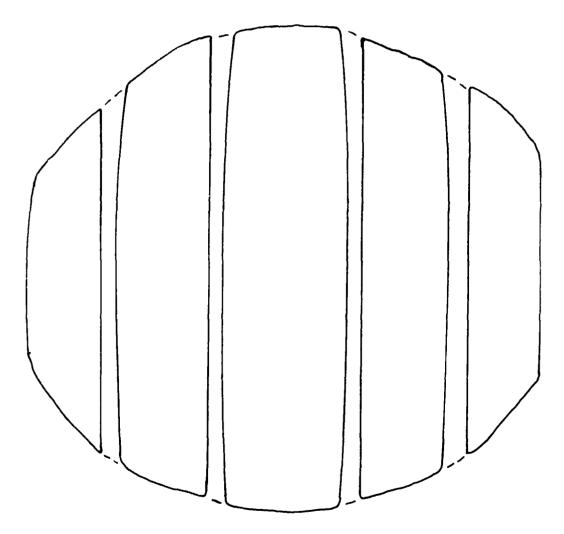


Figure C-12. Tire Contact Prints (Footprints)

TEST TIRE F SOTPRINT:	TIRE SIZE .	7.00=8	MER Zedren
MEW A. USED	RETREAD BY		
5. 0. No 73-21	CODE NO.	!! FIPL .	X FLWHL
SKIU DEPIH	IA.	MAX. FOOTPRINT LC	TH7.361 In.
RATED INFLATION 125		MAX. COGIPRINI WE	ин6.125 in.
.100 S RATED LOAD6650	LBS.	NET CONTACT AREA	37.46 so. 1%.
23.499 DEFLECTION		GROSS CONTACT ARE	A 41.19 SO 15.
OPERATO"	DATE 3/13	1/79 SERIAL NR	B068C1

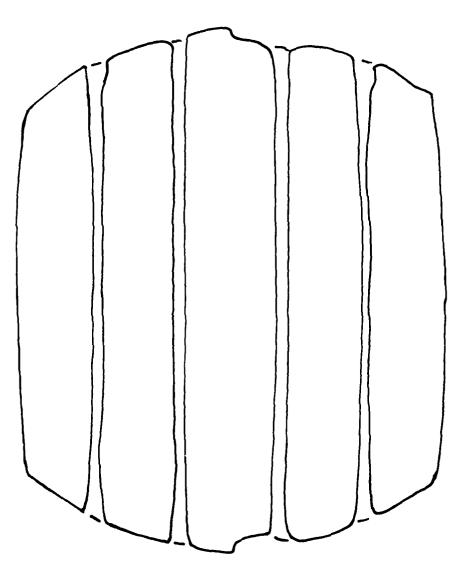


Figure C-13. Tire Contact Prints (Ecotprints)

TEST TIRE FOOTPRINT:	TIRE SIZE .	7.00-8	MFR Zedron
NEW X USED	RETREAD BY	N/A	
S. O. NO	CODE NO.	!! FLPL	x. ErMHT
SKID DEPTH	IN.	MAX. FOOTPRINT I	_GTH
DATED THE ATTON 125	DCT	MAY CONTROLNT I	INTU 2.938 IN
1.60 % RATED LOAD $1$	<sup>90</sup> LBS.	NET CONTACT AREA	$1 \dots 25 : 49 \text{ SQ. IN.}$
.17.14 DEFLECTION		GROSS CONTACT A	REA . 29:41 SQ. IN.
60 % RATED LOAD 39 17.14 DEFLECTION OPERATOR	DATE <sup>3/14</sup>	SERIAL NR.	B038C1

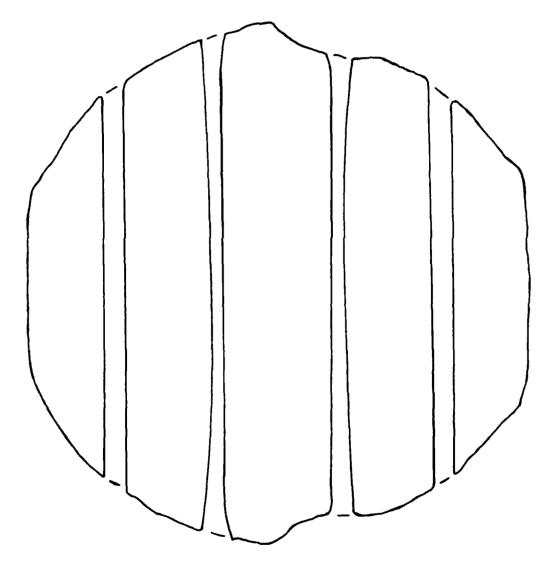


Figure C-14. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:			
NEWX. USED	RETREAD BY	. N/A	
S. O. NO	. CODE NO.	19 FLPL	. X FLWHL
SKID DEPTH	TN.	MAX. FOOTPRINT	IGTH 7.906 IN
RAJED INFLATION 125	PSI	MAX. FOOTPRINT	WDTH 6 . 1 5 6 IN .
.26:0% RATED LOAD6650	LBS.	NET CONTACT ARE	A <sup>36</sup> :19 SQ. IN.
DEFLECTION	2/0	<u>, G</u> ROSS CONTACT A	REA 40.83 SQ. IN.
RATED INFLATION 125 100 RATED LOAD 6650 26:06 DEFLECTION OPERATOR	. DATE 3/9/	SERIAL NR.	B078C4

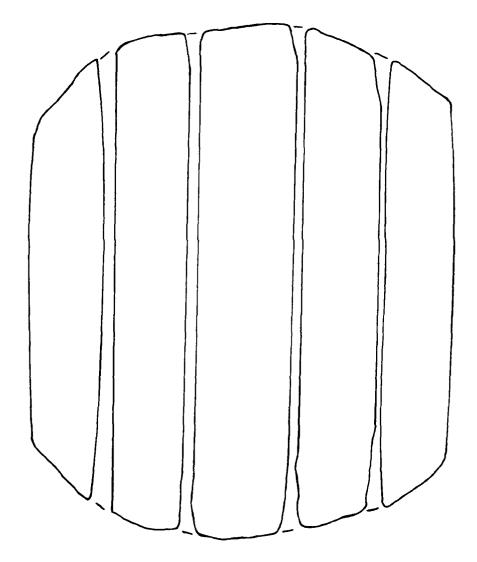


Figure C-15. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE .	7.00-8	MFR Zedron
NEW . A USED	RETREAD BY	Ν/Λ	
S. O. NO77-21	. CODE NO.	!9 FLPL	.X. FLWHL
SKID DEPTH	1N.	MAX. FOOTPRINT I	GTH6.Q63. IN.
RATED INFLATION 125. 17.13 RATED LOAD 35	PSI	MAX. FOOTPRINT V	JDTH5.938. IN.
	<sup>990</sup> LBS.	NET CONTACT AREA	1 24.01 SQ. IN.
1/:13. DEFLECTION		GROSS CONTACT AF	REA .28.23 SQ. IN.
OPERATOR	DATE 3/9/	79 SERIAL NR	.B078¢4

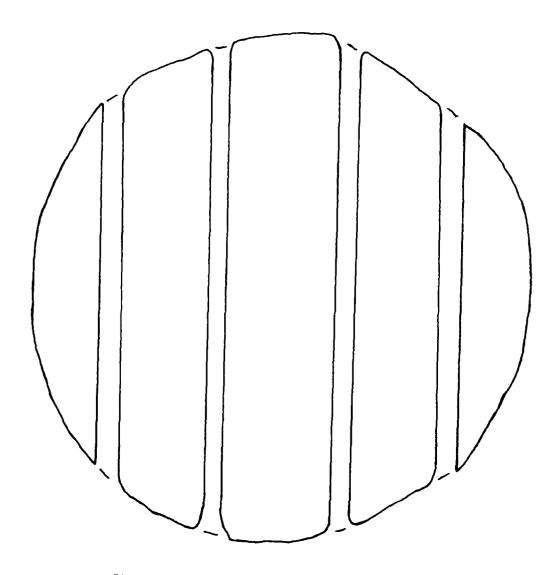


Figure C-16. Tire Contact Prints (Footprints)

TEST TEPT TOOTPRINT:	TIME SIZE .	7. Get=8.	MR Redren
M w	RETREAD BY		
5. O. NO 77-AL	CODE NO.	23 FIPE	Y FEWSE
SKID DEPTH	IN.	MAX. FOOTPRINE	tGTH7.414 In.
PATED INFLATION 425	PSI	MAX. FOOTPRINE	WEATH, 19.029 1 IN.
.100 % RATED LOADf			
20.791 DEFLECTION		GROSS CONTACT A	REA 38.98. SQ. Th
OPERATOR	DATE 3/7	779 SERIAL NR.	BUSSII 3

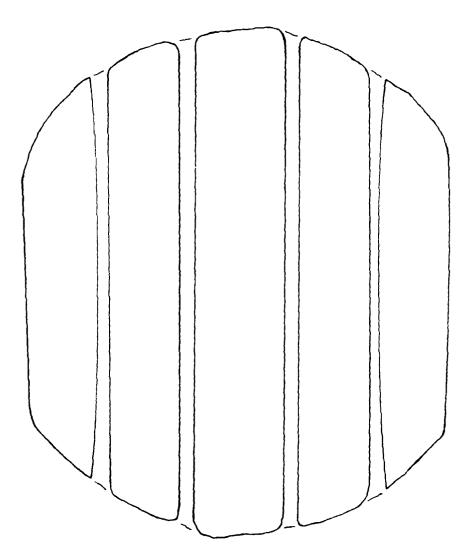


Figure C-17. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT: NEW USED S. O. NO	TIRE SIZE .	7.00-8	MFR Zedron
NEW USED 27	RETREAD BY	N/A	• • • • • • • • • • • • • • • • • • • •
S. O. NO	. CODE NO.	<sup>23</sup> FLPL	FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT L	.GTH5.938 IN.
RAJED INFLATION 125	PSI	MAX. FOOTPRINT W	DTH5:750 IN.
	LBS.	NET CONTACT AREA	\ <u>23</u> :7₫. SQ. IN.
DEFLECTION	2/7/	"GROSS CONTACT AR	EA 26:47. SQ. IN.
RATED INFLATION 125 60 % RATED LOAD 3990 15:93 DEFLECTION OPERATOR	. DATE 3///	SERIAL NR	B088H3

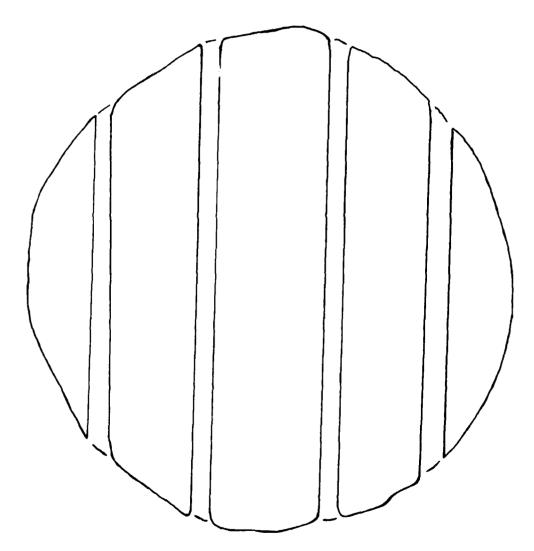


Figure C-18. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT: NEW 5 USED 77-21	LIRE SIZE	7.00-8	. MER Zedron
NEW A USED F	RETREAD BY	N/A 	· · • · · · · · · · · · · ·
S. O. NO	. CODE NO.	<sup>29</sup> FLPLX	FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT LGTH	7.750 IN.
RATED INFLATION125			
100 % RATED LOAD6650	LBS.	MET CONTACT AREA	38:02. SO. IN.
25.64 DEFLECTION		AIGA TIATUON 22090	42.08 sn tu
OPERATOR	. DATE - 2/1	27/78 SERIAL NRBO	8814

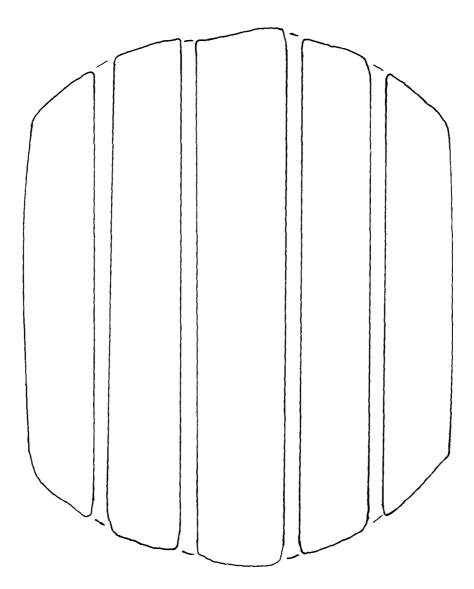


Figure C-19. Tire Contact Prints (Footprints)

TEST FIRE FOOTPRINE: NEW .X USED 77-21 S. O. NO	TIRE SIZE .		MFR Zedron.
S. O. NO	. CODE NO.	29 FLPL .X.	FLWHL
SKID DEPIH	IN.	MAX. FOOTPRINT LGTB	aL88 . IN.
RATED INFLATION125 60 % RATED LOAD399 17.06% DEFLECTION	PSI	MAX. FOOTPRINT WOTH.	.6.000. IN.
17.06 RATED LOAD???	'Υ LBS.	NET CONTACT AREA . 2	3.AL. SQ. IN.
OPERATOR	DATE 2/2	- GRUDS CUNTACT AREA . 7/79 CENTAL NO 8088	A. W. 1. 1 IN.
UPLKATUR	. DATE	7777 SERIAL NRT.T.	

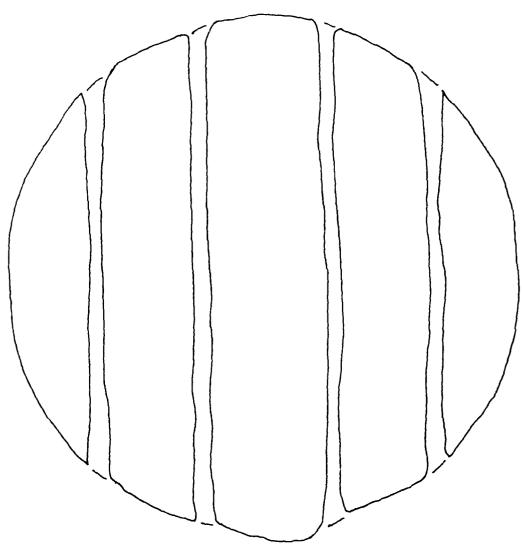


Figure C-20. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE	7.00-8	MFRZedron
NEWx USED	RETREAD BY		
S. 0. NO7.72.1			
SKID DEPTH	IN.	MAX. FOOTPRINT	LGTH. 7.00 IN.
RATED INFLATION 125	PSI	MAX. FOOTPRINT	WDTH. $6:19$ IN.
RATED INFLATION 125 100. % RATED LOAD6650	LBS.	NET CONTACT ARE	$A \dots 34:38 \text{ sq. } 10.$
30.43. DEFLECTION		GROSS CONTACT A	REA .38:04 SQ. IN.
20.43. DEFLECTION OPERATOR	DATE 11	/9/78 SERIAL NR.	B08813

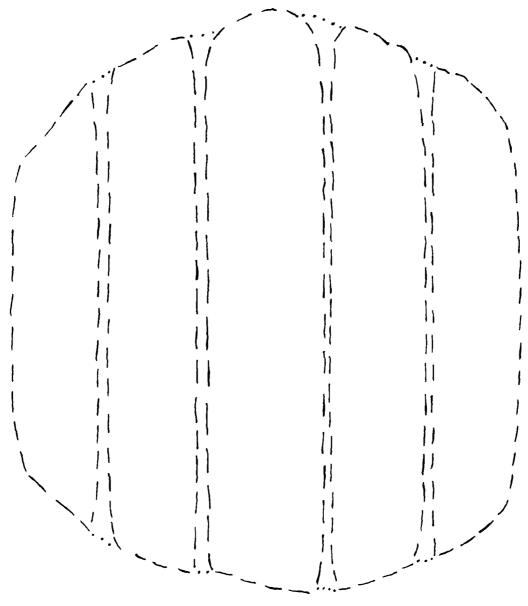


Figure C-21. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT: NEW . Y USED	TIRE SIZE .	7.00-8	MFR	Zedron
NEWX USED	RETREAD BY	N/A	.,	
S. O. NO77-21	CODE NO.	<sup>33</sup> FLPL	∴. FLW	Щ
SKID DEPTH	IN.	MAX. FOOTPRINT LGT	н	: IN.
RATED INFLATION125	. <i>.</i> PSI	MAX. FOOTPRINT WDT	ዝ३	٠٤٠. IN.
.60 % RATED LOAD3990	LBS.	NET CONTACT AREA .	23.80	SO. IN.
1371. DEFLECTION		GROSS CONTACT AREA	26.42	SQ. IN.
OPERATOR	DATE 11/9	9/78 SERIAL NR	Λ <b>\$</b> \$13	• • • • • • •

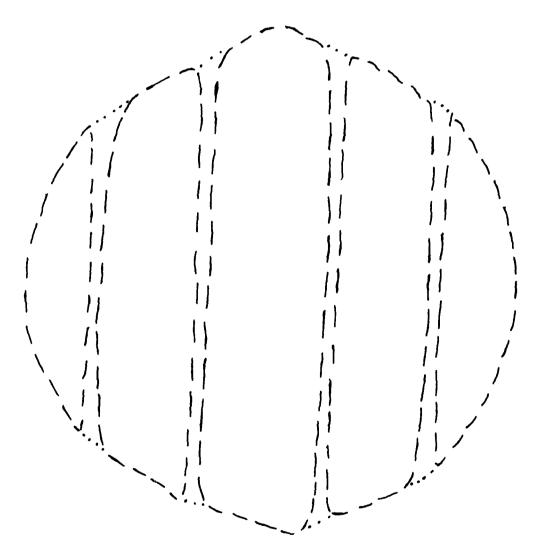


Figure C-22. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT: NEW X USED S. O. NO77-21	TIRE SIZE	7.00-8 N/A	MFR Zedron
NEW USED	KETREAU BY	'V.A	
S. U. NO	CODE NO.	FL	PL FLWHL
SKID DEPTH	IN.	MAX. FOOTPRIN	T LGTH7,259 IN.
RATED INFLATION 125 100 % RATED LOAD 66 23.22 DEFLECTION OPERATOR	PSI	MAX. FOOTPRIN	T WDTH6.188 IN.
RATED LOAD66	.50 LBS.	NET CONTACT A	REA36,07. SQ. IN.
.23:22 DEFLECTION	_ ,	GROSS CONTACT	AREA 38:99. SQ. IN.
OPERATOR	DATE 3/	<sup>2779</sup> SERIAL NR	B088K4

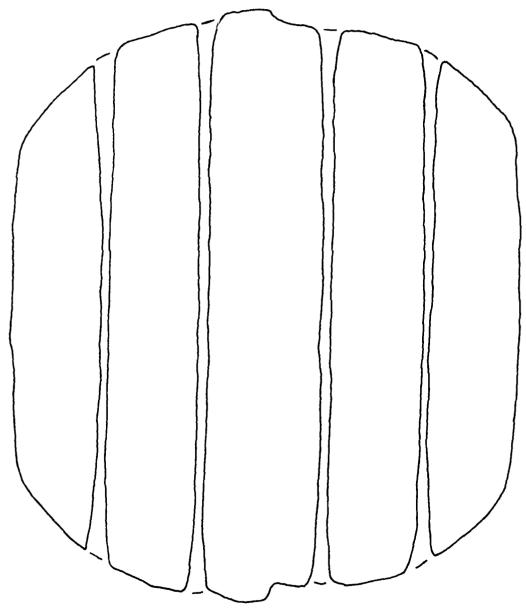


Figure C-23. Tire Contact Prints (Footprints)

TEST TURE FOOTPRINT:	TIRE SIZE	7.00-8	. MFR Zedron
5. 0. No	RETREAD BY	N/A	<i></i>
5. 0. 10	CODE NO.		FLWHL
S • 4 A Dis D. D. D.	T #.*	MAX FOOTDRINT LCTH	5.938 IN
RATED INFLATION 125 20 RATEL LOAD 3990 12:21 DEFELCTION OFERATOR	PSI	MAX. FOOTPRINT WOTH	. 31.701 IN.
- 'T'. RATEL LOAD????	LB5.	NET CONTACT AREA	138.98 Sú. IN.
ANAL DEFETICATION	2/2	, GROSS CONTACT AREA	SQ. IN.
- OPERATOR	DATE 3/-/	SERIAL NR	7.7

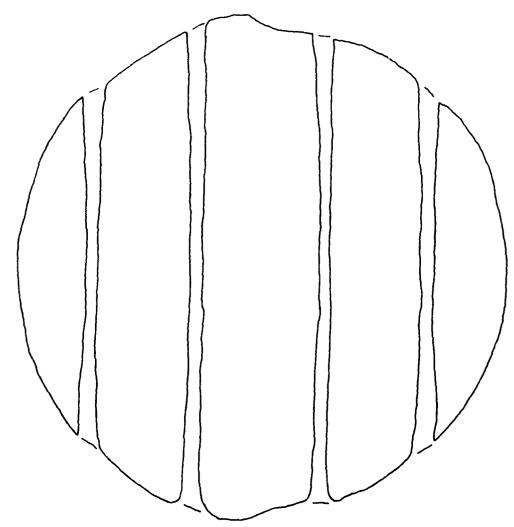


Figure C-24. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT: NEWX. USED 77-21 S. O. NO	TIRE SIZE RETREAD BY	7.00-18 	MFR Zedron
S. O. NO <sup>77–21</sup>	CODE NO.	<sup>43</sup> FLPL	X. FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT LO	STH6.750 IN.
RATED INFLATION 125 100 % RATED LOAD 6650 19.07 DEFLECTION OPERATOR	PSI	MAX. FOOTPRINT W	OTH. 6:063 IN.
.100 % RATED LOAD6650	LBS.	NET CONTACT AREA	30:77 SQ. IN.
.19.07. DEFLECTION		GROSS CONTACT ARI	A SQ. IN.
OPERATOR	DATE	3/1//9SERIAL NR	00901.3

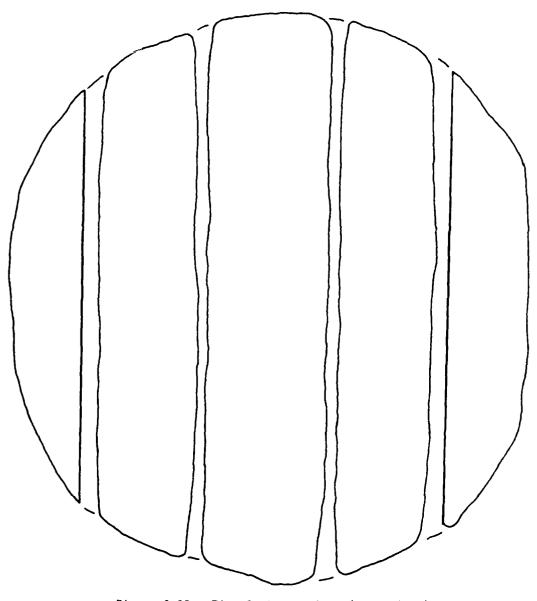


Figure C-25. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT: NEW X USED S. O. NO	TIRE SIZE .	7.00-8	MFR Zedron
NEW X USED	RETREAD BY	N/A	• • • • • • • • • • • • • • • • • • • •
S. O. NO	CODE NO.	<sup>13</sup> FLI	PL .X FLWAL
SKID DEPTH	IN.	MAX. FOOTPRIN	T LGTH5.438 IN.
RATED INFLATION125	PSI	MAX. FOOTPRIN	T WDTH. 5.375 IN.
.60 % RATED LOAD3990	LBS.	NET CONTACT A	REA $^{20.07}$ SO. IN.
.1.301 DEFLECTION		.GROSS CONTACT	AREA .22.84 SO. IN.
SKID DEPTH	DATE <sup>3/1</sup>	1/79 SERIAL NR	B0981.3

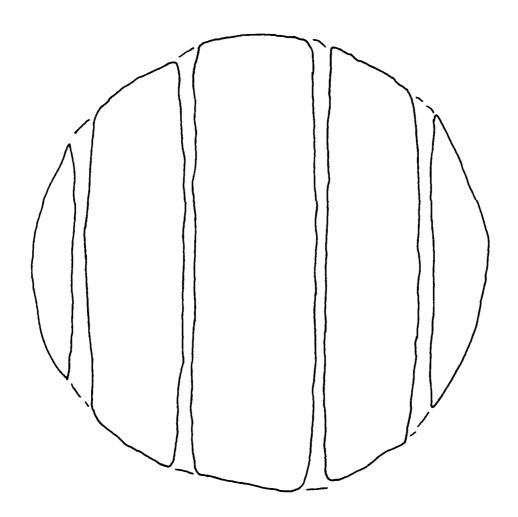


Figure C-26. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE .	7.00-8 CAST MFR Zedron
NEW USED	RETREAD BY	N/A
5. 0. No <sup>77-21</sup>	. CODE NO.	47 FLPL .X FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT LGTH6.375 IN.
RAJED INFLATION 125.	PSI	MAX. FOOTPRINT WOTH6.25 IN. NET CONTACT AREA31.95 SQ. IN. GROSS CONTACT AREA34.80 SQ. IN.
	LBS.	NET CONTACT AREA31:95 SQ. IN.
.18.60 DEFLECTION		GROSS CONTACT AREA .34:80 SQ. IN.
OPERATOR	. DATE 11/	15778 SERIAL NR B098M2

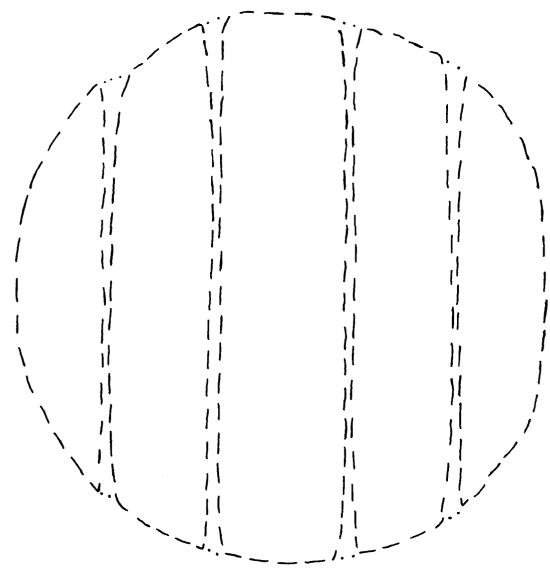


Figure C-27. Tire Contact Prints (Footprints)

HEST TIRE FOOTPRINT: SEW USED <sub>77221</sub>	HRE SIZE .	700-8 CAST	MER Zedron
WEN USED, I	RETREAD BY	- N/A 	
5 D MU	. CODE INC.		. IIWiiL
SELID DEPTH	Ba.	MAX. FOOTPRINE LGTH.	5.375. IN.
RATED INFLATION 125.	PSI	MAX. FOOTPRINE WOTH.	
	LBS.	NET CONTACT AREA	20:23 sq. in.
.M:// DEFLECTION		LUROSS CONTACT AREA .	22:20:50: IK.
SKID DEPTH	. DATE 11/1	5/78 SERIAL NP BOS	18M2 

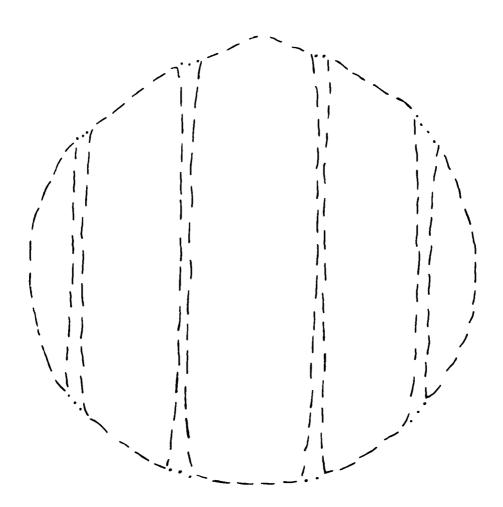


Figure C-28. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE 7.00-8 CAST MFR Zedron	
	RETREAD BY	
S. O. NO 77-21	CODE NO. 52 FLPL X FLWHL	
SKID DEPTH	IN. MAX. FOOTPRINT LGTH 6.875. IN	١.
RATED INFLATION125	PSI MAX. FOOTPRINT WOTH. $\frac{6.25}{100}$ IN	
.100 % RATED LOAD .6650	PSI MAX. FOOTPRINT WOTH. 6.25. IN. LBS. NET CONTACT AREA 33.55 SQ. IN.	
. 19.64 DEFLECTION	GROSS CONTACT AREA 37.17 SQ. IN DATE 11/16/78 SERIAL NR. B098N2	١.
OPERATOR	DATE 11/16/78 SERIAL NR BU98N2	

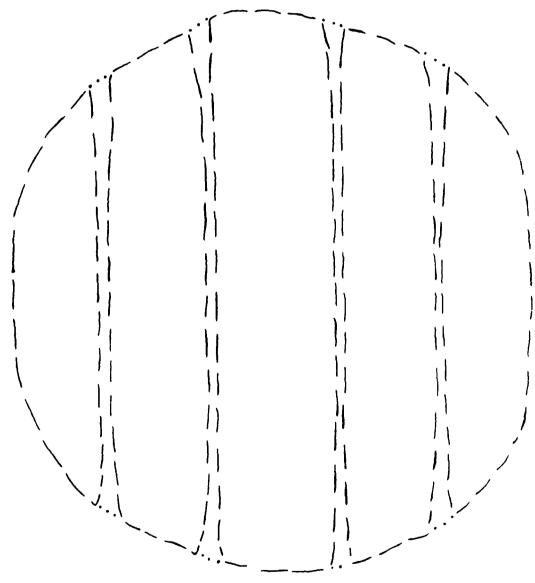


Figure C-29. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT: NEW .X USED 77-21 S. O. NO77-21	TIRE SIZE .	7.00-8 N/A	CAST	MFR Zedron
S. 0. NO	CODE NO.	52	FLPLX	. ELWHL
SKID DEPTH	IN.	MAX. FOOTP	RINT LGTH.	$\frac{1}{2}$ : $\frac{1}{2}$ : IN.
SKID DEPTH	PSI	MAX. FOOTP	R <b>INT</b> WDTH.	21.9% IN.
. 73.5% RATED LOAD	LBS.	NET CONTAC	T AREA	$\frac{21}{24}$ : $\frac{04}{28}$ SQ. IN.
DEFLECTION	11/	GROSS CONT	ACT AREA	29.70 SQ. IN.
OPERATOR	DATE '''	SERIAL	NR	.,

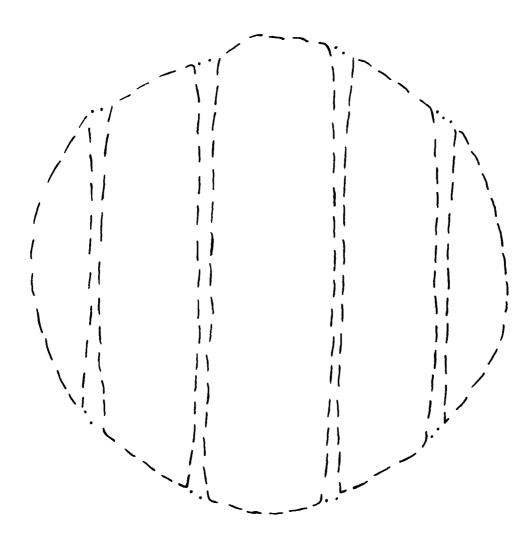


Figure C-30. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE.	7.60-8	CAST	MFR Zedro	ņ
NEWX USED	KETKFAD BA	MA		· · · · · · · · · · · ·	
S. O. NO	CODE NO.	5/	FLPL	¥ FLWHL	
cvin broiu	T NI	MAY COOTE	DIMIT LOTH	6.94	TAI
RAJED INFLATION 125.	PSI	MAX. FOOTP	RINT WDTH.	6.313	IN.
RATED LOAD	LBS.	NET CONTAC	T AREA	35:19 SQ.	IN.
. <sup>20</sup> :48 DEFLECTION		GROSS CONT	ACT AREA .	38.00 SQ.	IN.
RATED INFLATION 125 100 & RATED LOAD 6650 20:48 DEFLECTION OPERATOR	DATE 11/	13/78 SERIAL	NR B	09802	

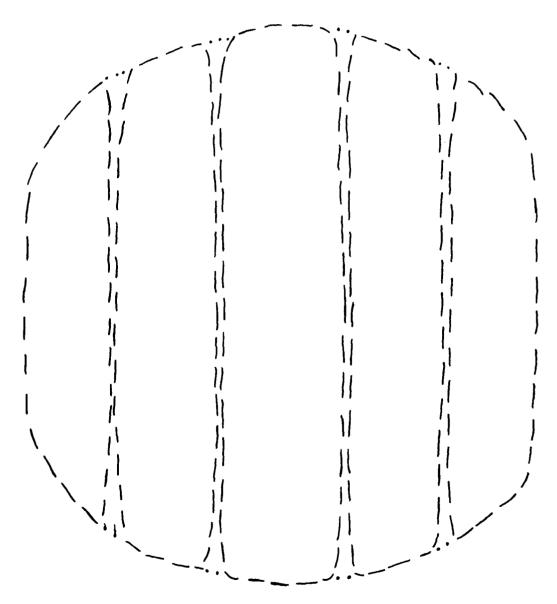


Figure C-31. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT: T	TRE STZE	7.00-8	.CAST M	FR Zedron.
NEWX USED F	RETREAD BY			
S. O. NO 77-21	CODE NO. 57.	FL	PLx	FLWHL
SKID DEPTH				
RATED INFLATION 125	129	MAX. FOOTPRIN	IT WDTH	.5.635. IN.
.60 % RATED LOAD3990	? LBS. 1	IET CONTACT A	REA41	.73 SQ. IN.
13.85 DEFLECTION	(	ROSS CONTACT	AREA .34	.86 SO. IN.
OPERATOR	. DATE <sup>11/13/</sup>	<sup>78</sup> SERIAL NE	В098	802.,

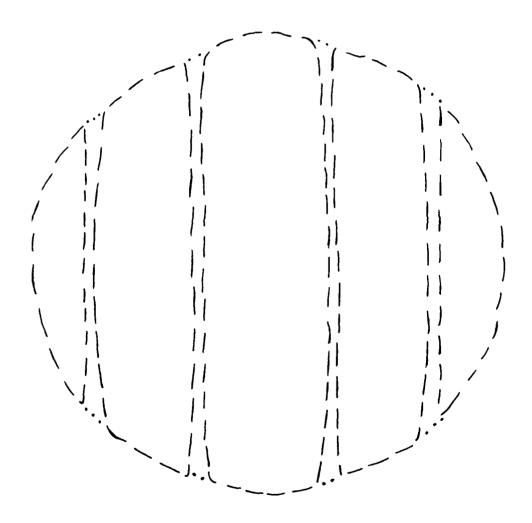


Figure C-32. Tire Contact Prints (Footprints)

Test the Footballing	1.3 51/6	7.0945 (14.1)	. Med Zadalelli
March 1810	PETER SE	///	
-3. 6. ha	(O.a. Nil.)		ELWar
5840 00 de la	IN.	MAX. FOOTPRINE Line	
-RATES USEATION			
199 - PATER LOAD555	1	THE ECONTACT AREA	
REW DEFILERION		GROSS CONTACT ALLE	. Marie 1907, 180
CPERALP	DATE <sup>117</sup>	<sup>14778</sup> SERIAL NR	Markali Markali

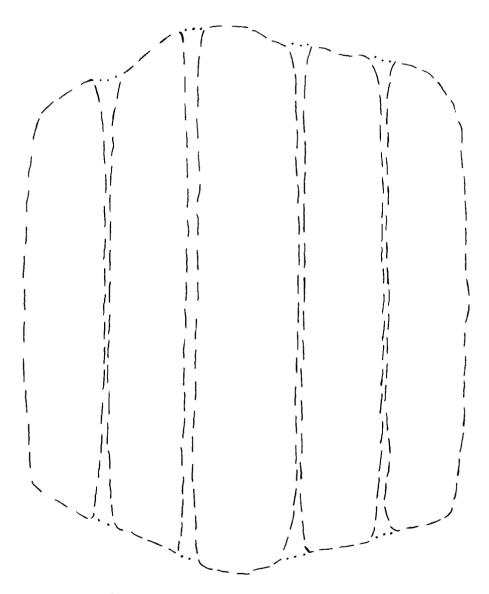


figure (-33. Time Contact Prints (Foetprints)

TEST TIRE FOOTPRINT:					
NEWX USED	RETREAD BY	N/A			
S. 0. NO. 77-21	CODE NO.	$\frac{62}{1}$	FLPL X	. FLWHL .	
SKID DEPTH	IN.	MAX. FOOTPR.	INT LGTH.	6.00	. IN.
SKID DEPTH 125 RATED INFLATION 125 19:24 RATED LOAD 31 DEFLECTION	PSI	MAX. FOOTPR	INT WDTH.	6.25	.IN.
RATED LOAD3	990 LBS.	NET CONTACT	AREA	.27.57 <sub>SD</sub> .	. IN.
.19.24 DEFLECTION		GROSS CONTAI	CT AREA .	.30.96s0.	. III.
OPERATOR	DATE 11/14	4/78 SERIAL I	VR. 30	98P2	

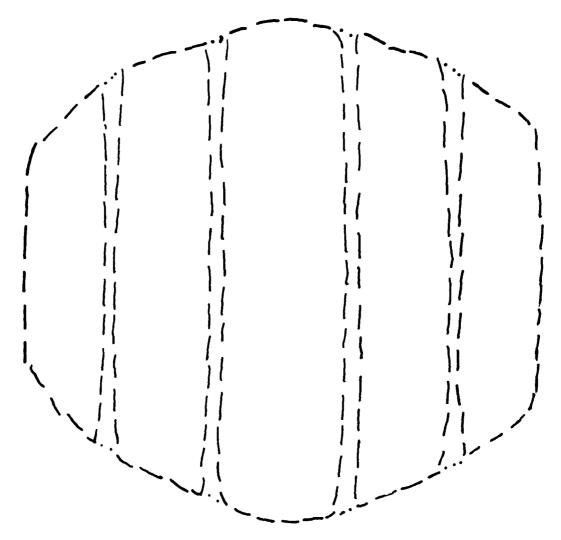


Figure C-34. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:				
NEW X. USED	RETREAD BY	N/A		· • • • • • • • • • • • • • • • • • • •
S. O. NO	CODE NO.	68	. flPL .¾	FLWHL
SKID DEPTH	IN.	MAX. FOOTS	RINI LGIH.	. 7:0 IN.
RATED INFLATION125				
100 % RATED LOAD 66	50 LBS.	NET CONTAC	J ARLA	33:76. SQ. IN.
19.87 DEFLECTION		GROSS CONT	ACT AREA	<sup>17</sup> :52. 50. IN.
OPERATOR	DATE 11/3	<sup>R/78</sup> SERIAL	NR	9803

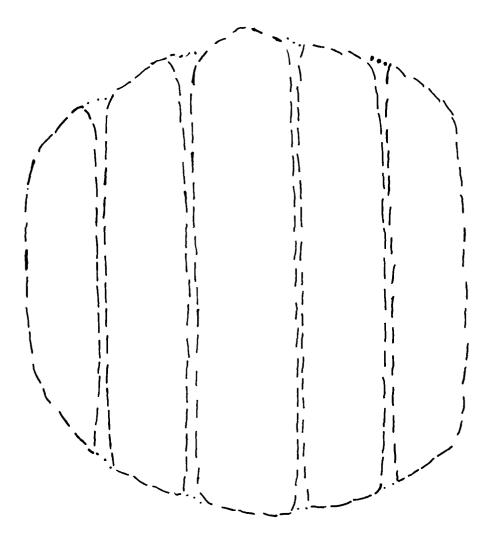


Figure C-35. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE . 00-	8 CAST	. MFR Zedron
MEW A. USED	RETREAD BY		· • • • • • • • • • • • • • • • • • • •
S. O. NO 77-21	CODE NO	.68 FLPL .X.	FLWHL
SKID DEPTH			
RATED INFLATION125.			
60 % RATED LOAD3990.			
.1.37.1. DEFLECTION	, G	ROSS CONTACT AREA	.25.69 SQ. TN.
OPERATOR	DATE 11/8/7	8 - SERTAL NR. BO98	<u> </u>

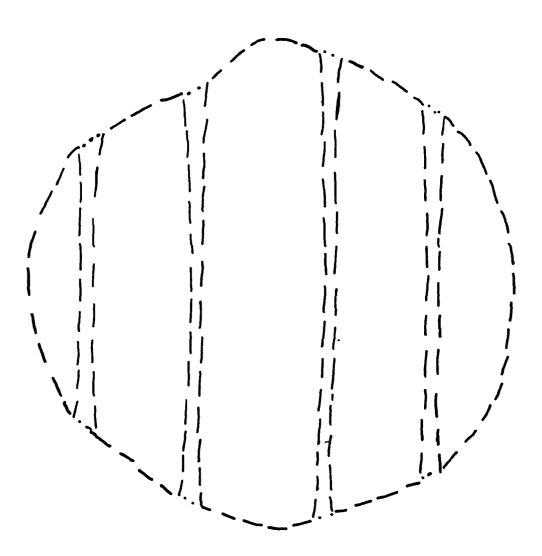


Figure C-36. Tire Contact Prints (Footprints)

HIST TIPE FOOTPRINT:	TIRE SIZE .	7.00-8 CAST., MER Zedron
- NewY USED	RLIEFALL BY	MA
		72 FLPL .X FLWHL
		MAX. FOOTPRINT LGTH8.625 IN.
RATED INCLATION	2 PSI	MAX. FOOTPRINT WOTH. 16:625. IN.
₹00 ¼ RATED LOAD552!	L85.	NET CONTACT AREA44.12 SQ. IN. GROSS CONTACT AREA48.58 SQ. IV.
.2344 DEFLECTION	11/	GROSS CONTACT AREA . 28:28 SQ. 111.
OPERATOR	DATE '''	17/78 SERIAL MR B098k2

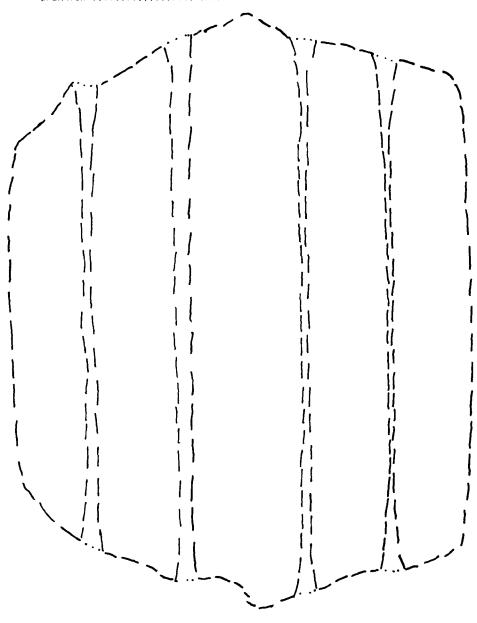


Figure C-37. Tire Contact Prints (Footprints)

LEST TIRE FOOTPRINT:	TIRE SIZE	7.00-8	CAST	MFR Zedron
NEW X USED	RETREAD BY	N/A		
S. O. NO	CODE NO.	72	. FLPLX.	. FLWHL
SKID DEPTH	IN.	MAX. FOOT	PRINT LGTH.	6.875. IN.
RAJED INFLATION 1.2.5	PSI	MAX. FOOT	PRINT WOTH.	6.44 IN.
% RATED LOAD	.3990 LBS.	NET CONTA	CT AREA	$\frac{29.30}{10.00}$ SQ. IN.
.21:20 DEFLECTION		GROSS CON	TACT AREA .	34.26 SQ. IN.
RATED INFLATION 125 60 % RATED LOAD 21.20 DEFLECTION OPERATOR	DATE 11/1	///8 SERIA	L NR	B098R2

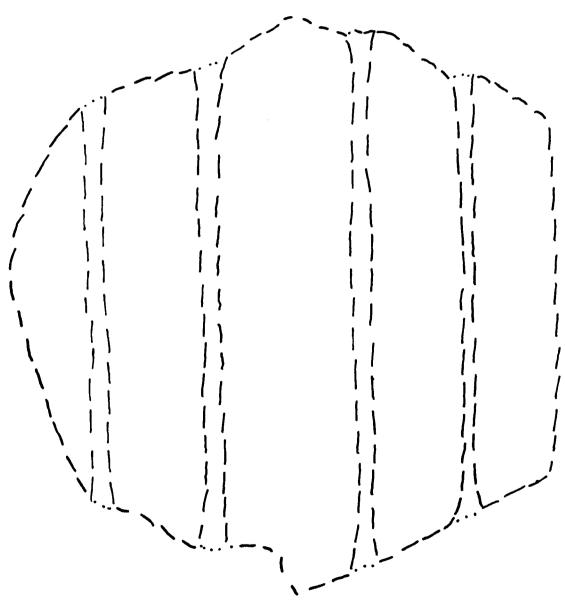


Figure C-38. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT: NEWX. USED	TIRE SIZE .	7.00-8 N/A	MFR Zedron
S. O. NO			
SKID DEPTH	IN.	MAX. FOOTPRINT L	GTH6.750 IN.
RATED INFLATION125.			
.100 % RATED LOAD6650	LBS.	NET CONTACT AREA	.31.12 SQ. IN.
.19.42 DEFLECTION	0.104	GROSS CONTACT AR	EA .34:46 SQ. IN.
OPERATOR	DATE 2/28	3/78 SERIAL NR	B098 <b>S</b> 3

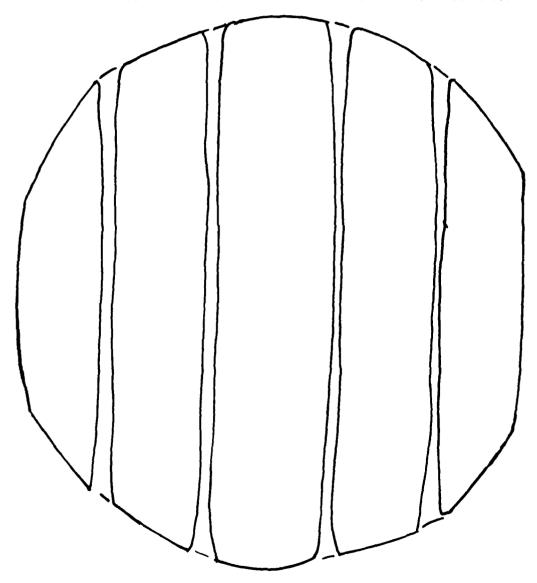


Figure C-39. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE	7.()()-8	MFR Zedron
NEWX. USED			
S. O. NO77-21			
SKID DEPTH	IN.	MAX. FOOTPRINT LG	TH. 15:563 IN.
RATED INFLATION125	PSI	MAX. FOOTPRINT WD	TH. 3.344 IN.
.50 S RATED LOAD 3990	LBS.	NET CONTACT AREA	${33}^{20}:_{82}^{82}$ SQ. IN.
13.66 DEFLECTION OPERATOR	2.7	GROSS CONTACT ARE	$A_{p,\alpha}$ SQ. IN.
OPERATOR	DATE - <sup>27.</sup>	<sup>28779</sup> SERIAL NR	B03023

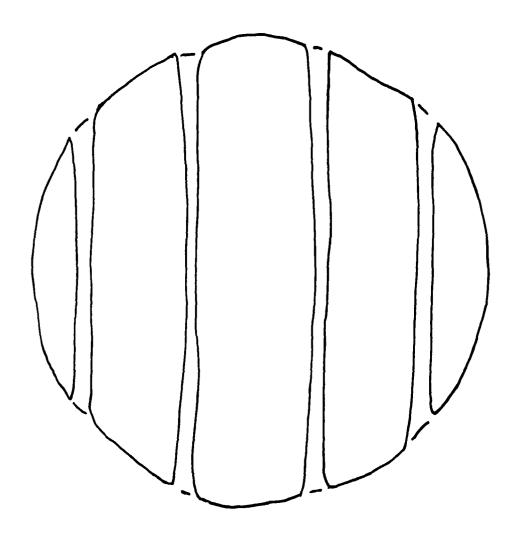


Figure C-40. Tire Contact Prints (Footprints)

TEST FIRE FOOTPRINT: NEW . S. USED S. O. NO	TIRE SIZE .	7.00-8 N/A	MFR Zedron
NEW USED 77.31	KETKEAU BY	85	
S. 0. NO	CODE NO.	, FLPL	ELMAR
SKID DEPTH	IN.	MAX. FOOTPRINT	LGTH. $\frac{9.578}{1.53}$ . IN.
RATED INFLATION .125	PSI	MAX. FOOTPRINT	WDTH 6.3/16. IN.
LUU 9/ DATEN LAAN - DD	DO 10C	NET CONTACT ADD	A JU.49 CA IM
18:46 DEFLECTION OPERATOR		_GROSS CONTACT A	REA 33:30 SQ. IN.
OPERATOR	DATE 11/1	///8 SERIAL NR.	R098.L5

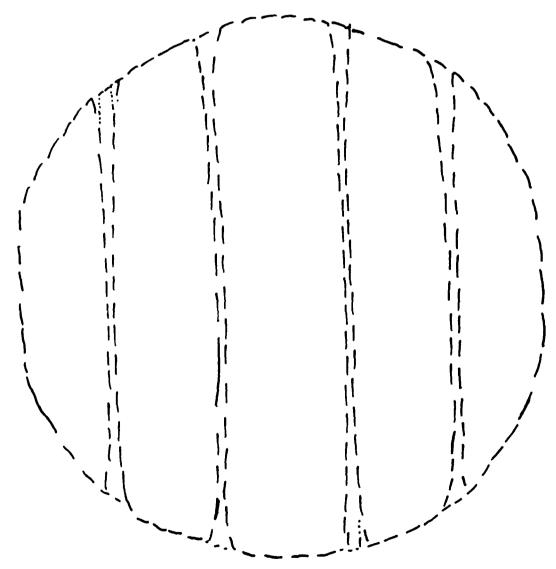


Figure C-41. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE .	7.00-8	MFR Zedron
NEW X. USED	RETREAU BY	N/Λ	
NEW S. USED	CODE NO.		X FLWHL
SKID DEPTH			
RATED INFLATION 125	PSI	MAX. FOOTPRINT	WDTH. 5.1/4 IN.
. 60. % RATED LOAD .3990	LBS.	NET CONTACT ARE	A19.42. SQ. IN.
J2-75 DEFLECTION		GROSS CONTACT A	NREA 22.18. SQ. IN.
OPERATOR	DATE 11/1	7/78 SERIAL NR.	B0987?

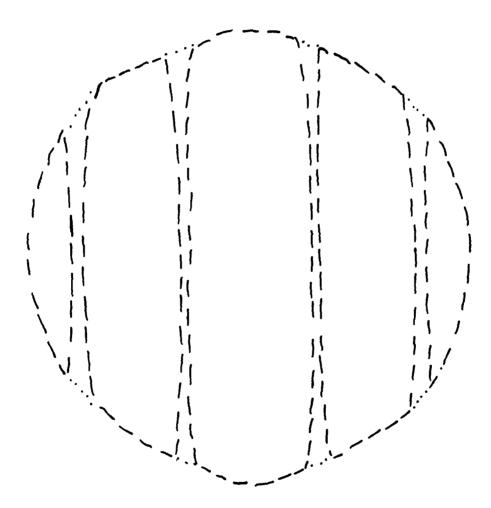


Figure C-42. Tire Contact Prints (Footprints)

HST TIRE FOOTPRINT:	TIRE 517E		MFR Zedron
NEW X USED	RETREAD BY		
5. 0. No77:21			
SKID DEPTH	IN.	MAX. FOOTPRINT LGTH.	.7.750 . IN.
RATED INFLATION12	5 PSI	MAX. FOOTPRINE WUTH.	.6.375 IN.
.100. % RATED LOAD665	O LBS.	NET CONTACT AREA	18.77 SQ. IN.
25.90 DEFLECTION		CROSS CONTACT AREA	43.04 SO TN
OPERATOR	DATE 7/2	3/79 SERIAL NR8128	;;3

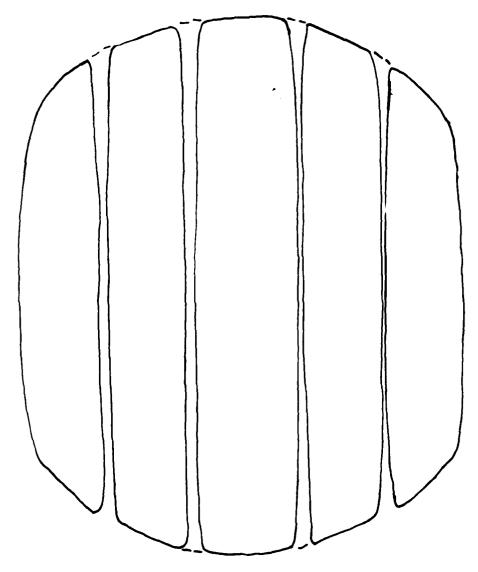


Figure C-43. Tire Contact Prints (Footprints)

SEAL TIRE FURSEPRIMES			
War and a Colog Comment	Priliteral EY		
o. o. w <i>"Add</i>			
Selvicity			
PARLO MALATIONLES.	Pp1	MAX, FOUTERINE Weller	
! RATED LOAD3990.	1 bear	THE CONTACT AREA ?	
High Developer		Capasa Citate V maja il	
Firm Develorition (September 11	DATE 🧺	"BARAN TRO LINES	·

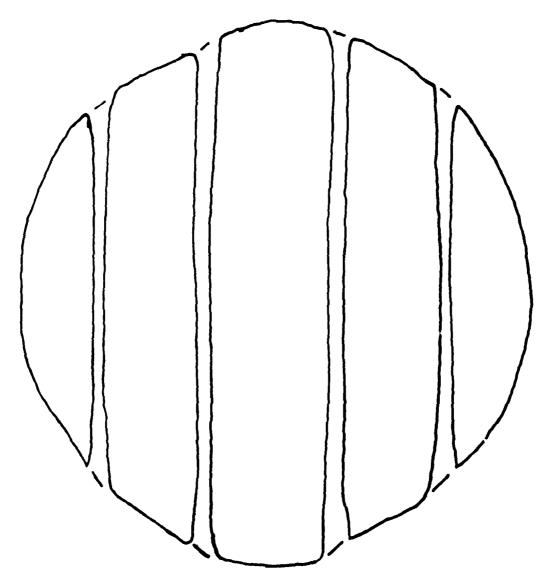
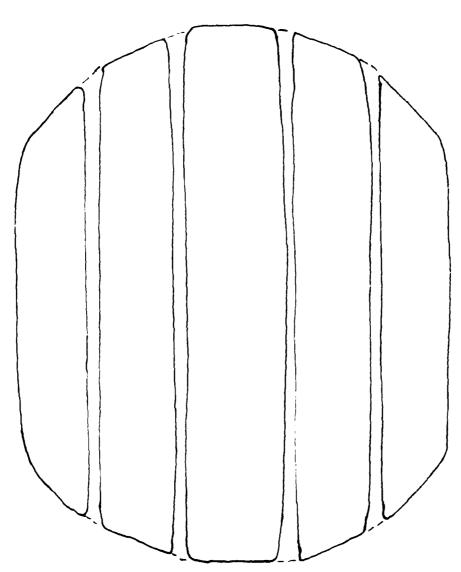


Figure C-44. Tire Contact Prints (Footprints)

		1.41 5171	7. a.s.	MI २ शुल्तुहरूक्
,		The thirt Y		
			State Constant Consta	184 miles
			March 1, of state the group.	1
 ·	11 10 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	e e e e e e e e e e e e e e e e e e e	THE CONTRACT PRIA	그를 뭐. 뭐.
		i line.	POST CONTACT AREA	13503 34 14.



The second secon

TEST TIRE FOOTPRINT: I	TIRE SIZE .	7.00-8	MFR Zedron
NEW X USED F	RETREAD BY	N/A	. <b></b>
5. 0. No	. CODE NO.	91 FLPL	. FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT LGTH.	. <u>6:313</u> IN.
PATED INFLATION 125	120	MAY FOOTODINT MOTH	2.688 IV
<u>60</u> % RATED LOAD3990	LBS.	NET CONTACT AREA?	24.28 SQ. IN.
.15.16 DEFLECTION	0.10	GROSS CONTACT AREA	28.85 SQ. IN.
.60 % RATED LOAD .3990 .15.16 DEFLECTION OPERATOR	. DATE 2/2	<sup>22/79</sup> SERIAL NR	B128V3

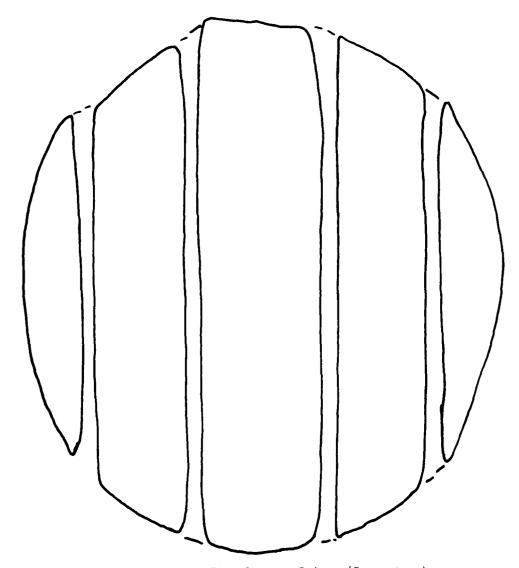


Figure C-46. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT: NEW X USLD 777-31 S. O. NO77-31	TIRE SIZE	7.00-8	. MER Zedrer
NEW Z USED	RETREAU BY		<i>.</i>
S. O. NO	CODE NO.		Hagai
SKID DEPTH	IN.	MAX, FOOTPRIAT LGTH	/.500. IN.
RAJED INFLATION	PSI	MAX. FOOTPRINT WOTH	fs:{!3. m.
100 \$ RATED LOAD	LBS.	NEL CONTACT AREA	. 15:22 Sq. 10.
SKID DEPTH 1.55 PATED INFLATION 1.55 100 % RATED LOAD 5650 23:55 DEFLECTION OPERATOR	. ,	, GROSS CONTACT AREA.	. <u>39:3</u> 5 SQ. IN.
OPERATOR	DATE	<sup>PIZZA</sup> SERIAL NR	11791.14 

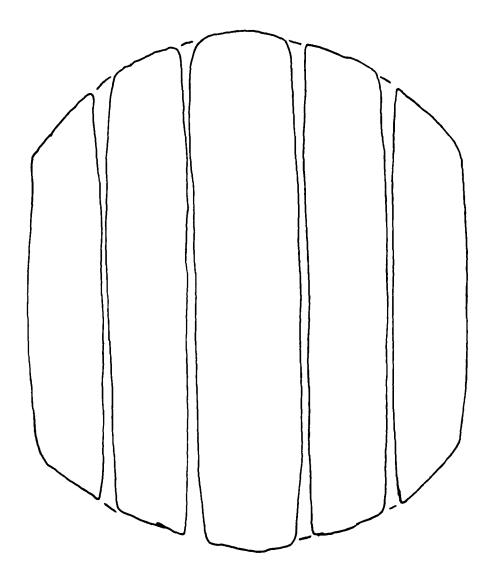


Figure C-47. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE	7.00-8	MER Zedron
NEW .X USED	RETREAD BY		
c 0 M0 //=21	CODE NO	24 (10)	A FILMUI
SKID DEPTH  RATED INFLATION 12  60 % RATED LOAD 399 15.69 DEFLECTION  OPERATOR	IN.	MAX. FOOTPRINT	LGTH:::00 IN.
RATED INFLATION $\dots$ $12$	.5 PSI	MAX. FOOTPRINT	WOTH. , 5; 50 III
	<sup>∩</sup> LBS.	NET CONTACT ARE	A $\frac{24}{34}$ : 10 SQ. IN.
15:69. DEFLECTION		GROSS CONTACT A	REA . 27.10 SQ. 13.
OPERATOR	DATE $^{3/2}$	1/79 SERIAL NR.	B029W3

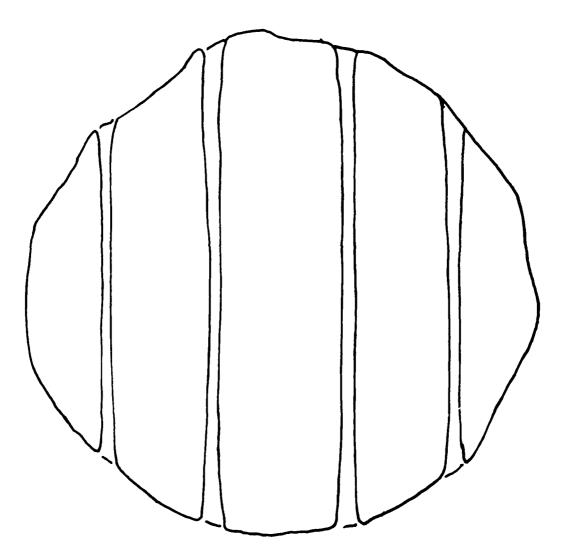


Figure C-48. Tire Contact Prints (Footprints)

TEST TIGHT FOR FRINT:	TIRE SIZE .		. MER Zedice
State of the state	Pi IRino by	<u>.</u> YA	<b></b>
-5. 0. Mat	coor to.	27 ftpt .š.	H.WHU
SKID 0rF16	IN.	MAX. FOOTPRINT LGTH	7.375 14.
- RAJED INH ALION 点点	PSI	MAX, FOOTPRINE WOTE	. 5.313
PATED INFLATION 125 11 (s. 8 RATED LUAD 5650 DEFILITION	L85.	NET CONTACT AREA	.,15-21 sq
CINITE DEFECTION		JUROSS CONTACT AREA	. 49:42 SQL 15 -
05F49105	DVIF 3/1;	<sup>I//Y</sup> SERIAL NRBO:	*9X

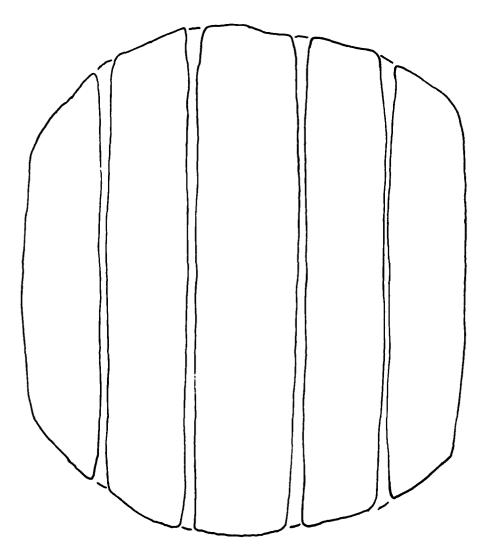


Figure C-49. Tire Contact Prints (Footprints)

ILST TIRE FOOTPRINT:			
NEW .X USED			
S. O. NO. 77-21	CODE NO.	97 FLPL	FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT L	GTH6:063 IN.
RATED INFLATION125	PSI	MAX. FOOTPRINT W	DTH6:99 IN.
.60. % RATED LOAD3990	LB:	NET CONTACT AREA	<sup>24</sup> :02 Sq. IN.
ኒን.65. DEFLECTION		GROSS CONTACT AR	EA .28:09 SQ. IN.
15.65. DEFLECTION OPERATOR	DATE <sup>3/21</sup>	1/79 SERIAL NR	B039X3

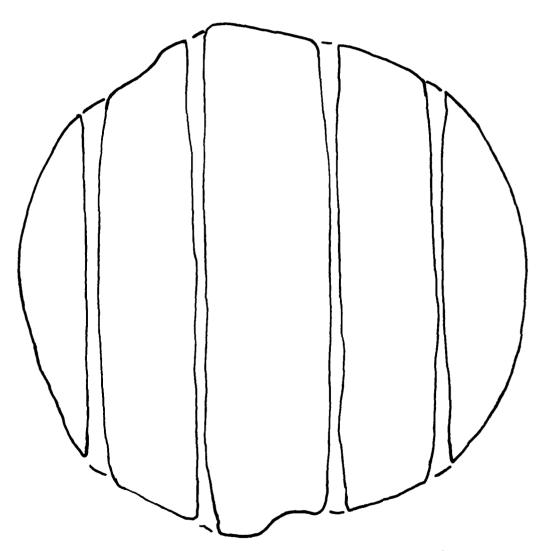


Figure C-50. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE .	7.00-8	MFR <sup>Zedron</sup>
NEWX USED			
S. O. NO7.7-21	CODE NO.	<sup>100</sup> FLF	PLX. ELWHL
SKID DEPTH	IN.	MAX. FOOTPRINT	LGTH7:016 IN.
RATED INFLATION . 125	129	MAX. FOOTPRINT	WDTH6:297 IN.
.100 % RATED LOAD	.6650 LBS.	NET CONTACT AF	REA35:88 SQ. IN.
.23:26 DEFLECTION		GROSS CONTACT	AREA .38.56 SQ. IN.
.23.26 DEFLECTION OPERATOR	DATE $^{3/2}$	<sup>2/79</sup> SERIAL NR.	B029Y3

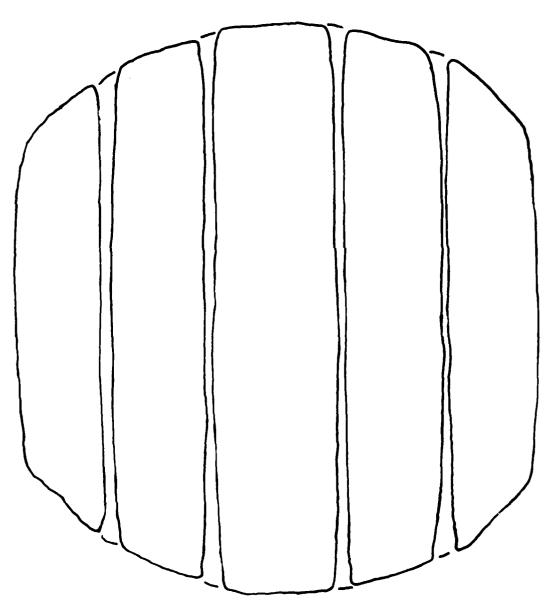


Figure C-51. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:			
NEW .X USED	RETREAD BY	N/A	
S. O. NO	CODE NO.	100 FL	PL Y. FLWHL
SKID DEPTH	IN.	MAX. FOOTPRIN	IT LOTH. 5:906. IN.
RATED INFLATION125	PSI	MAX. FOOTPPIN	T WDIH6:000 IN.
	90LBS.	NET CONTACT A	REA24:90 SO. IH.
15.56 DEFLECTION		FORTER 2 20CO	- дргд - 27.46 ка — н
OPERATOR	DATE <sup>3/20</sup>	<sup>2779</sup> SERIAL NR	

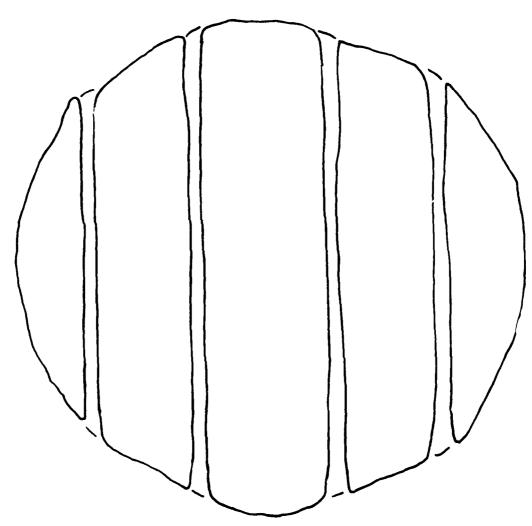


Figure C-52. Tire Contact Prints (Footprints)

itst lijke footPRIM:	TIRE SIZE	7,00-8	Mfg Zedrom
NEW USED	RETREAD BY		
NEW USED S. O. NO	CODE NO.		. El Wal
SKID DEPTH	IN.	MAX. FOOTPRINT LGIH.	7.781 15
RAJED INFLATION	ÇQ., PSI −	MAX. FOOTPRINE WOTH.	
RATED LOAD	BhSC $LBSC$	NET CONTACT AREA	40:13 so. in.
.TT: CO DEFLECTION		GROSS CONTACT AREA .	$1.12^{+3}80$ . III.
SKID DECTH  RATED INFLATION  100 RATED LOAD  24.91 DEFLECTION  OPERATOR	DATE 3/2	<sup>2779</sup> SERIAL NR <u>B</u> 02	923

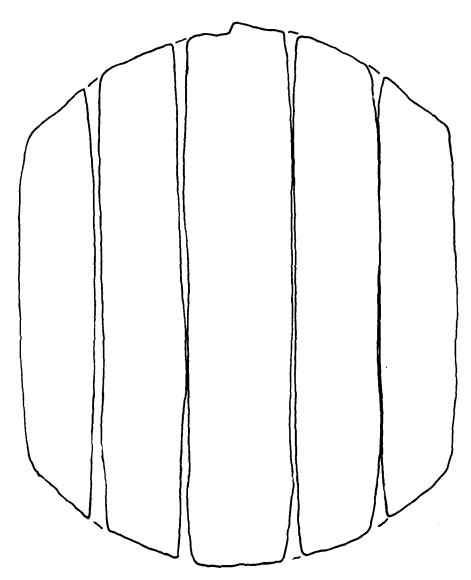


Figure C-53. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT: NEWX USED S. O. NO	TIRE SIZE .	7.00-8	MFR Zedron
NEW USED 55.51	RETREAD BY	······································	• • • • • • • • • • • • • • • • • • • •
S. O. NO	CODE NO.	FLPL A	. FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT LGTH.	.6.200 IN
RATED INFLATION	5 PSI	MAX. FOOTPRINT WDTH.	
.60. % RATED LOAD399	<sup>0</sup> LBS.	NET CONTACT AREA	26.82 SQ. IN.
16.55. DEFLECTION		GROSS CONTACT AREA .	1.129.730. IN.
16:55 DEFLECTION OPERATOR	DATE $3/2$	<sup>2/79</sup> SERIAL NR	B029Z3

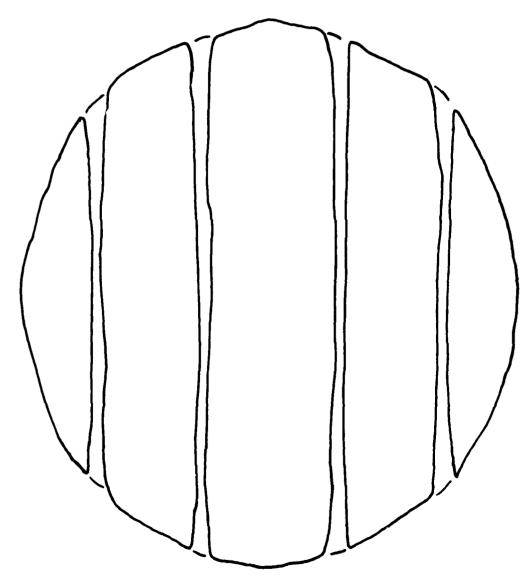
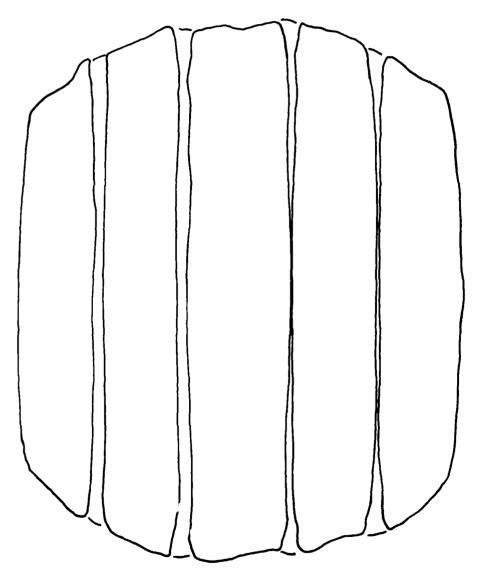


Figure C-54. Tire Contact Prints (Footprints)

Hal liki bodadali	1100 50.6	······································
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- KATE 2 156 AT 196	aran aran etsi.	Well Feedbriet words Assett 15
The State Collins		MILLIONIALI AREA MALLESS. IN
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COLUMN CO	· · · I with the	1.1 (1.8.M) (48



Eugene ( G. Tire Confact Prints (Footprints)

TEST THE FOOTPRINT:	TIRE SIZE .	7.00-8	MFR Zedron
THE WALL USED	RETREAD BY	N/A	<b></b>
3. 0. 0. 0 77-21	. CODE NO.	106 FLPL .	.X FLWHL
SELD DEPTH			
Paird instation $\dots$ .125			
THE BATED LOAD . 3990	LBS.	NET CONTACT AREA	<sup>28</sup> :50 SQ. IN.
ARMS DEFLECTION		GROSS CONTACT ARE	A .30:81 SQ. IN.
GERALDR	DATE <sup>3/19</sup>	9/79 SERIAL NR	В029АА3

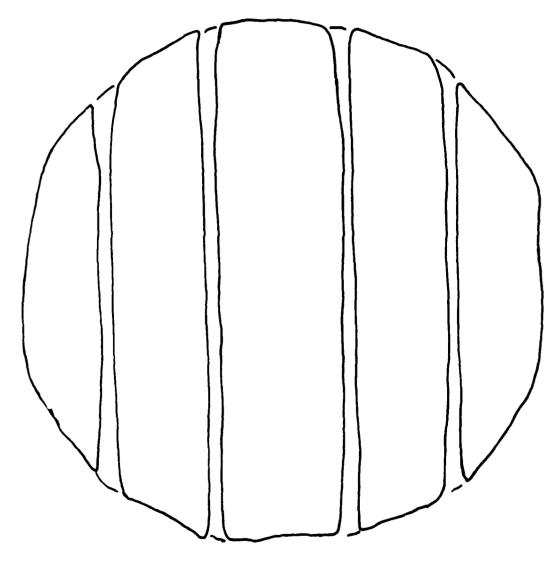


Figure C-56. Tire Contact Prints (Footprints)

The High Form RIME	11ki 51/I		Med Galace
-900 (50) <del>77.5</del> 7.55	RUTREAD BY		
And L. Y. LOSED MARK. Sections of the Control of th	CODE Tag. The	THE COLUMN TO SEE	UW,
SETT PER THE TOTAL SET TO SET	PST	MAX FOOTPRINE (S):	
RAIL INTAILON 1771 - A RAIL LOAD 1866 - A RESERVED 108 OPENATOR	185.	NET CONTACT AREA	10 grafieres - 100 grafie
- 2/1/4 15 to FC 110%	, , ,	CAROSS FORTH LANGER	
Of Charles and the contract of	DAH 🔭	1277 OFBIGE NR	

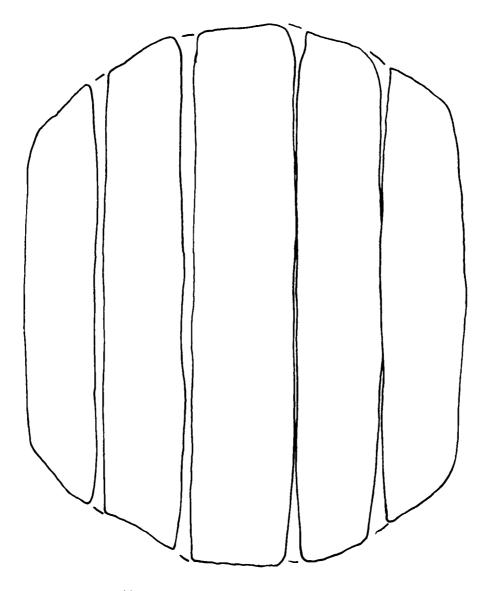


Figure C-57. Tire Contact Prints (Footprints)

TEST TIRE FOOTPRINT:	TIRE SIZE .	7.00-8	MFR Zedren.
NEWX USED	RETREAD BY	Ņ/Ā	
S. O. NO 77-21	CODE NO.	109 FLPLX	. FLWHL
SKID DEPTH	IN.	MAX. FOOTPRINT LGTH.	.6.375 IN.
RATED INFLATION	125 PSI	MAX. FOOTPRINT WOTH.	.6.125 IN.
50 % RATED LOAD3999	) LBS.	NET CONTACT AREA	27.39 SO. IN.
.17.478 DEFLECTION		GROSS CONTACT AREA .	30.54 SQ. IN.
OPERATUR	DATE 3/20	<sup>1/79</sup> SERIAL NR	B029BB3 `

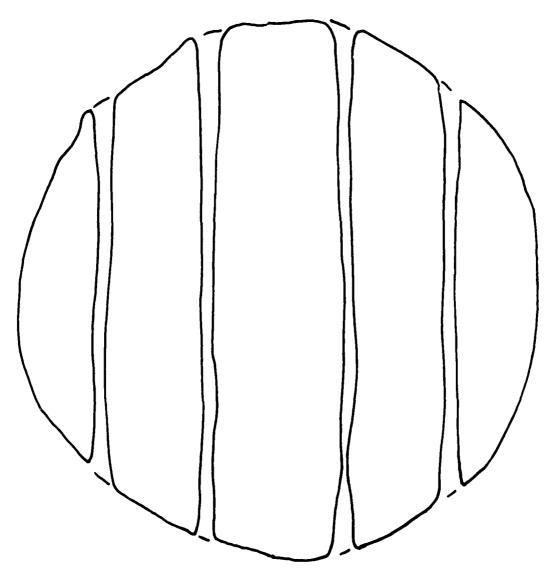


Figure C-58. Tire Contact Prints (Footprints)

of Olympia System	1784 1912	Mig ?edron.
the second second second second	. Rolling has been	
	or or state of the transfer	TIV FIRE Y FINN
		FIRS EDUTUDING (7.1)
The state of the s		MAX FOOTPRINT WOTH, 5:344 IN.
The state of the s		- MET CONTACT AREA
Contract to the contract of th		APPROVE CONTACT AREA 49:21 SQ. IN. SERIAL NR. P029003
	i. last	MERIAL NR POLYCCI

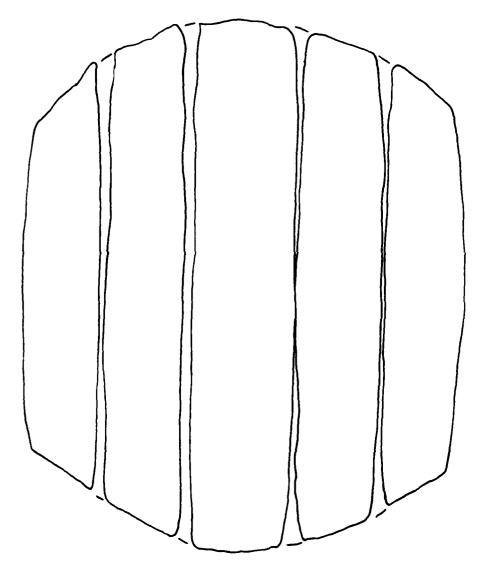


Figure Feed. Them Contact Prints (Footprints)

graf içk formanı:	TIBL SIZE		Land Brown District
54	. REINCAD OY		
radio (Maria Maria) Profesio de Cartos	tible 10.	Million Committee	Links Links
CALL CHIEF TO THE STATE OF THE	PS	- May - Footbothr :	A STATE OF S
The total Jack Miller	185.	- SET CONTAGE ARE	$K_{\rm cons} = 24 \cdot 100 \cdot 100 \cdot 100$
		- GROSS COLLACT AM	SEA CONTRACT THE
s francis	DATE "	frif i bliklidt det	13.6 20.46 x 3

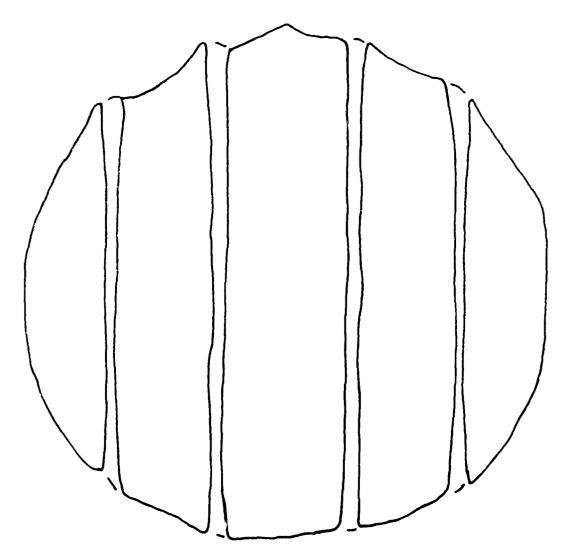
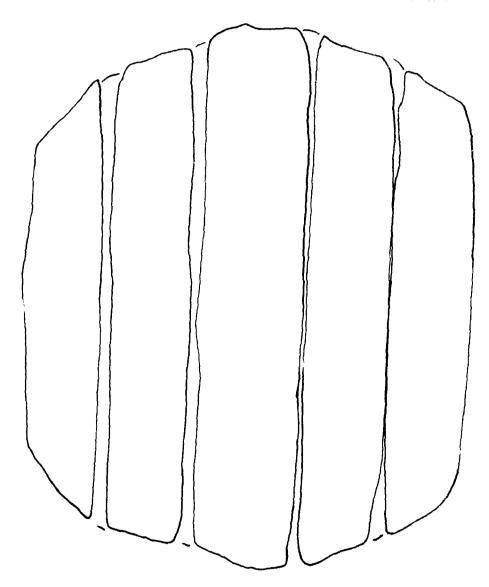


figure C-60. Tire Contact Prints (Footprints)

Andrew Comptain	Title of the	(A)	••
	er en	143	•
Reference of the American		MAKE HERT STATES	
Control of the Contro	iAH 1/1		



Congressions, Core Contact Frants (Lagrange)

TEST TIRE FOOTPRINT: NEWX. USED	TIRE SIZE .	7.00-8 M/A	MFR Zedron
5. 0. NO	. CODE NO.	!!5 FL	PLX FLWHL
SHID DEPTH	IN.	MAX. FOOTPRIN	T LGTH. 19:313 IN. T WOTH 6:078 TN
. 60. % RATED LOAD 3990	LBS.	NET CONTACT A	REA27.20 SQ. IN.
17.46. DEFLECTION OPERATOR	DATE 3/20	GROSS CONTACT 7/79 SERIAL NR	AREA 39:04. SQ. IN. B029DD3

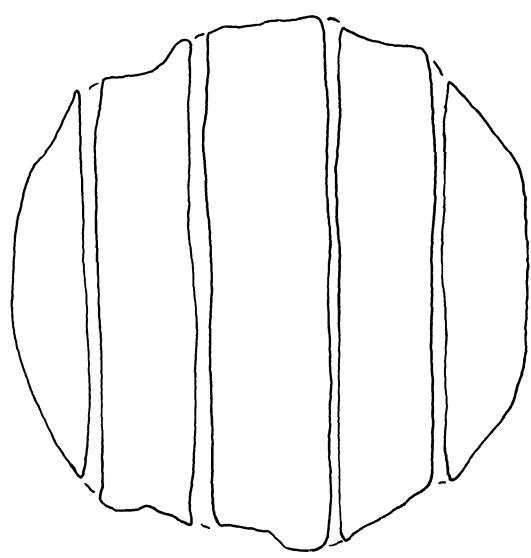


Figure C-62. Tire Contact Prints (Footprints)

#### APPENDIX D

### VERTICAL LOAD VS VERTICAL DEFLECTION PLOTS

Notes: For the following plots

- 1. Inflation Pressures are in PSIG
- 2. Outside Diameter (OD) in inches
- 3. Cross Section (CS) in inches

CAST TIRE EVALUATION
FRAT PLATE
TAN ORGE (STANDARD TIRE)

PRESS.	9()	c <u>s</u>
91	26,453	7.000
124	ac.500	7.016
164	20.57	7.016

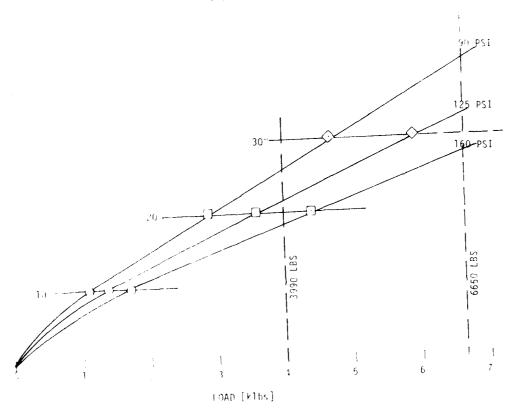


Figure D-1. Deflection Vs Load

CAST TIPE EVALUATION
FLYWHEEL (84" DIA.)

SZN 0920 (STANDARD TIPE)

PRESS	οίυ	cz
JU	20.453	6.953
1.25	20.516	7,016
166	20.57P	7.016

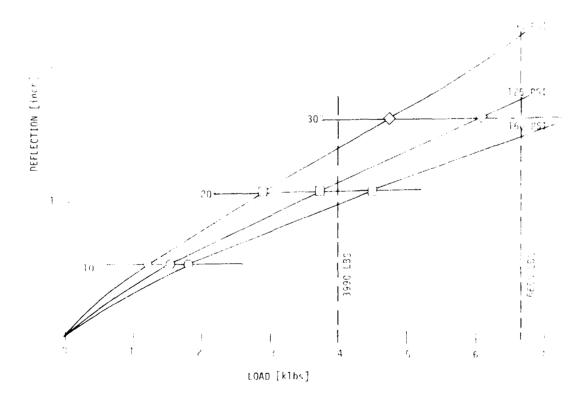


Figure 5-1. Deflection 7. Load

# CAST TIRE EVALUATION FLAT PLATE S/N A077A1

	PRESS.	<u>00</u>	<u>cs</u>
3	90	20.278	8.228
	125	20.394	8.216
	160	20.506	8.210

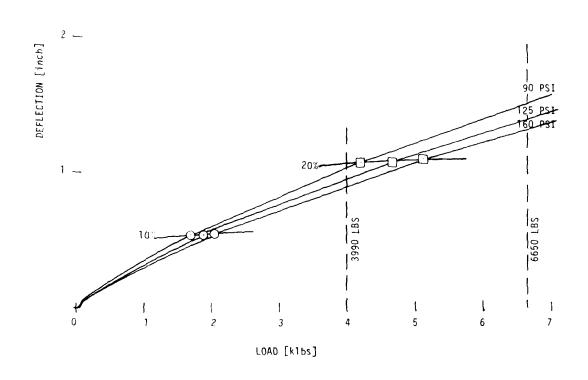


Figure D-3. Deflection Vs Load

## CAST TIRE EVALUATION FLYWHEEL (84" DIA.) S/N A077A1

	PRESS.	00	<u>cs</u>
3 —	90	20.274	8.297
	125	20.392	8.224
	160	20.502	8.215

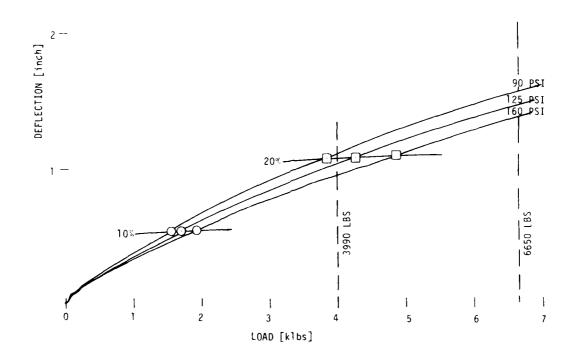


Figure D-4. Deflection Vs Load

CAST TIPE EVALUATION,
FLAT PLAT:
5/N A097881



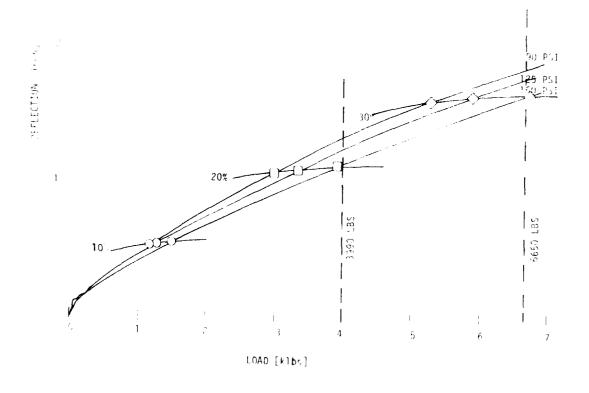


Figure 16% - Deflection Vs. Load

PRESS.		* * * * * * * * * * * * * * * * * * *
94.		.1):
125		1 ↔
160	7 ( F.1 )	111

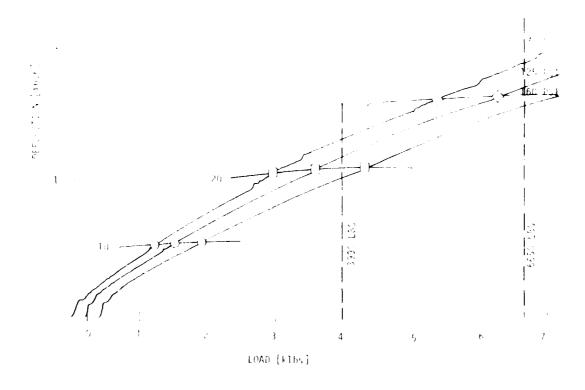


Figure 5-6. Deflection Vs Load

**?** -

# CAST TIRE EVALUATION FLAT PLATE S/N A028C4

PRESS.	<u>od</u>	<u>cs</u>
90	20.330	8.320
125	20.500	8.310
160	20.698	8.312

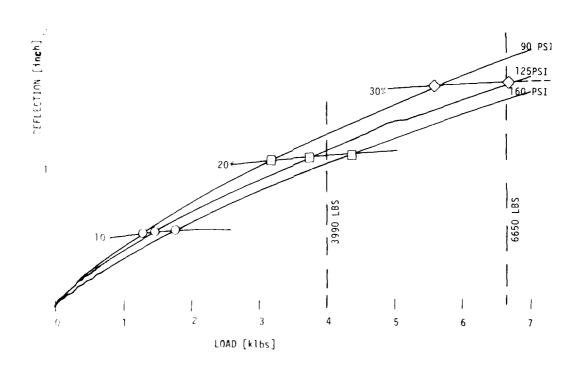


Figure D-7. Deflection Vs Load



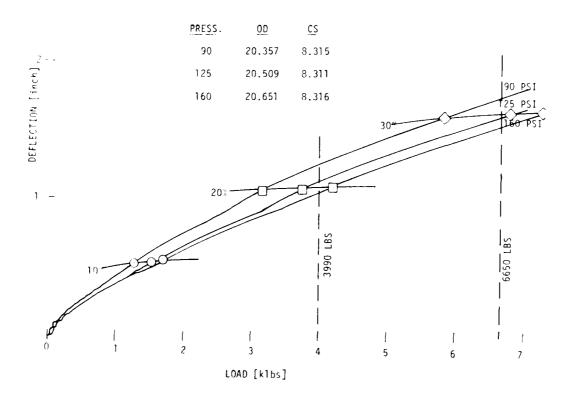
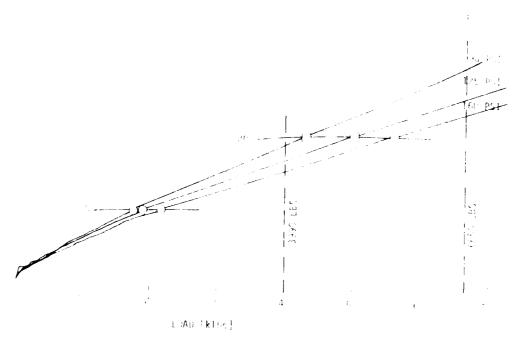


Figure D-8. Deflection Vs Load

### TADI TIRE LYILDATION FLAT PLATE 5/N 8097A3





Prime 1 C. Deficition 2 (1911)

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<del>-</del>			

CAST TIRE EVALUATION
FLAT PLATE
S/N B028B4

PRESS.	ob	<u>cs</u>
90	20,611	8.427
1,15	20.683	8.446
160	20,725	8.472

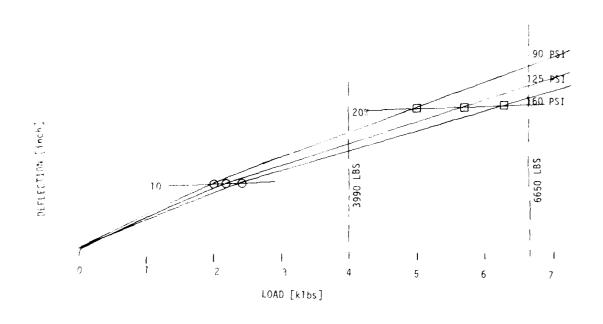


Figure D-11. Deflection Vs Load

## CAST TIRE EVALUATION FLYWHEEL (R4" DIA.) G/N B02884

PM: .	uţi	Ċ5
4	16.620	÷.441
1.14	.T.,636	8,467
14,	.0.14	4,494

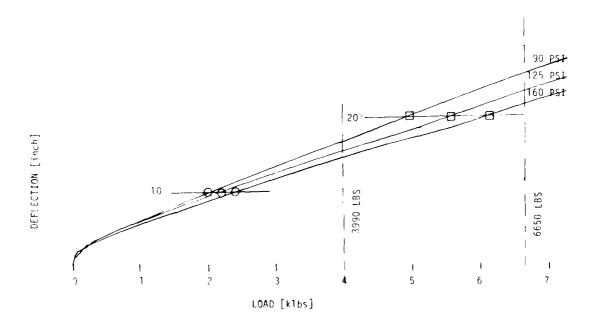
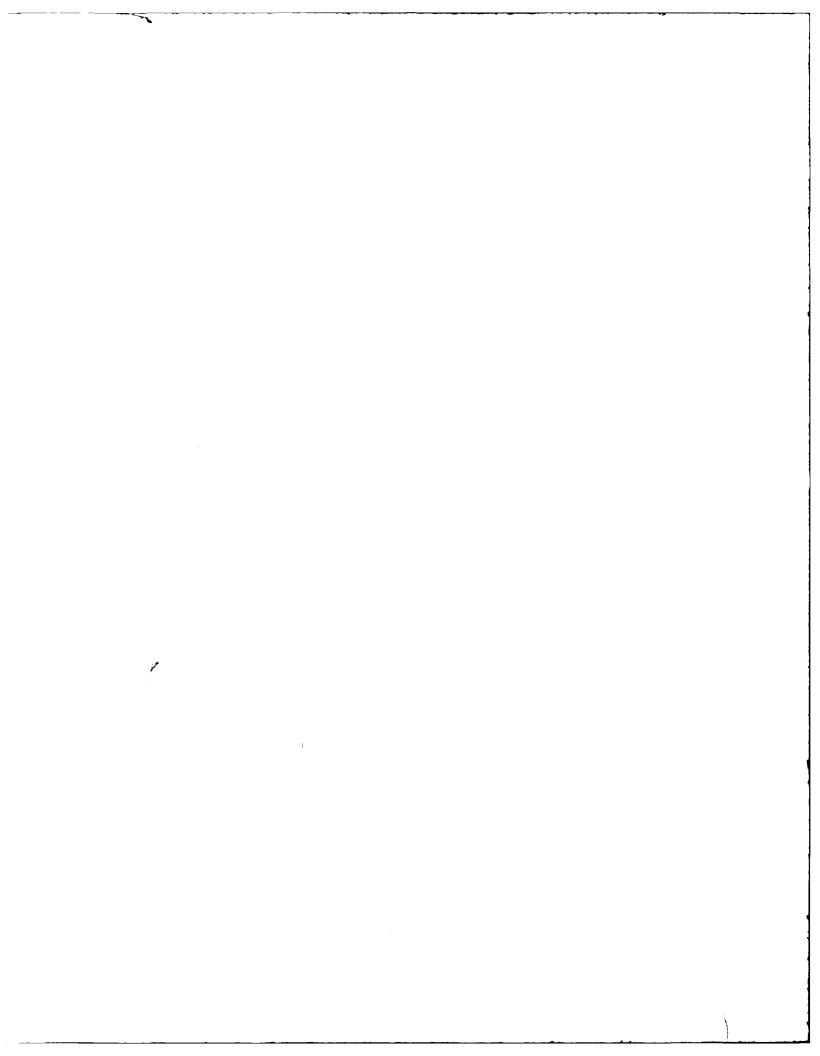


Figure D-12. Deflection Vs Load





CAST TIRE EVALUATION
FLAT PLATE
S/N B078C4

PRESS.	ΩO	C <sub>S</sub>
90	26.790	8.474
125	20.850	8.533
160	20.≥94	8.607

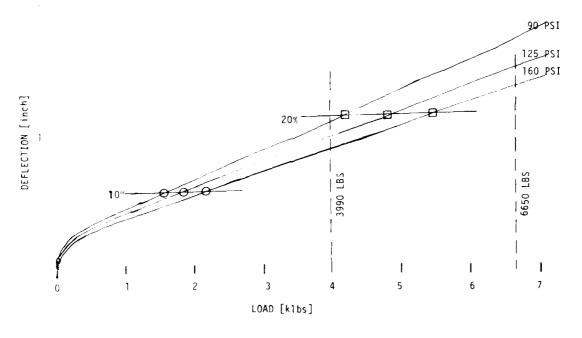


Figure D-15. Deflection Vs Load

CAST TIRE EVALUATION
FLYWHEEL (84" DIA.)
S/N B078C4

PRESS.	<u>OD</u>	<u>c</u> s
90	20.803	8.470
125	20.837	8 <b>.521</b>
160	20.879	8.593

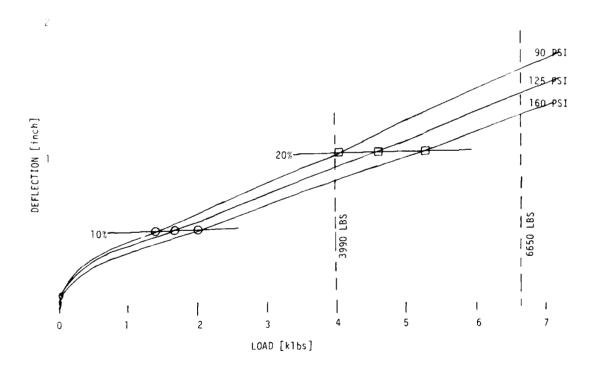


Figure D-16. Deflection Vs Load

٠ 4. in the second . . . . . . .

for the second s

### CAST TIRE EVALUATION FLAT PLATE

S/N\_B08814

PRESS.	OD	<u>cs</u>
90	20,740	8.387
125	20.792	8.447
160	20.834	8.525

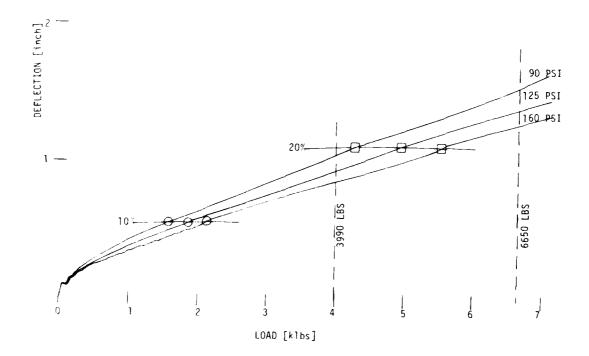


Figure D-19. Deflection Vs Load

## CAST TIRE EVALUATION FLYWHEEL (84" DIA.) S/N B08814

3			
	PRESS.	<u>od</u>	<u>cs</u>
	90	20.756	8.402
	125	20.796	8.461
	160	20.834	8.534

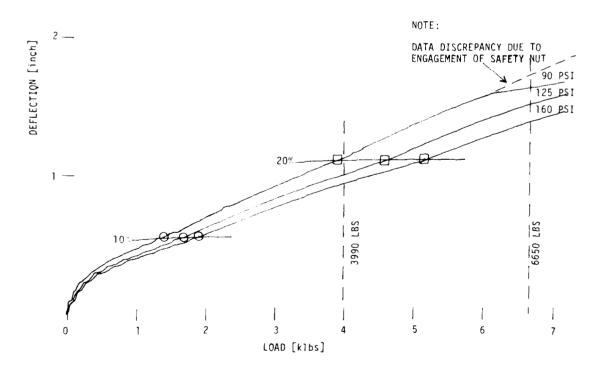
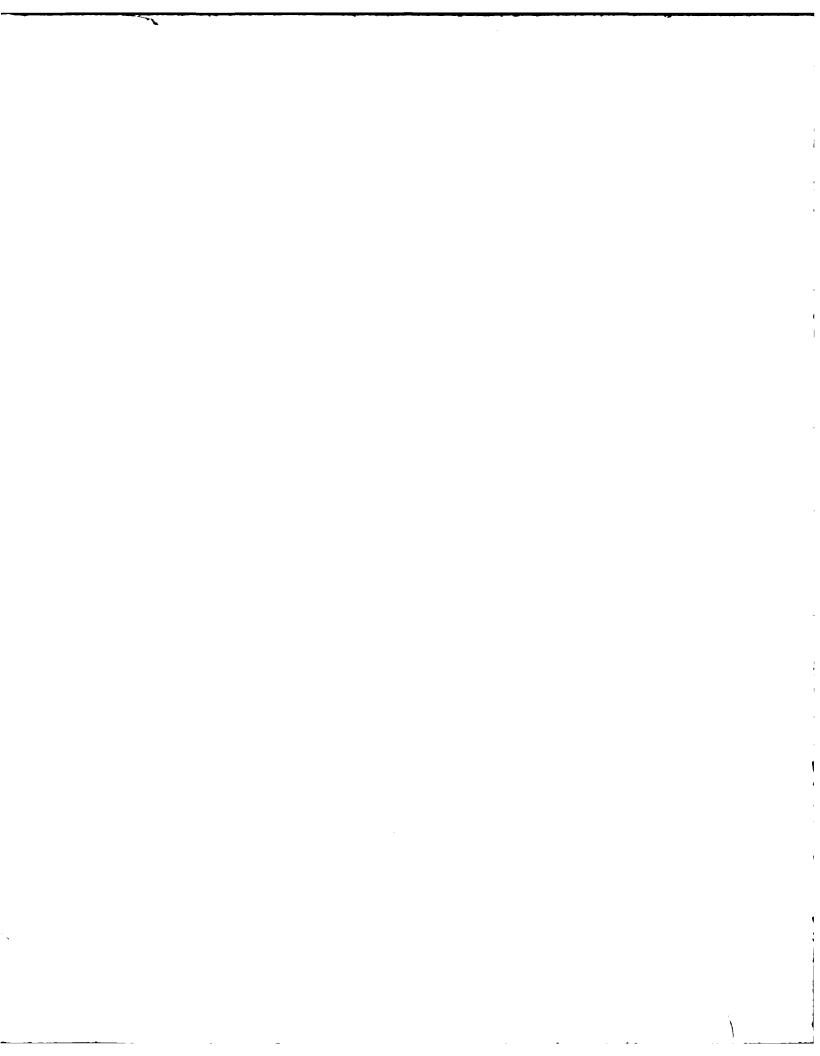


Figure D-20. Deflection Vs Load





# CAST TIRE EVALUATION FLAT PLATE S/N B088K4

PRESS.	ÓD	<u>cs</u>
90	20.748	8.263
125	20,842	8.280
160	20 894	8 321

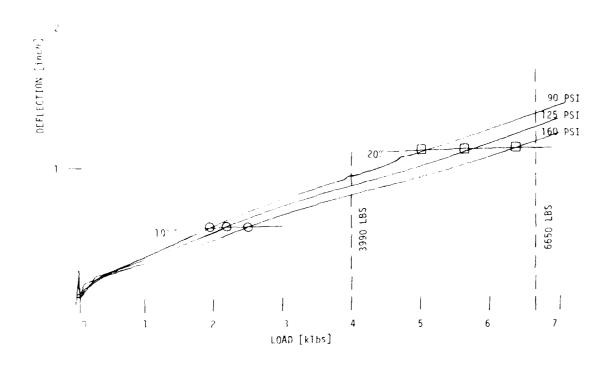


Figure D-23. Deflection Vs Load

CAST TIRE EVALUATION
FLYWHEEL (R4" DIA.)
S/N BO38r4

	PRESS.	OD.	CS
3	90	20.742	8.267
	125	20.836	9.276
	160	20.886	≥.314

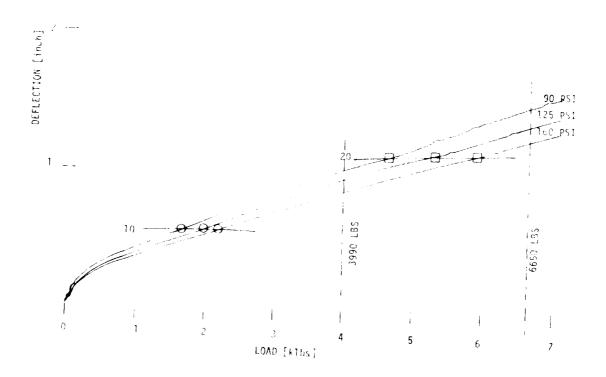
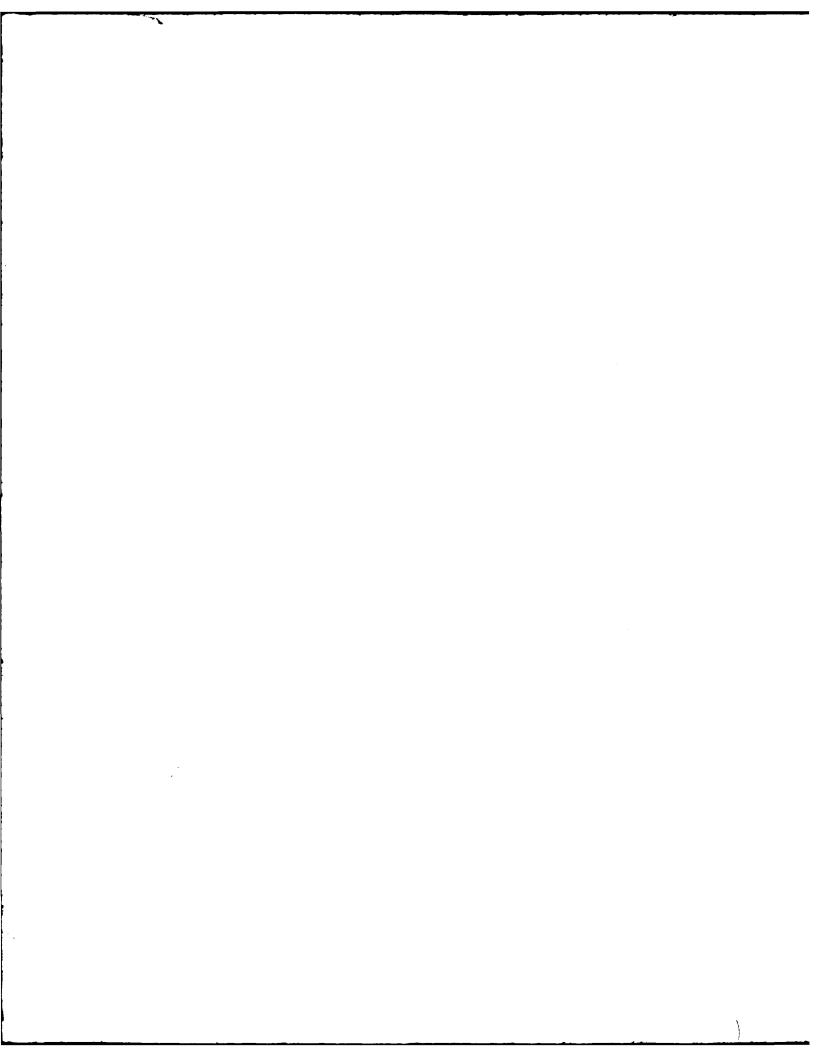
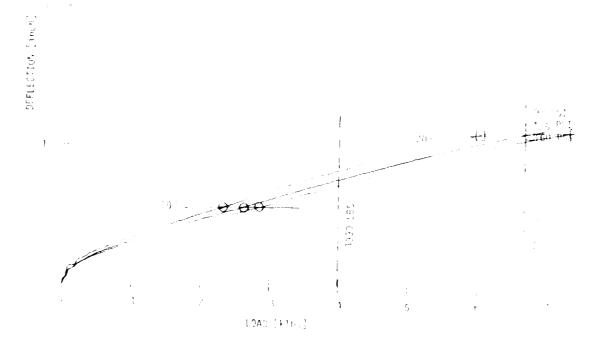


Figure D-74. Deflection Vs Load

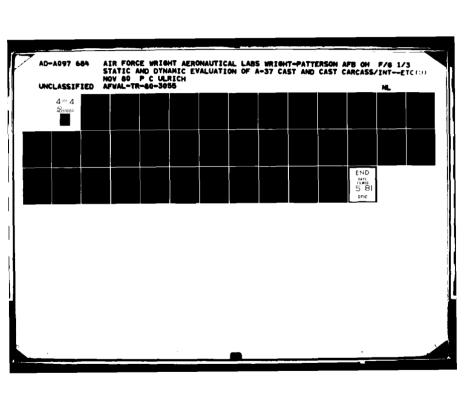


### CAST TIPE EVALUATION FLYWHEEL (1141 DIA.) S/N BOOKL3

PRESS.	υü	CS.
90	20.310	8.256
125	2064	8.269
160	20.900	100



Transport, deflection count



CAST TIRE EVALUATION
FLAT PLATE
S/N B098M2

3 ---

PRESS.	<u>OD</u>	<u>cs</u>
90	20.852	8.3 <b>9</b> 9
125	20.910	8.417
160	20.952	8.441

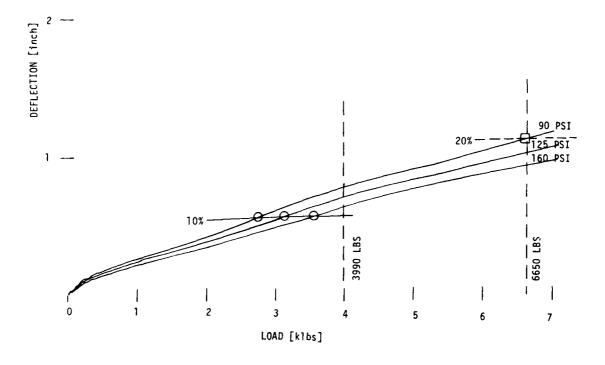


Figure D-27. Deflection Vs Load

#### CAST TIRE EVALUATION FLYWHEEL (84" DIA.) S/N B098M2

3 —	PRESS.	<u>OD</u>	<u>cs</u>
	90	20.850	8,395
	125	20.924	8.418
	160	20.982	8.454

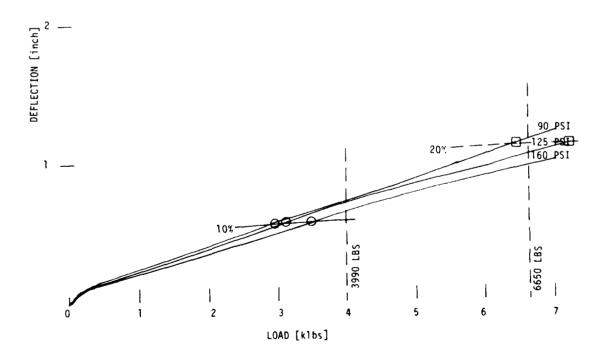


Figure D-28. Deflection Vs Load

### CAST TIRE EVALUATION FLAT PLATE S/N B098N2

PRESS. OD CS
90 20.824 8.432
125 20.874 8.464
160 20.912 8.498

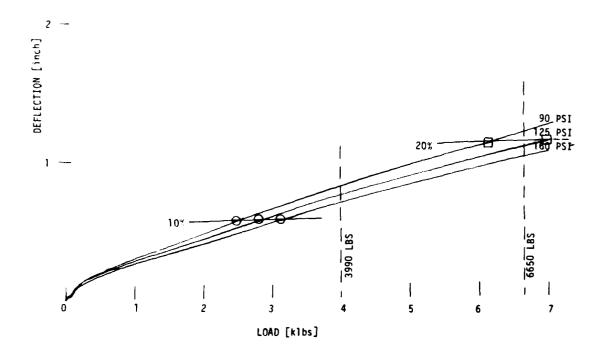


Figure D-29. Deflection Vs Load

#### CAST TIRE EVALUATION FLYWHEEL (84" DIA.) S/N B098N2

PRESS.	<u>00</u>	<u>cs</u>
90	20.804	8.411
125	20.864	8.437
160	20.908	8.478

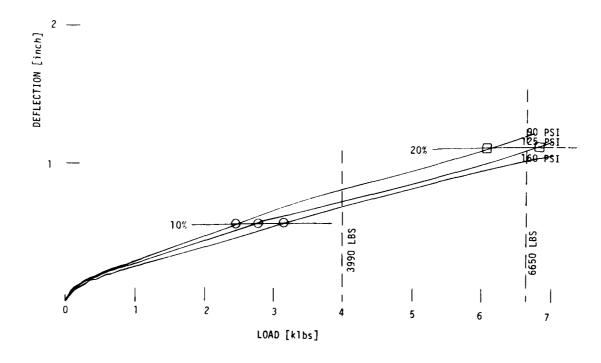


Figure D-30. Deflection Vs Load

#### CAST TIRE EVALUATION FLAT PLATE S/N B09802

3 —	PRESS.	<u>00</u>	<u>cs</u>
	90	20.701	8.338
	125	20.766	8.366
	160	20.812	8.411

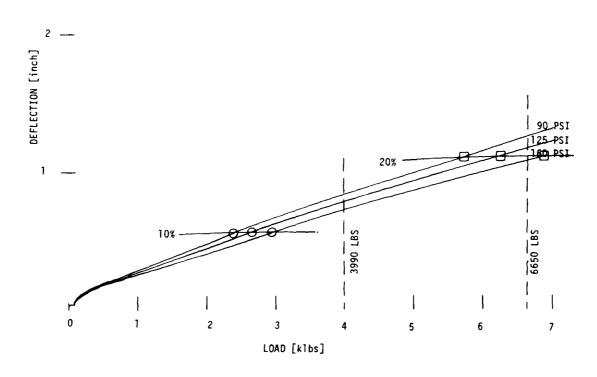


Figure D-31. Deflection Vs Load

#### CAST TIRE EVALUATION FLYWHEEL (84" DIA.) S/N B09802

PRESS.	<u>OD</u>	<u>cs</u>
90	20.724	8.361
125	20.778	8.387
160	20.818	8.424

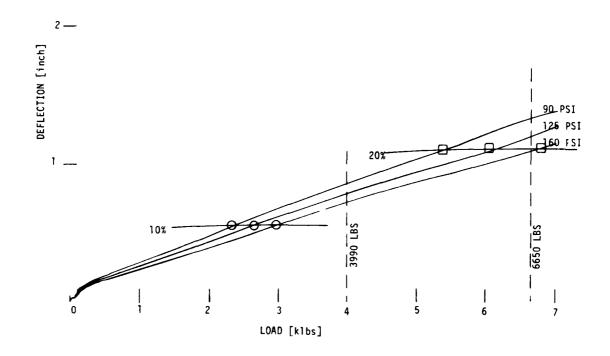


Figure D-32. Deflection Vs Load

CAST TIRE EVALUATION
FLAT PLATE
S/N B098P2

PRESS.	ОD	C <u>S</u>
90	20,980	8.769
125	21.070	8.949
160	21.150	9.157

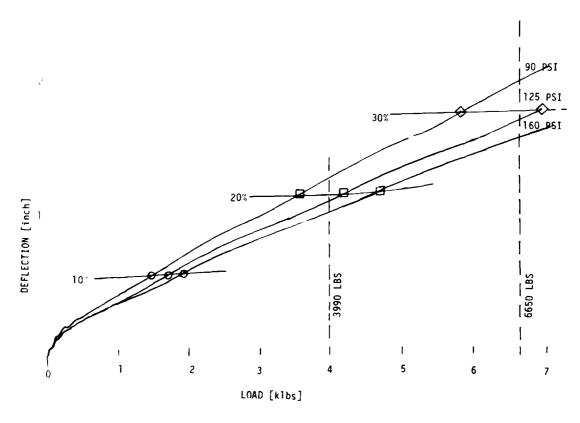


Figure D-33. Deflection Vs Load

CAST TIRE EVALUATION
FLYWHEEL (84" DIA.)
S/N B098P2

PRESS.	<u>0D</u>	<u>cs</u>
90	20.920	8.649
125	20.988	8.796
160	21.036	8.981

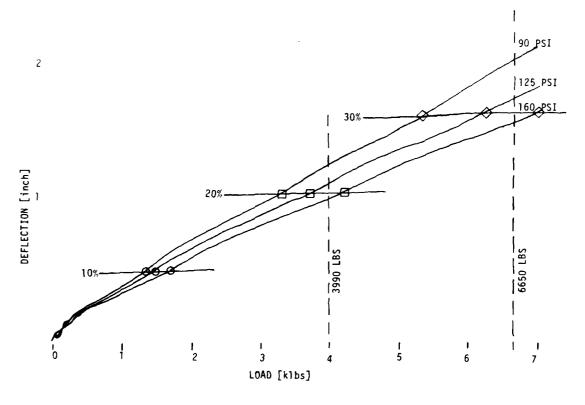


Figure D-34. Deflection Vs Load

CAST TIRE EVALUATION

FLAT PLATE

S/N B098Q3

PRESS.	$\overline{OD}$	<u>cs</u>
90	20.726	8.261
125	20.787	8.301
160	20.820	8.3295

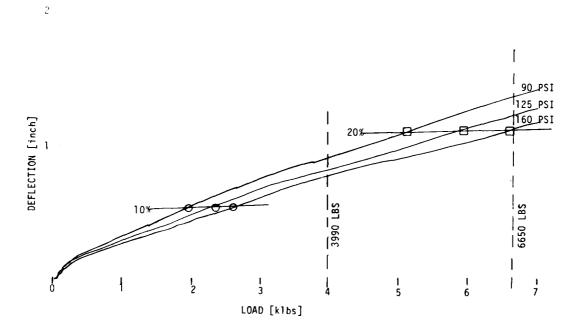


Figure D-35. Deflection Vs Load

CAST TIRE EVALUATION
FLYWHEEL (84" DIA.)
S/N B098Q3

PRESS.	00	<u>cs</u>
90	20.719	8.257
125	20.771	8.275
160	20.812	8.328

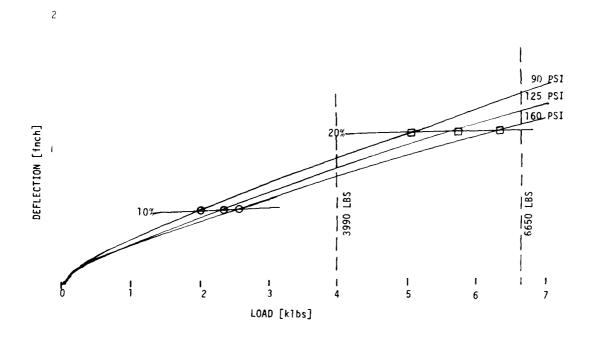


Figure D-36. Deflection Vs Load

#### CAST TIRE EVALUATION FLAT PLATE S/N B098R2

PRESS.	00	czs
90	20.934	8,896
125	21.024	9.013
160	21.104	9.252

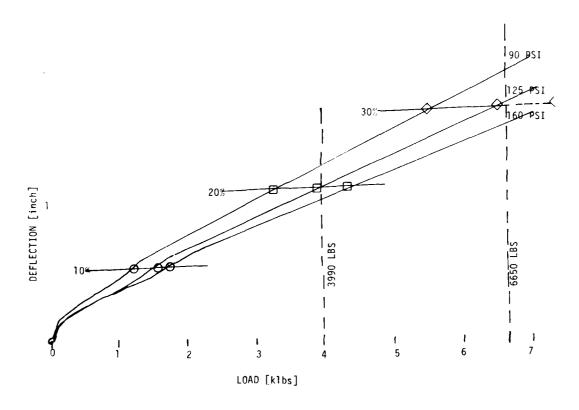


Figure D-37. Deflection Vs Load

CAST TIRE EVALUATION
FLYWHEEL (84" DIA.)
S/N BO98R2

PRESS.	ο̈́D	C <sub>i</sub> S <sub>i</sub>
90	20,914	8.795
125	21.016	9.003
160	21.096	9 196

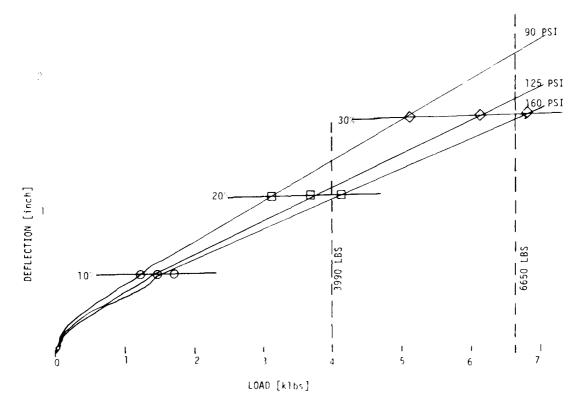


Figure D-38. Deflection Vs Load

#### CAST TIRE EVALUATION FLAT PLATE

S/N B09853

PRESS.	OD	<u>cs</u>
90	20.686	8.204
125	20.762	8.200
160	20.812	8.211

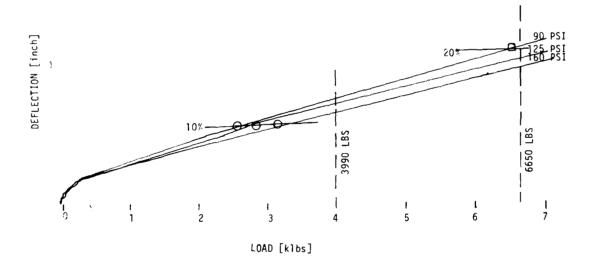


Figure D-39. Deflection Vs Load

CAST TIRE EVALUATION

FLYWHEEL (84" DIA.)

\$/N B098S3

PRESS.	ÖD	ĊS
90	20.674	8.217
125	20.750	8.210
160	20, 806	8 21/1

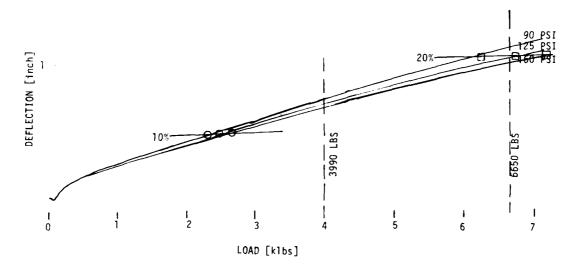


Figure D-40. Deflection Vs Load

CAST TIRE EVALUATION
FLAT PLATE
S/N B098T2

PRESS.	ŌD	CS
90	20.778	8.275
125	20.836	8.283
160	20.874	8.298

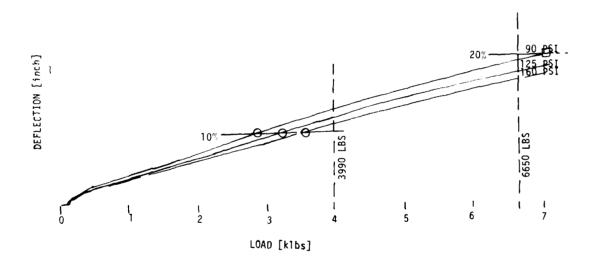


Figure D-41. Deflection Vs Load

CAST TIRE EVALUATION
FLYWHEEL (84" DIA.)
S/N B098T2

PRESS.	OD	<u>cs</u>
90	20.768	8.274
125	20.830	8.280
160	20.874	8.295

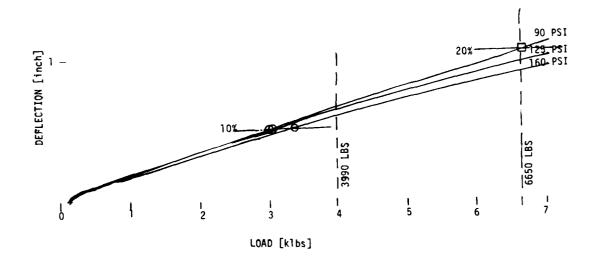


Figure D-42. Deflection Vs Load

CAST TIRE EVALUATION
FLAT PLATE
S/N B128U3

PRESS.	OD	cs
90	20.848	8. <b>436</b>
125	20.918	8.486
160	20 970	8 543

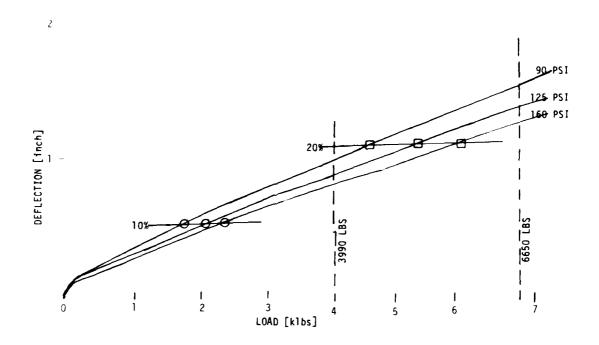


Figure D-43. Deflection Vs Load

CAST TIRE EVALUATION
FLYWHEEL (84" DIA.)
S/N B128U3

PRESS.	ÓD	cz
90	20.874	8.446
125	20.9 <b>26</b>	8.494
160	20.967	8.549

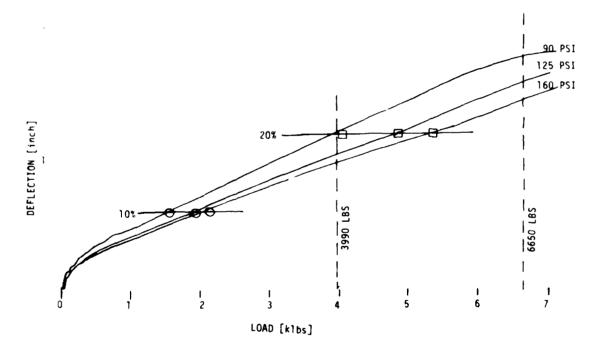


Figure D-44. Deflection Vs Load

CAST TIRE EVALUATION
FLAT PLATE
S/N B128V3

PRESS.	OD	<u>cs</u>
90	20.932	8.417
125	20.946	8.460
160	21.046	8.506

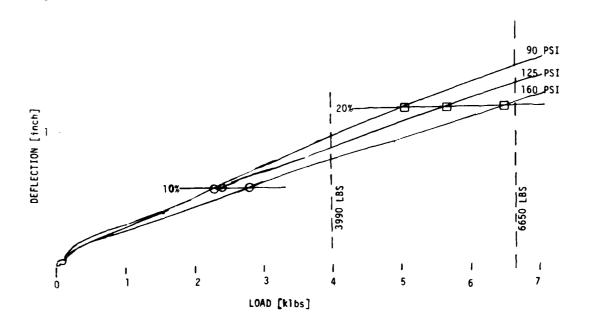


Figure D-45. Deflection Vs Load

CAST TIRE EVALUATION
FLYWHEEL (84" DIA.)
S/N B128V3

PRESS.	<u>00</u>	<u>cs</u>
90	20.942	8.407
125	20.998	8.453
160	21.068	8.491

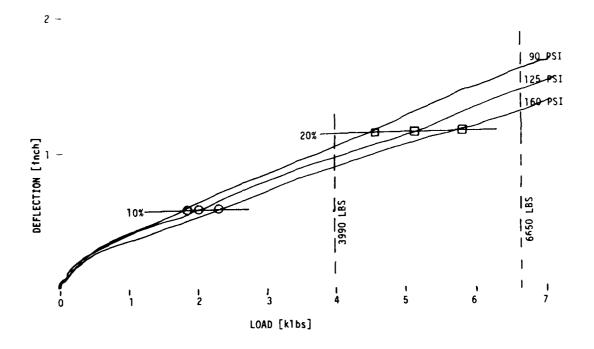


Figure D-46. Deflection Vs Load

#### CAST TIRE EVALUATION FLAT PLATF 5/N B029W3

PRESS.	ο̈́ο	C <sub>S</sub>
90	20,788	8.415
125	20,368	8,468
160	20 912	g 52 <b>0</b>

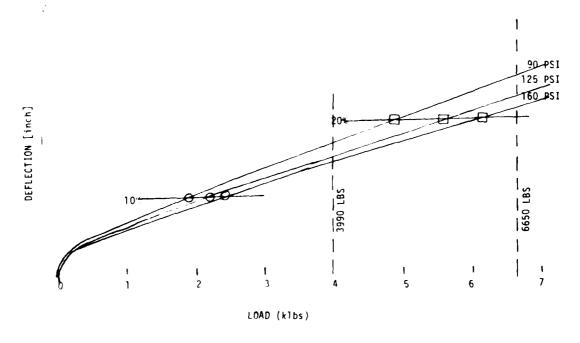


Figure D-47. Deflection Vs Load

CAST TIRE EVALUATION
FLYWHEEL (84" DIA.)
S/N B029W3

PRESS.	ŌĎ	$\overline{c}$ s
90	20.790	8.432
125	20.838	8,482
160	20 072	0 527

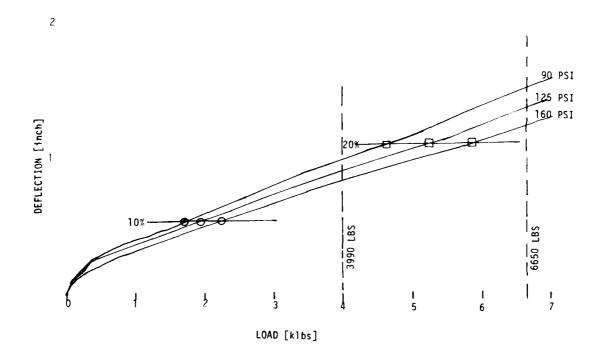


Figure D-48. Deflection Vs Load

2

CAST TIRE EVALUATION
FLAT PLATE
S/N B029X3

PRESS.	ŌD	<u>cs</u>
90	20.800	8.459
125	20.860	8.507
160	20 910	8 570

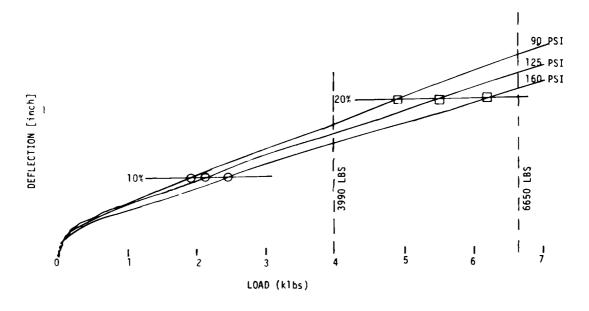


Figure D-49. Deflection Vs Load

CAST TIRE EVALUATION
FLYWHEEL (84" DIA.)
S/N B029X3

PRESS.	OD	cz
90	20.812	8.463
125	20.862	8.509
160	20.904	8.564

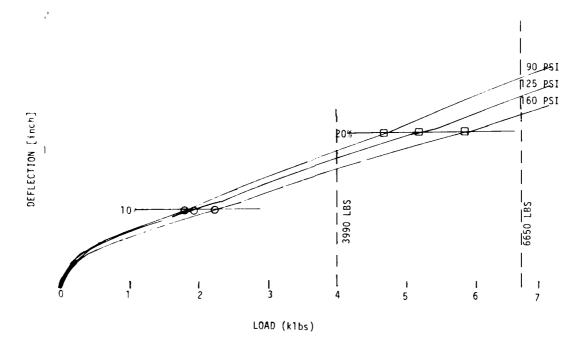


Figure D-50. Deflection Vs Load

#### CAST TIRE EVALUATION FLAT PLATE S/N B029Y3

3			
	PRESS.	OD	<u>cs</u>
	90	20.752	8.357
	125	20.820	8.394
	160	20.858	8.440

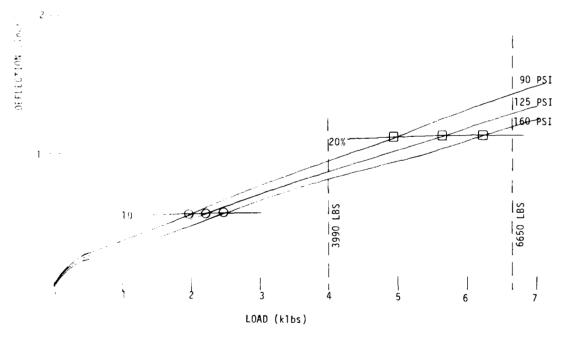


Figure D-51. Deflection Vs Load

#### CAST TIRE EVALUTION FLYWHEEL (84" DIA.) S/N B029Y3

	3							
			PRESS.	<u>0D</u>	<u>cs</u>			
			90	20.758	8.368			
			125	20.806	8.399			
			160	20.844	8.442			
DEFLECTION (inch)	2							   90 PSI   125 PSI
	1—-				     		<del></del>	160 PSI
		10*	<i>7</i>		3990 LBS			6650 LBS
	0	1	2	}	4	ļ	6	7
				LOAD (klbs	)	-	v	,
				. ,	•			

Figure D-52. Deflection Vs Load

CAST TIRE EVALUTION
FLAT PLATE
S/N B029Z3

3 —

PRESS.	<u>00</u>	<u>cs</u>
90	20.812	8.456
125	20.852	8.507
160	20.882	8.572

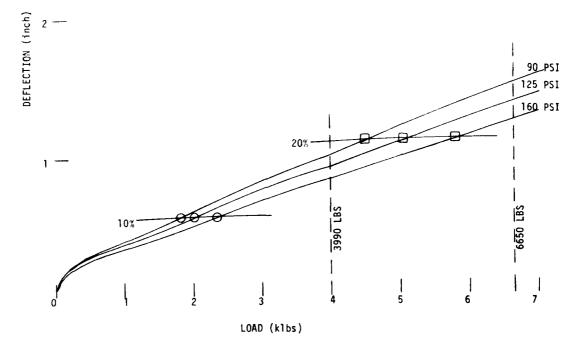


Figure D-53. Deflection Vs Load

#### CAST TIRE EVALUATION FLYWHEEL (84" DIA.) S/N B029Z3

PRESS. OD CS
90 20.822 8.468
125 20.858 8.521
160 20.888 8.584

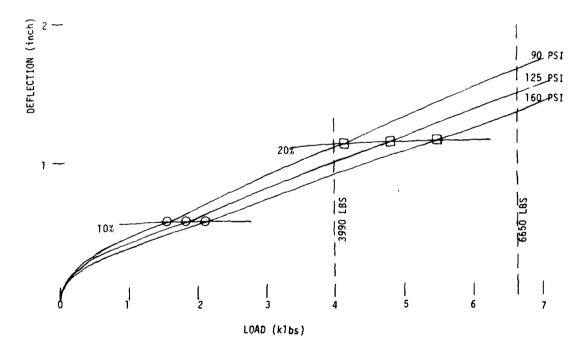
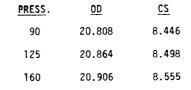


Figure D-54. Deflection Vs Load

CAST TIRE EVALUATION
FLAT PLATE
S/N B029AA3

3 —



DEFLECTION (inch)

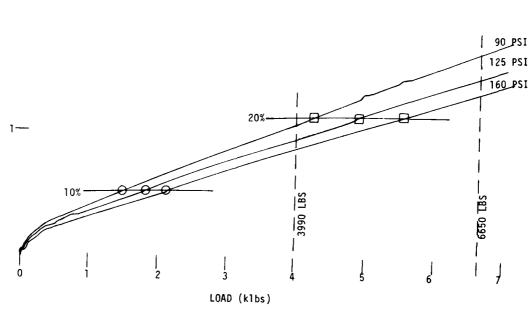


Figure D-55. Deflection Vs Load

#### CAST TIRE EVALUATION FLYWHEEL (84" DIA.) S/N B029AA3

3 ---

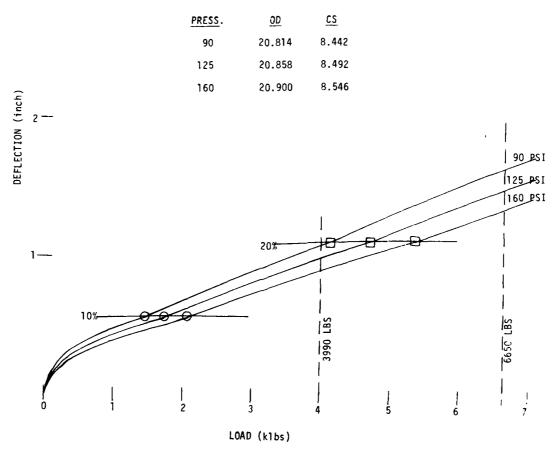


Figure D-56. Deflection Vs Load

CAST TIRE EVALUATION
FLAT PLATE
S/N B029BB3

3 —

PRESS.	<u>od</u>	<u>cs</u>
90	20.830	8.550
125	20.872	8.631
160	20.894	8.739

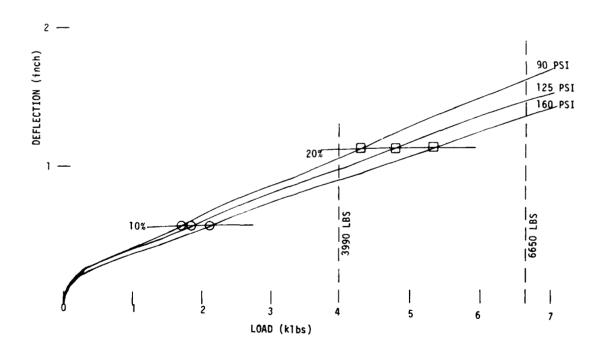


Figure D-57. Deflection Vs Load

#### CAST TIRE EVALUATION FLYWHEEL (84" DIA.) S/N B029BB3

3--

PRESS.	<u>od</u>	<u>cs</u>
90	20.834	8.579
125	20.860	8.662
160	20.884	8.753

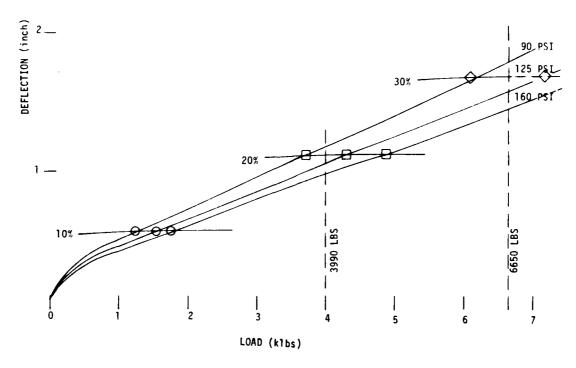


Figure D-58. Deflection Vs Load

#### CAST TIRE EVALUATION FLAT PLATE S/N B029CC3

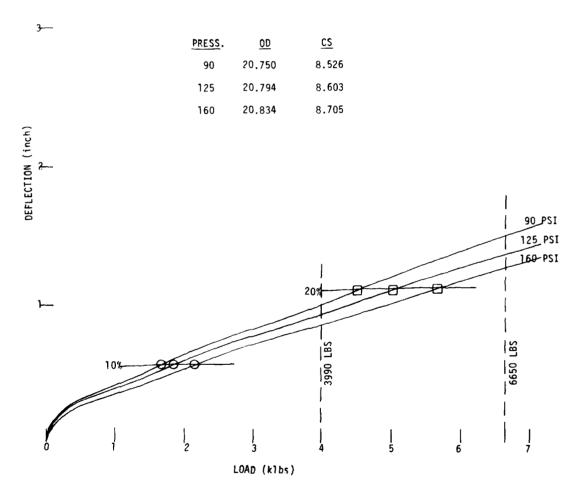


Figure D-59. Deflection Vs Load

#### CAST TIRE EVALUATION FLYWHEEL (84" DIA.) S/N BO29CC3

PRESS.	<u>od</u>	<u>cs</u>
90	20.776	8.564
125	20.816	8.643
160	20.848	8.736

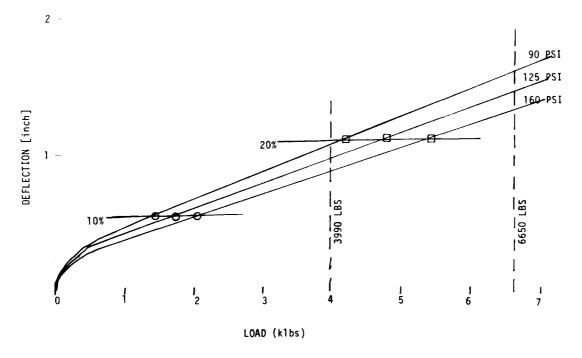


Figure D-60. Deflection Vs Load

#### CAST TIRE EVALUATION FLAT PLATE S/N B029DD3

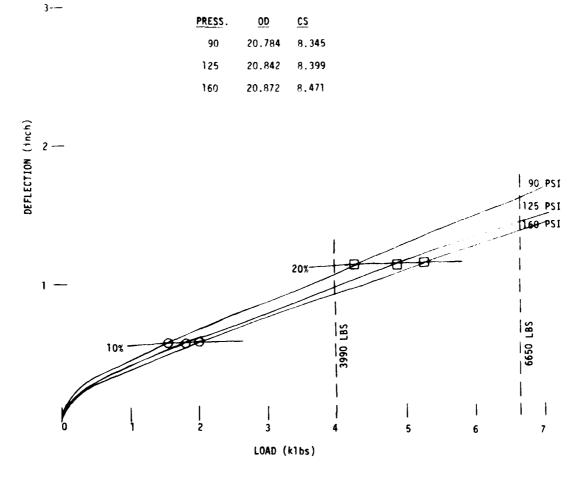


Figure D-61. Deflection Vs Load

CAST TIRE EVALUATION
FLYWHEEL (84" DIA.)
S/N B029DD3

PRESS.	<u>od</u>	<u>cs</u>
90	20.792	8.374
125	20.870	8.427
160	20.910	8.489

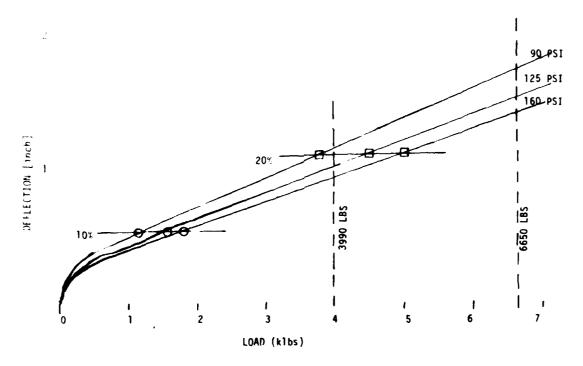


Figure D-62. Deflection Vs Load

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- 7. Robert F. Smiley and Walter B. Horne, <u>Mechanical Properties of Pneumatic Tires with Special Reference to Modern Aircraft Tires</u>, <u>National Aeronautics and Space Administration</u>, <u>NASA-TR-R-64</u>, 1960.

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